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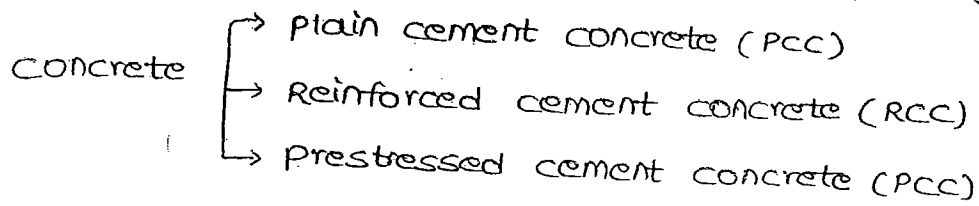
CIVIL ENGINEERING STUDENTS AND GRADUATES



UNIT - 1

MATERIAL AND SPECIFICATION (1-2M)

Construction material for Civil Engineering structures



Advantages of structural steel sections Vs concrete member

1. Steel sections having a High strength than concrete.
2. More economical
3. $\left(\frac{\text{Strength}}{\text{weight}}\right)_{\text{structural steel section}} > \left(\frac{\text{Strength}}{\text{weight}}\right)_{\text{concrete member}}$
4. Fast and Rapid construction is possible only with steel
5. Ductile material (steel) (Min. percentage of elongation of steel is

Ductility can be measured by percentage of elongation and percentage of area reduction.

$$\text{percentage of elongation} = \frac{\text{change in length}}{\text{Original length}} \times 100$$

$$= \frac{\Delta l}{l} \times 100$$

- percentage of elongation > 15% - Ductile material
> 5-15% - Intermediate ductile material
< 5% - Brittle material

6. 100% scrap value
7. Overall cost of construction is less.
8. Easy modification (or) Repairs
9. High quality and reliability

Safety norms:-

Working stress method - Factor of safety $\left\{ \begin{array}{l} \rightarrow \text{concrete} = 3 \\ \rightarrow \text{Steel} = 1.7 - 1.8 \end{array} \right.$

Limiting stress method - partial safety factors (γ_m) $\left\{ \begin{array}{l} \rightarrow \text{concrete} = 1.5 \\ \rightarrow \text{Steel} = 1.15 \end{array} \right.$

Steel :-

Iron (By % of wt. >98%) + small % of carbon (By % of wt. 0.1% - 1.1%) + Sulphur, Manganese, Silicon, phosphorous, Nickle, Chrome etc → Steel

Mechanical properties :-

1. Yield strength (f_y)
2. Ultimate tensile strength (f_u)
3. Hardness
4. Ductility
5. Toughness

→ Mild steel Fe 250 (carbon % = 0.23) → Low carbon steel

→ High Tensile steel Fe 415, 500 (carbon = 0.27%) → Medium carbon steel.

Classification of Steel :-

Based on amount of carbon content :-

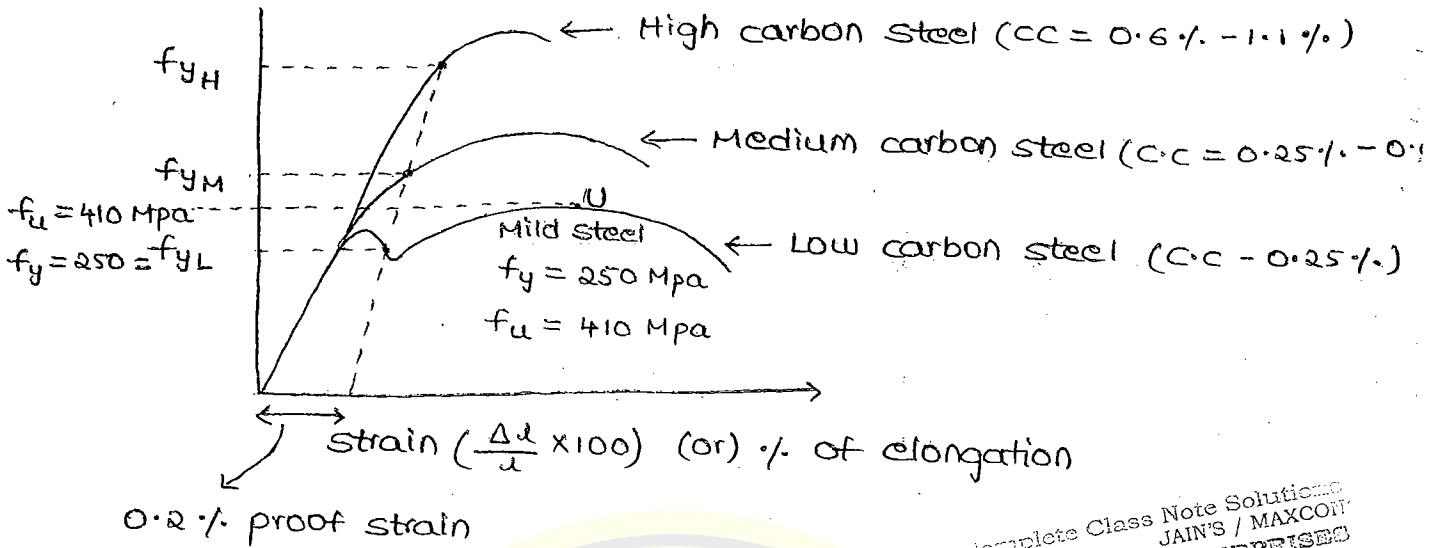
1. Low carbon steel (0.1 - 0.25% of carbon by % of weight)
Ex:- Mild steel bars used in RCC constructions, structural steel sections used in steel buildings (or) Bridge constructions.
2. Medium carbon steel (0.25 - 0.6% carbon)
Ex:- High tensile steel, Rail, tyre, hammer etc.
3. High carbon steel (0.6% - 1.1% carbon)
Ex:- Stone masonry tools, drills, punches etc.

** Note :-

By increasing amount of carbon content yield strength of steel (f_y), ultimate tensile strength of steel (f_u), hardness of steel increases. However it reduces ductility and toughness of steel.

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Stress strain curve :-



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General classification of steel :-

In general steel may be two types.

1. Reinforcing steel
2. structural steel

Reinforcing steel :-

It is used as Reinforcing bars in R.C.C. construction. Designated by its yield strength.

EX:- Fe 250, Fe 415, Fe 500 etc.

Fe = steel
250 = yield strength of steel

Structural steel :-

It is used in steel buildings, bridges etc. Designated by its ultimate tensile strength.

EX:- Fe 410, Fe 570 etc.

Codes and standards :-

IS 800	}	1956-1962 (1 st code book published)	code of practise for use of structural steel in general building construction
		1984 (2 nd , revised, W.S.M)	
		2007 (3 rd , revised, L.S.D)	

IS 875: 1987 (five parts) code of practise for design loads for building and structures.

IS Hand book no.1 (steel table)

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Rolled steel sections:-

1. Rolled steel I-sections or beam sections
2. Rolled steel channel section
3. Rolled steel Tee section
4. Rolled steel Angle section
5. Rolled steel tube section
6. Rolled steel bars
7. Rolled steel flats
8. Rolled steel plates
9. Rolled steel sheets
10. Rolled steel strips

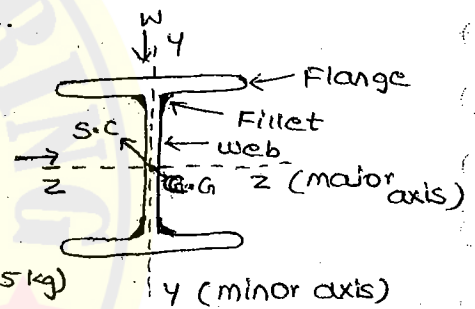
1. Rolled steel I-sections (or) Beam sections:-

An I-section may be designated by its depth and weight. C.G and S.C lies on same point.

Eg:- An ISLB 500 at 735.8 N/m

Overall depth of I-section : 500 mm
(Outer-Outer)

Weight per running meter : 735.8 N (≈ 75 kg)



C.G = Centre of gravity (or) S.C

S.C = Shear centre (or) centre of flexure

Note:-

For same weight (same sectional area) I-section provides large moment of inertia or sectional modulus about major axis (z-z axis). Hence I-section is best section or most efficient section for a beam or girder (major beam).

To support transfers loads which are parallel to the minor axis, more moment of inertia or section modulus required, about z-z axis.

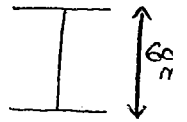
To support lateral loads which are parallel to the major axis more moment of inertia or section modulus required about yy-axis.

I-section is arranged 90-95% of moment will be taken care by flanges and web will support 90-95% of SF.

The junction between Flange and web is called Fillet will minimise stress concentration due to heavy concentrated loads. More steel is provided at Fillet such that we can avoid the "web crippling failure" in I-beam. The applied load is passing through shear centre then "twisting" will not occur.

Types of I-sections:-

1. Indian standard Junior beam (ISJB)
2. Indian standard Light beam (ISLB)
3. Indian standard Medium weight beam (ISMB)
4. Indian standard wide flange beam (ISWB)
5. Indian standard Heavy weight beam (ISHB)



Maximum depth available is 600 mm (outer - outer)

Note:-

ISWB section has more lateral Bending strength or buckling strength as compared with its counter parts (ISLB, ISMB, ISHB)

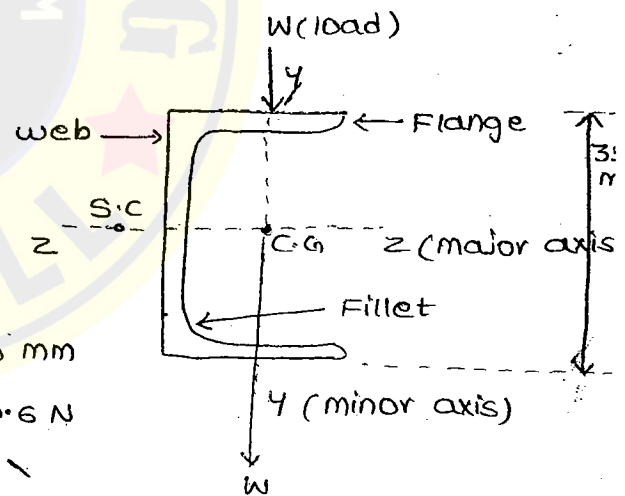
2. Rolled Steel channel section:-

A channel section may be designated by its depth and weight.

EX:- ISLC 350 at 380.63 N/m

Overall depth of channel : 350 mm

Weight per running meter : 380.6 N



Types of channel sections:-

1. ISTC (Junior channel)
2. ISLC (Light weight channel)
3. ISMC (Medium weight channel)
4. ISMCP (Medium weight channel with parallel flanges)
5. ISGC (Gate channels)

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Moment of Inertia is more at z-z axis. so it is called Major axis. M.I is less at y-y axis (only one side area then it is called minor axis. The applied load is not passing to Shear centre so twisting will occur.

Rolled steel angle section (or) L-section :-

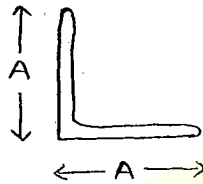
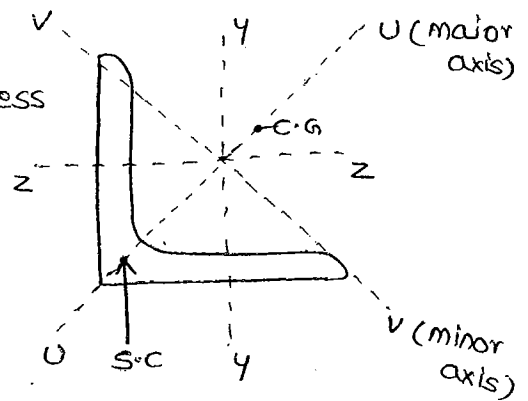
An angle section may be designated by its width and thickness

EX:- ISA 100 X 100 X 10

Types of Angle sections:-

1. Equal angles (EA)

Designated as ISEA (or) ISA

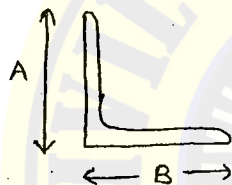


EX:- ISEA 100 X 100 X 10 (Equal angles)

It is a best section of compression member. Radius of gyration is more.

2. Unequal angles

Designated as ISA (Angles)

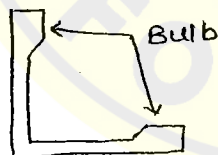


EX:- ISA 125 X 75 X 10

Unequal angles are best suited for tension members.

3. Bulb angles

Designated as ISBA (Bulb angles)



critical load (or) compressive strength of a member ($P_{c,n}$):-
(Euler's theory)

$$P_{c,n} = \frac{\pi^2 EI}{L^2} = \frac{\pi^2 EA \eta^2}{L^2}$$

$$P_{c,n} = \frac{\pi^2 EA}{\left(\frac{L}{\eta}\right)^2}$$

$$\therefore \eta = \sqrt{\frac{I}{A}}$$

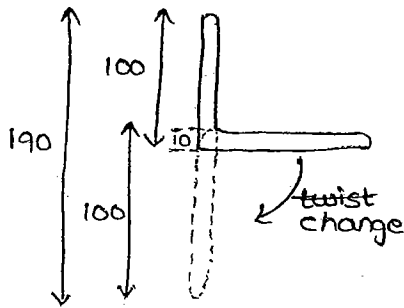
$$\eta^2 = \frac{I}{A}$$

$\frac{\text{Effective length}}{\text{Least radius of gyration } \eta} = \frac{L}{\eta} = \text{slenderness ratio.}$

Radius of gyration is more, slenderness ratio is less then $P_{c,n}$ value will be more.

Equal angles

ISA 100 X 100 X 10



$$A = 190 \times 10$$

$$= 1900$$

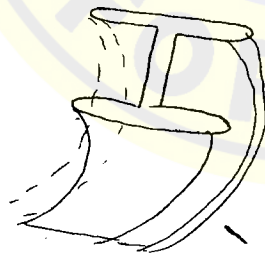
$V = A \times L \Rightarrow V \times \gamma = \text{Weight}$
 Radius of gyration = 19.1 mm
 about minor axis

$$(P_{cr})_{zz} = \frac{\pi^2 EA}{(L/\gamma_{zz})^2}$$

$$(P_{cr})_{yy} = \frac{\pi^2 EA}{(L/\gamma_{yy})^2}$$

$$(P_{cr})_{zz} > (P_{cr})_{yy}$$

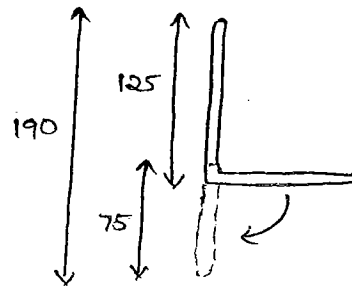
If $P > (P_{cr})_{yy}$ then section will be buckled



unequal angles

4

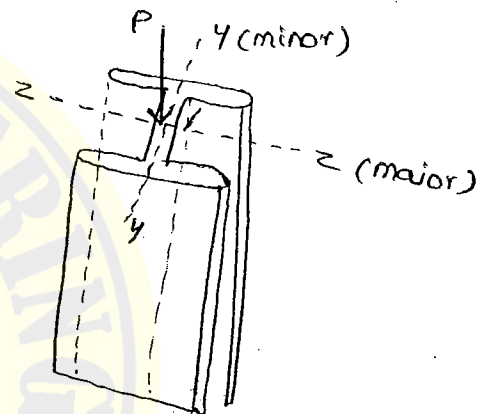
ISA 125 X 75 X 10



$$A = 190 \times 10$$

$$= 1900$$

Radius of gyration = 19.16 mm
 about minor axis



Note:-

1. Equal angle section provides higher minimum radius of gyration than unequal angle section, for same sectional area or weight. Hence equal angle section will have more axial compressive strength than unequal angle section.

2. Unequal angle section longer with its longer length connected to gusset plate will provide more axial tensile strength than equal angle.

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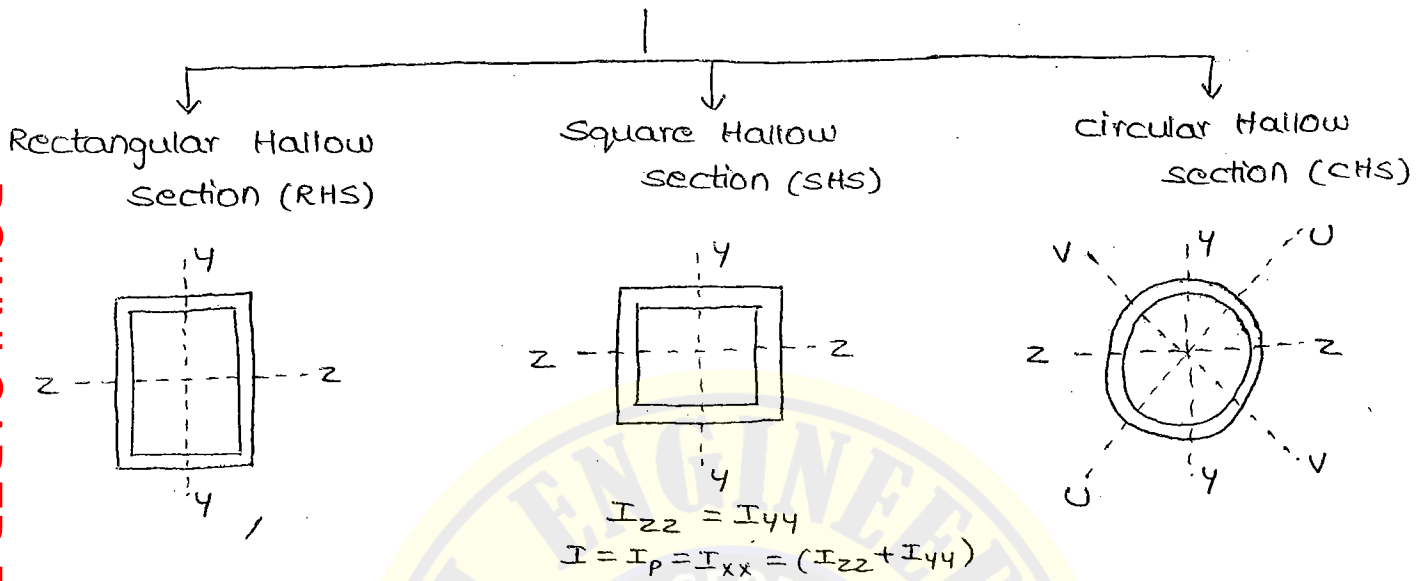
Rolled steel tube sections:-

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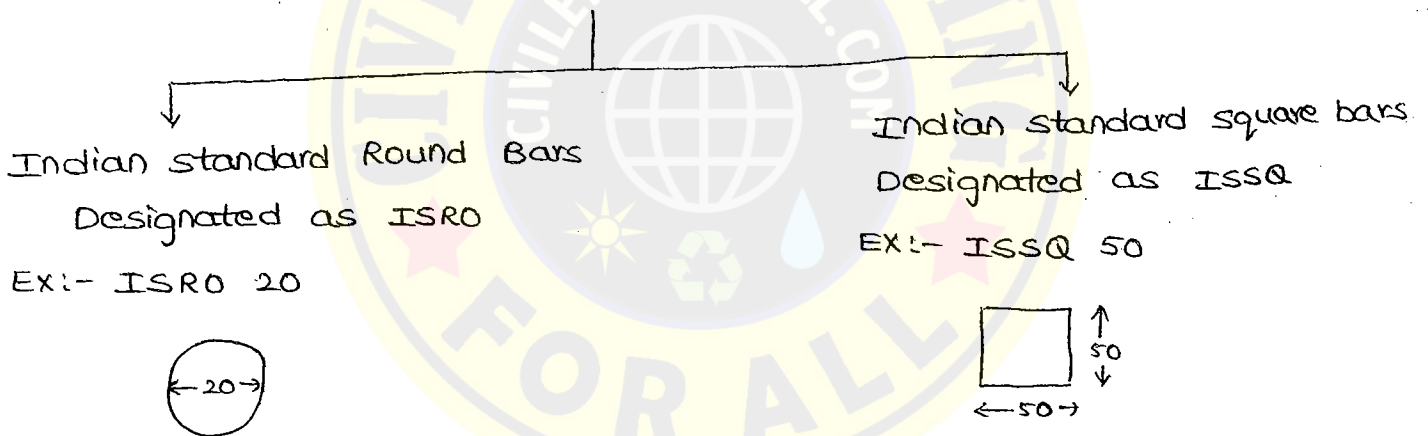
Tube sections are designated by its outside dimension and thickness

I_p = polar moment of inertia.

Types of tube sections



Rolled steel bars are two types



Note:-

1. Closed sections (Box sections) will provides higher torsional strength (or) twisting moment carrying capacity than open section with same sectional area (or) weight.
2. Solid circular sections and hollow circular sections are best sections (or) most efficient sections as a compression member such as column (or) strut.

For same weight (same cross sectional area) hollow circular section provides large value of radius of gyration than solid circular section. Hence Hollow circular sections are best sections as a compression member than solid circular section.

$\therefore (P_{cr})_{hollow} > (P_{cr})_{solid}$

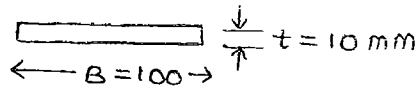


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Rolled steel Flats (when thickness, $t > 5 \text{ mm}$):-

Flat sections may be designated as its width and thickness.

Ex:- 100 ISF 10



Rolled steel plates (when $t \geq 5 \text{ mm}$):-

plates may be designated by its length, width and thickness.

Ex:- ISPL 7500 X 2500 X 10 mm
L X B X t

Rolled steel sheets (when $t < 5 \text{ mm}$):-

Ex:- ISSH 1000 X 600 X 2 mm
L X B X t

Analysis and Design:-

Method of Design:-

- | | | |
|--------------------------------|-----------------|-----------------------|
| 1. Working stress method (WSM) | } IS 800 : 1984 | → F.O.S |
| 2. Ultimate Load design (ULD) | | → L.F |
| 3. Limit state design (LSD) | } IS 800 : 2007 | → partial safety fact |

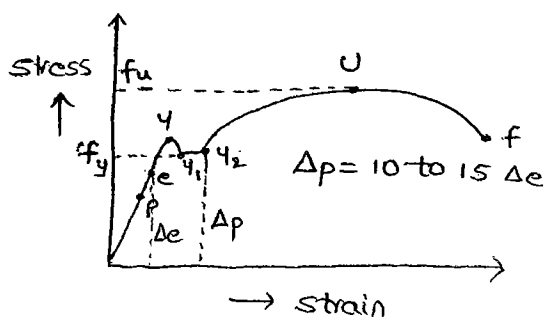
Safety Norms:-

$$\text{Factor of safety (FOS)} = \frac{\text{yield stress}}{\text{working stress}}$$

$$\text{Load factor (LF)} = \frac{\text{Ultimate load}}{\text{working (or) service load}}$$

Working stress method (WSM):- (IS 800 : 1984)

stress strain curve of mild steel:-



- P = proportionality limit
- e = elastic limit
- y = Upper yield point
- y₁ = Lower yield point
- y₂ = plastic yielding
- U = Ultimate stress point
- f = failure stress point.

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In working stress method it is assumed that the stress strain curve is linear upto yield point. In this method of design stresses are to be calculated for various critical load combinations and members are designed for working loads.

For safety of structural member working stresses due to various working load combination should be less than or equal to permissible stresses.

Permissible stresses is the fraction of yield strength of the material which is ratio between yield strength to Factor of Safety.

For safety of structural member design requirements

1. Working stress due to $D.L + L.L \leq$ permissible stresses
2. Working stress due to $D.L + W.L \leq$ permissible stresses
3. Working stress due to $D.L + L.L + W.L \leq 1.33 \times$ permissible stress

Permissible stresses (allowable stress) IS 800: 1984 :-

1. Permissible average shear stress = $0.4 f_y$ — $\frac{0.4 f_y}{0.4 f_y} = 2.5$ $\frac{F.O.S}{0.4 f_y}$
2. Permissible maximum shear stress = $0.45 f_y$ — $\frac{f_y}{0.45 f_y} = 2.11$
3. Permissible axial tensile stress = $0.6 f_y$ — $\frac{f_y}{0.6 f_y} = 1.67$
4. Permissible bending tensile (or) compressive stress = $0.66 f_y$
5. Permissible bearing stress = $0.75 f_y$
6. Permissible combined bending + Bearing stress = $0.90 f_y$

Minimum thickness of main structural (serviceability criteria) member :-

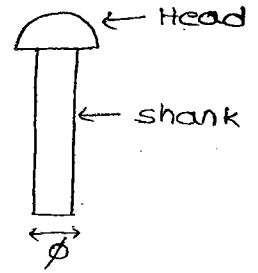
1. If steel member is directly exposed to weather accessible for cleaning and painting. The min. thickness of structural member should be more than or equal to 6 mm.
2. If steel member is directly exposed to weather and not accessible for cleaning and painting. The min. thickness is 8 mm.

RIVERTED AND BOLTED CONNECTIONS

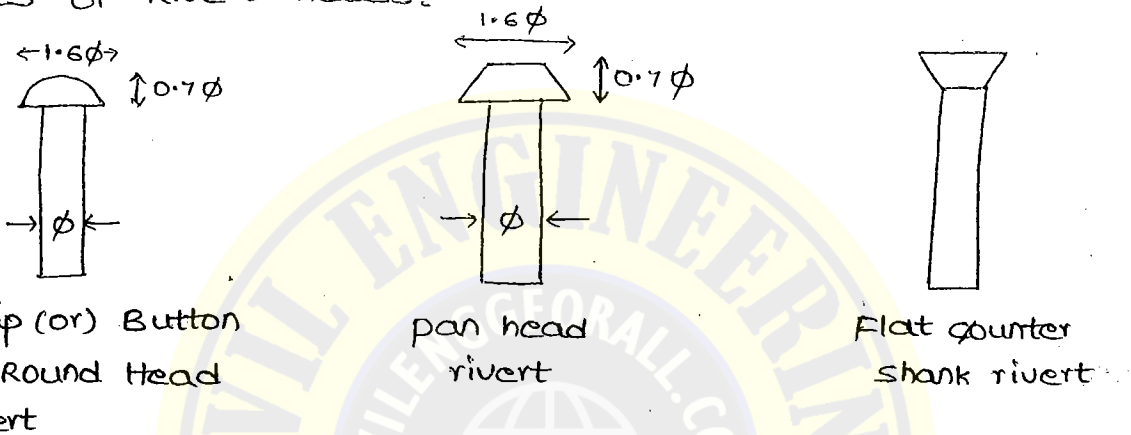
Rivert and Riverting:-

Rivert may be designated with its shank diameter (ϕ) is also called Nominal diameter (ϕ).

Rivert may consists of head and shank and are made of mild steel, or high tensile steel. The size of the rivert is the diameter of the Shank.



Types of Rivert heads:-



Snap (or) Button (or) Round Head Rivert

Pan head rivert

Flat counter Shank rivert



$l_g =$ grip length of rivert

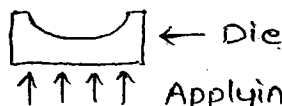
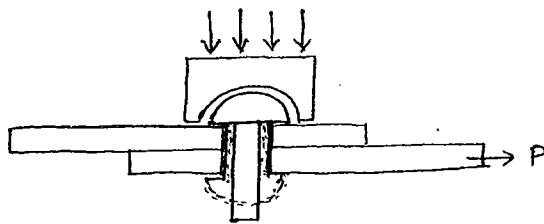
$\phi \leq 25 \text{ mm}, \quad d = \phi + 1.5 \text{ mm}$

$\phi > 25 \text{ mm}, \quad d = \phi + 2.0 \text{ mm}$

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Nominal diameter of rivert (ϕ):-

It is a diameter of rivert before riverting process which is also same as shank diameter of rivert.



Applying load then rivert compressed and filled the empty portion and form a Head at bottom side. This process is called Riverting process.

Assume nominal diameter of rivert by using UNWIN'S formula

$$\phi = 6.04 \sqrt{t} \quad (\phi, t \text{ are in mm})$$

$$\phi = 1.91 \sqrt{t} \quad (\phi, t \text{ are in cm})$$

t = thickness of the connected member

Gross (or) effective diameter of rivert (d):-

It is the diameter of rivert after riverting process which is also same as diameter of rivert hole.

$$d = \phi + 1.5 \text{ mm} \quad (\phi \leq 25 \text{ mm})$$

$$d = \phi + 2.0 \text{ mm} \quad (\phi > 25 \text{ mm})$$

For calculating the strength of rivert used gross (or) effective diameter in equation.

Classification of Riverts:-

1. Based on method of Driving force:-

a. power driven riverts (uniform pressure can be applied)

It have a approximately 20% higher strength than hand driven riverts.

b. Hand driven riverts (load may be sometimes eccentric)

2. Based on method of placing:-

a. Workshop riverts (Better quality control may be achieved for riverting)

b. Field riverts (or) site riverts (quality control may be poor).

Note:-

Field riverts have approximately 10% lesser strength than workshop riverts.

3. Based on method of heating:-

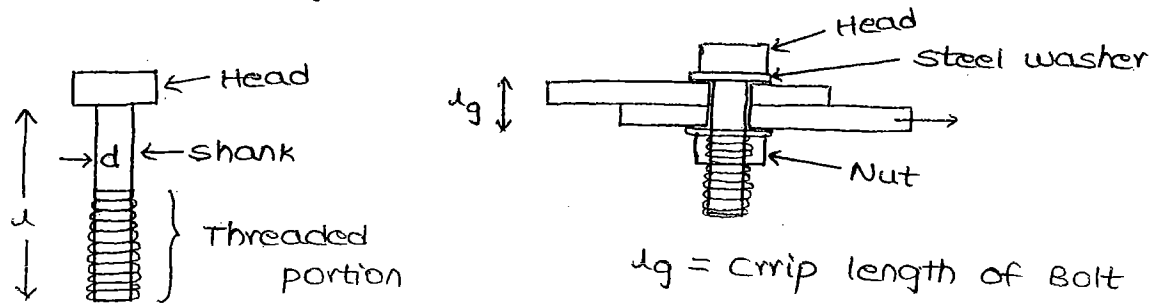
a. Hot driven riverts (riverts may be subjected to High temperature)

b. Cold driven riverts (riverting may be done at normal temperature)

Note:-

Cold driven rivert has relatively higher strength than Hot driven riverts.

Bolts and Bolting :-



Bolt may be designated as $M \cdot d \cdot L$

M = Metric size

d = Shank diameter of bolt

L = Length of bolt

Ex:- A bolt may be designated Hex bolt designated as $M16 \times 70$ refers [d]

(a) shank dia of bolt is 70 mm

(b) Length of bolt is 16 mm

(c) cross sectional area $16 \times 70 \text{ cm}^2$

(d) shank dia of bolt is 16 mm

Hex = Hexagonal head



Types of Bolts:-

1. Unfinished bolts (or) black bolts
2. Close tolerance bolts (or) finished bolts
3. High strength friction grip bolts (HSFGB)

$$d_o = d + 1.5 \text{ mm}$$

d_o = diameter of hole

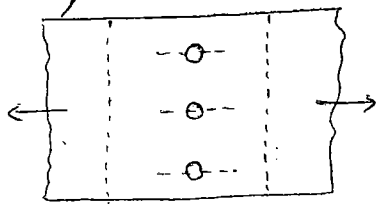
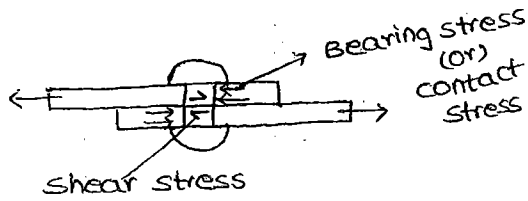
d = shank diameter

Types of bolted (or) joints (or) riveted connections :- (shear)

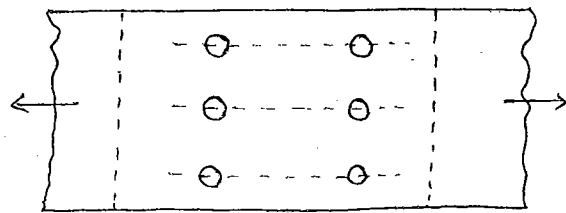
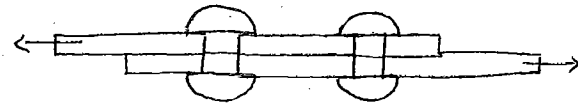
Bolted connection may be two types

1. Lap connection
2. Butt connection (or) Butt joint
 - a. single cover butt connection
 - b. Double cover butt connection

Lap connections (or) joints:-



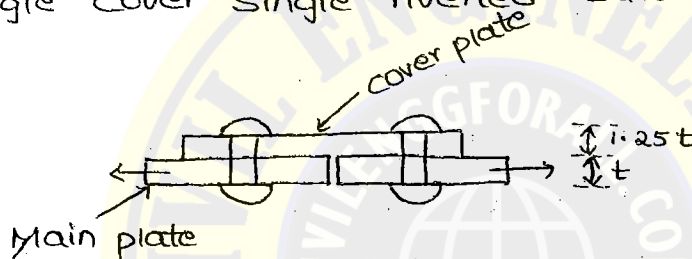
Single riveted lap connection



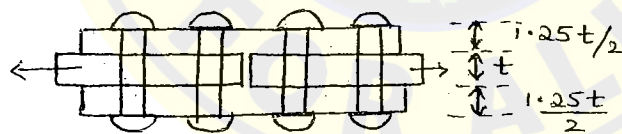
Double riveted lap connection

Butt connections (or) Butt joints:-

a. Single cover single riveted butt joint



b. Double cover double riveted butt joint



Eccentricity is zero in this connection.

Note:-

1. It is desirable to use double cover butt connection for the following reason.

a. In case of double cover butt connection the centre of gravity of load in one connected member is lying with centre of gravity of load in another connected member. Hence double cover butt connection is free from moment (eccentricity of load is zero).

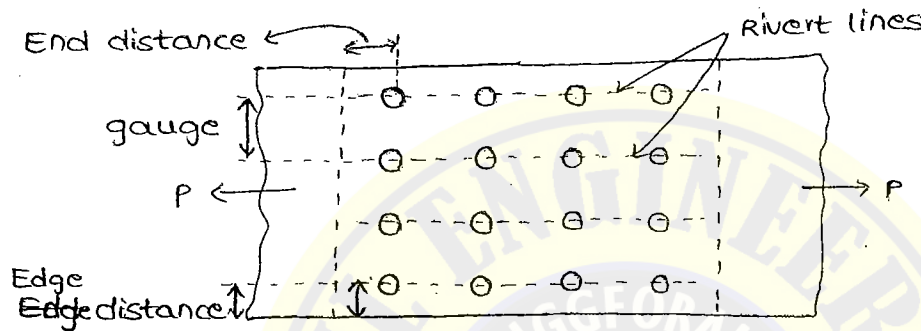
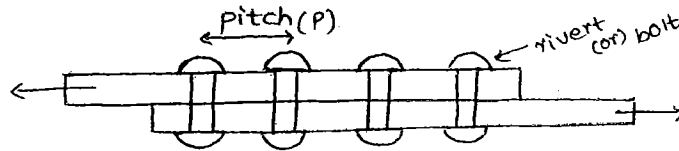
b. Eccentricity of load exist in lap connection or single cover butt connection.

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b. The shear strength of each rivert or bolt in double cover butt connection is 2 times higher than shear strength of each rivert or bolt in lap connection or single cover butt connection.

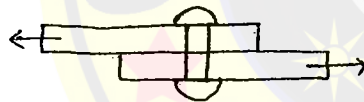
$$= 2 \times \frac{\pi (\phi)^2}{4}$$

(1M) ** Specifications for Riverted (or) Bolted connections :-



1. pitch (P)
2. gauge (g)
3. End distance
4. Edge distance

(Lap connection with chain pattern of riveting)



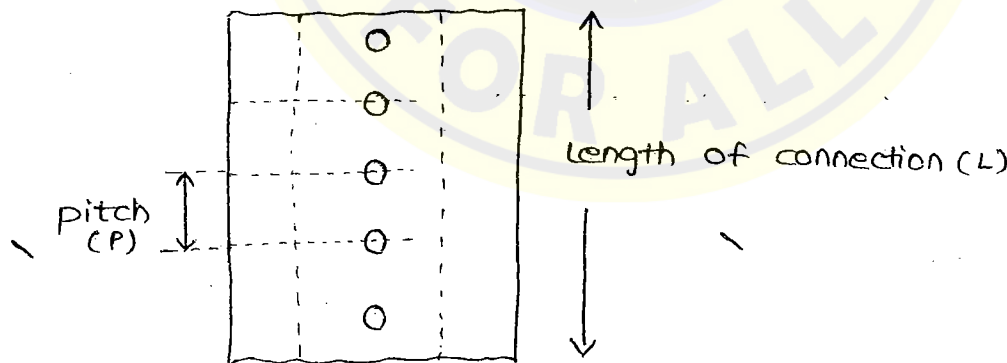
$$P = 1600 \text{ KN}$$

(pitch) $P_n = 100 \text{ KN}$

$$n = \frac{P}{P_n} = \frac{1600}{100}$$

$$n = 16 \text{ no. rivert}$$

$P_n = \text{Strength of one rivert}$



(Lap connection between wide plates)

pitch :-

It is a centre to centre distance between two consecutive riverts measure parallel to the direction of a load in a member

For wide plates it is centre to centre distance between two adjacent riverts measured along length of connection.

Gauge:-

It is centre to centre distance between two adjacent rivets measured perpendicular to the direction of load in a member (or) it is a distance between two adjacent rivet lines or bolt lines.

Minimum pitch (P_{min}):-

$$P_{min} \neq 2.5 \times \text{nominal dia of rivet } (2.5 \phi)$$

(or)

$$P_{min} \neq 2.5 \times \text{shank diameter of Bolt}$$

$$(2.5 \phi) P_{min} \leq P_o \leq P_{max}$$

P_o = Optimum pitch

Maximum pitch (P_{max}):-

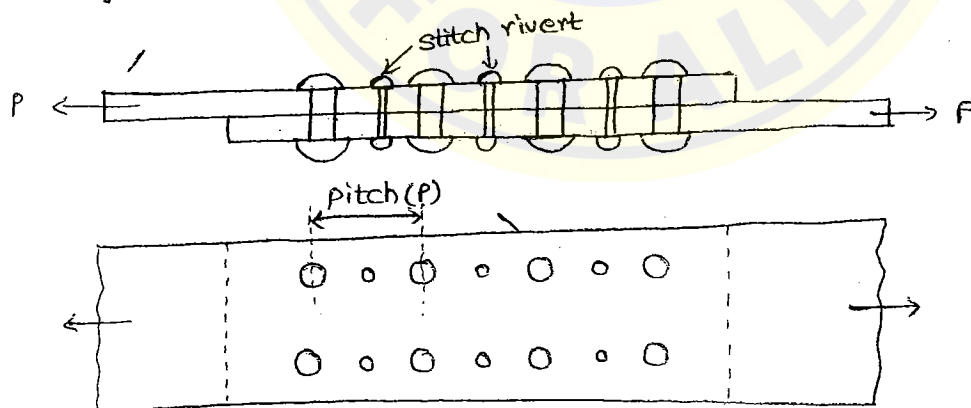
$$P_{max} = 12t \text{ (or) } 200 \text{ mm, whichever is less for compression member}$$

$$P_{max} = 16t \text{ (or) } 200 \text{ mm, whichever is less for tension member}$$

where,

t = thickness of the thinner connected member.

Tacking rivets or stitch rivets:-



P_n = strength of one rivet

If $P > P_n$ (pitch rivet) then buckling occur. we avoid such buckling, use stitch (or) Tacking rivets.

Maximum pitch for tacking rivets:-

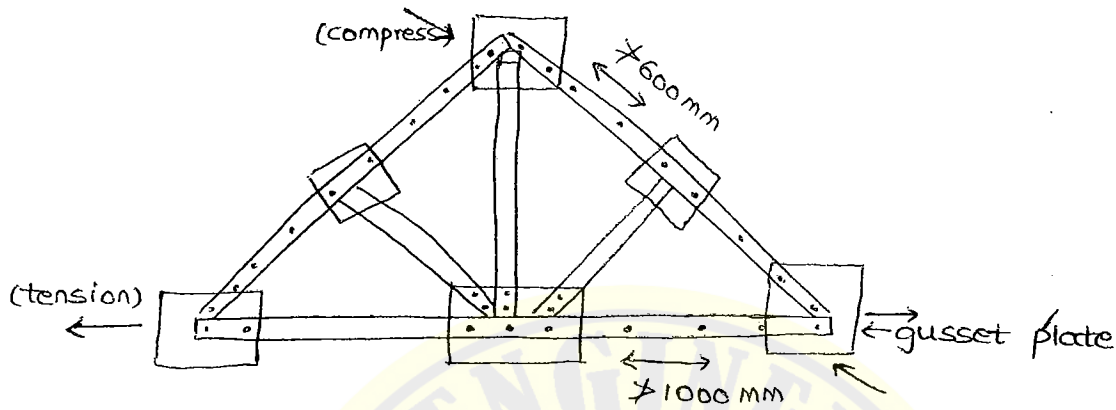
1. For plates:-

$$P_{max} = 32t \text{ (or) } 300 \text{ mm, whichever is less. when plates are not exposed to weather}$$

$P_{max} = 16t$ (or) 200 mm, whichever is less, when plates exposed to weather.

2. For Angle channels:-

$P_{max} = 600$ mm for compression member
 $= 1000$ mm for tension member



Note:-

Tacking rivert or stitch rivert will joint two or more sections together so that the joint sections may behaves as single rolled steel section.

End distance:-

It is the distance between centre of rivert hole to the nearest edge of a main member or cover plate measure parallel to the direction of a load in a member.

Edge distance:-

It is the distance between centre of rivert hole to the nearest edge of a main member (or) cover plate measured perpendicular to the direction of a load in a member.

Minimum end (or) edge distance (e_{min}):-

$$e_{min} \approx 1.5 \times \text{gross diameter of rivert (1.5d)}$$

(or)

1.5 x dia of bolt hole, for machine flame cut edge:

$$e_{min} \approx 1.7 \times \text{gross dia of rivert (or) dia of bott hole,}$$

For hand flame cut edges

Maximum end or edge distance (e_{max}):-

$$e_{max} = 4t + 37 \text{ mm}$$

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Failures of Riveted or Bolted connections:-

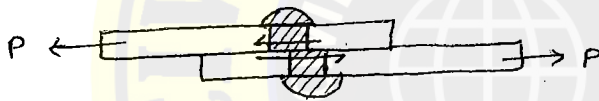
1. Shear failure of rivert (P_s)
2. Bearing failure of rivert (P_b)
3. Bearing failure of plate
4. shear failure of plate
5. splitting failure of plate (or) Edge cracking
6. Tearing failure of plate (P_t)

Note:-

By providing minimum end distance the following three failures can be eliminated.

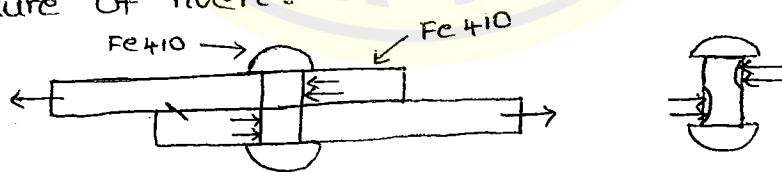
1. Bearing failure of plate
2. shear failure of plate
3. splitting failure of plate (or) edge cracking

Shear failure of rivert:-



These failure may occur when shear stresses in riverts due to applied loads may exceeds working shear stress (permissible shear stress or τ_{vt}) in riverts. shear stress are generated in riverts because plate due to applied forces.

Bearing failure of rivert:-



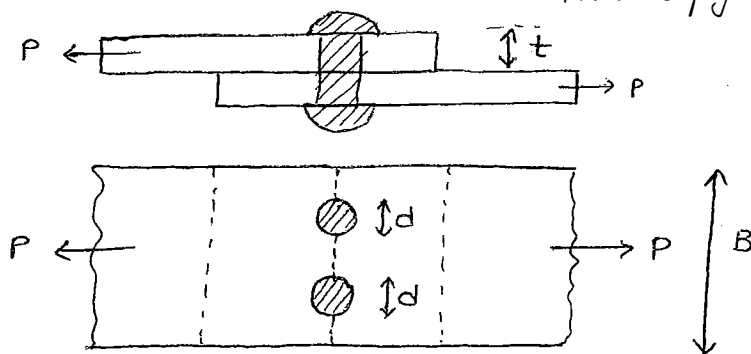
The half shank of the rivert may be crushed when plate may be strong in bearing and heavily bearing stress in plate may change to press the rivert shank.

This failure may not occur in practise except plates are strong in bearing

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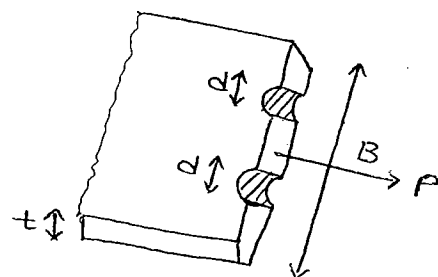
Tearing failure of plate:-

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$$\text{Net Area} = (B - 2d)t$$

$$\text{shear stress} = \frac{P}{A_{\text{net}}} = \frac{P}{(B - 2d)t}$$

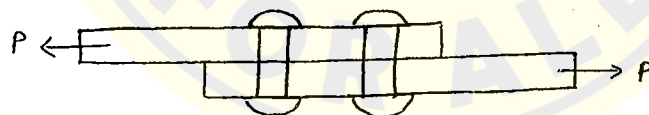


The tensile stress in plate at a net section may exceed working tensile stresses in the plate to cause this failure. This failure may also occur where rivets or bolts may be stronger than plates.

Note:-

This failure occurs where the net area is less than that portion caused failure.

Strength of riveted connection (or) joints (P_c (or) P_T):-



a. Shear strength of rivert (P_s):-

$P_s = \text{shear area} \times \text{permissible shear stress in rivert} (\tau_{vf})$

→ Shear strength of one rivert in single shear

$$P_s = \frac{\pi(d)^2}{4} \times \tau_{vf}$$

→ Shear strength of one rivert in double shear

$$P_s = 2 \times \frac{\pi d^2}{4} \times \tau_{vf}$$

→ Shear strength of riveted joint in single shear

$$P_s = n \times \frac{\pi d^2}{4} \times \tau_{vf}$$

$d = \text{Gross (or) effective dia. of rivert}$

→ Shear strength of riveted joint in double shear

$$P_s = 2 \times n \times \frac{\pi d^2}{4} \times \tau_{vf}$$

$n = \text{no. of riverts in joint}$

(b) Bearing strength of rivert (P_b):-

$$P_b = \text{bearing area} \times \text{permissible bearing stress in rivert } (\sigma_{pf})$$

→ Bearing strength of one rivert

$$P_b = (d \times t) \times \sigma_{pf}$$

→ Bearing strength of riverted joint

$$P_b = n \times (d \times t) \times \sigma_{pf}$$

d = Gross diameter of rivert

t = thickness of thinner bearing plate

n = no. of riverts in the joint.

(c) Strength of rivert (or) rivert value (P_n (or) R_v):-

P_n (or) R_v is smaller of P_s (or) P_b

P_s = shear strength of rivert

P_b = bearing strength of rivert

(d) NO. of riverts required to a support an axial load (P):-

$$n = \frac{\text{Ax. of axial load}}{\text{Strength of one rivert}} = \frac{P}{P_n}$$

(Rounded off to nearest higher value).

(e) Tearing strength (or) Tensile strength of plate (P_t):-

P_t = Net effective sectional area (A_{net}) × permissible axial tensile stress in plate (σ_{at})

$$P_t = A_{net} \times \sigma_{at}$$

$A_{net} = (B - nd) \cdot t$ — for chain riverting

B = width of the plate

n = no. of rivert holes at a section

d = Gross diameter of rivert.

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Strength of Riveted joint (P_j) (or) connection (P_c):-

$$P_c = P_j = \text{minimum of } P_s \text{ (or) } P_b \text{ (or) } P_t \\ = \text{minimum of } P_s \text{ and } P_t$$

Efficiency of riveted joint (or) connection (η) [or] percentage strength of riveted joint:-

It is the ratio between strength of riveted joint or connection to the strength of main plate (or) solid plate

$$\eta = \frac{\text{Strength of riveted joint}}{\text{Strength of main (or) solid plate}} \times 100 \rightarrow (\text{without any hole})$$

$$\eta = \frac{P_{rj}}{P_{mp}} \times 100$$

Strength of solid plate (or) main plate (P_{mp}) = $A_g \times \sigma_{at}$

A_g = gross sectional area of plate = $B \times t$

σ_{at} = permissible axial tensile strength in plate = $0.6 \sigma_{at}^{SS}$

$$\therefore P_{mp} = Bt \times \sigma_{at}$$

P.g NO:- 15

15. Given

$$P_t = 90000 \text{ N} \quad P_s = 60,000 \text{ N} \quad P_b = 1,20,000 \text{ N}$$

$$P_{mp} = 1,80,000 \text{ N}$$

P_c = minimum of P_s (or) P_b (or) P_t

$$P_c = 60,000 \text{ N}$$

$$\eta = \frac{P_c}{P_{mp}} \times 100 \\ = \frac{60,000}{180,000} \times 100$$

$$\eta = 33.33\%$$

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P.9 No: 14

1. Given $t = 1.0 \text{ cm}$

Maximum pitch of rivet for tension

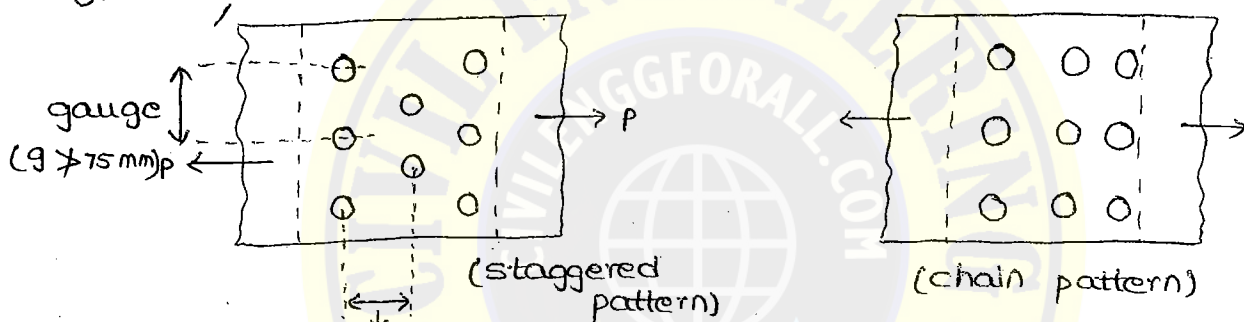
$$P_{\max} = 16t \text{ (or) } 200 \text{ mm (or) } 20 \text{ cm whichever is less}$$
$$= 16 \times 1$$
$$= 16 \text{ cm (or) } 20 \text{ cm, whichever is less.}$$

$$\therefore P_{\max} = 16 \text{ cm}$$

2. Minimum pitch of rivet ($\phi = 20 \text{ mm}$)

$$P_{\min} = 2.5\phi$$
$$= 2.5(20)$$
$$= 50 \text{ mm}$$

3.



Staggered pitch
(or)
Reeled pitch
(or)
Alternate pitch

$$P_{\max} \text{ (tension)} = 16t \text{ (or) } 200 \text{ mm}$$
$$P_{\max} \text{ (compression)} = 12t \text{ (or) } 200 \text{ mm}$$

increased \leftarrow Multiply by 1.5 for both

$$P_{\max} \text{ (tension)} = 24t \text{ (or) } 300 \text{ mm}$$
$$P_{\max} \text{ (compression)} = 18t \text{ (or) } 300 \text{ mm}$$

It may be increased by 50%.

$$7. \phi = 6.04 \sqrt{t} \text{ (}\phi, t \text{ are in mm)}$$

$$\phi = 1.91 \sqrt{t} \text{ (}\phi, t \text{ are in cm)}$$

Efficiency of riveted connection:-

$$\eta = \frac{\text{strength of the riveted connection}}{\text{Strength of the main plate}} \times 100$$

$$\eta = \frac{P_c}{P_{mp}} \times 100$$

P_c is minimum of $\underbrace{P_s \text{ (or) } P_b \text{ (or) } P_t}_{P_{sn} \text{ (or) } P_t}$

$$P_t = A_{net} \times \sigma_{at}$$

$$P_t = (B - nd) \times t \times \sigma_{at}$$

$$P_{mp} = B \times t \times \sigma_{at}$$

If $P_t \leq P_{sn}$,

$$\frac{P_c}{P_{mp}} \times 100 = \frac{P_t}{P_{mp}} \times 100$$

$$= \frac{A_{net} \times \sigma_{at}}{A_g \times \sigma_{at}} \times 100$$

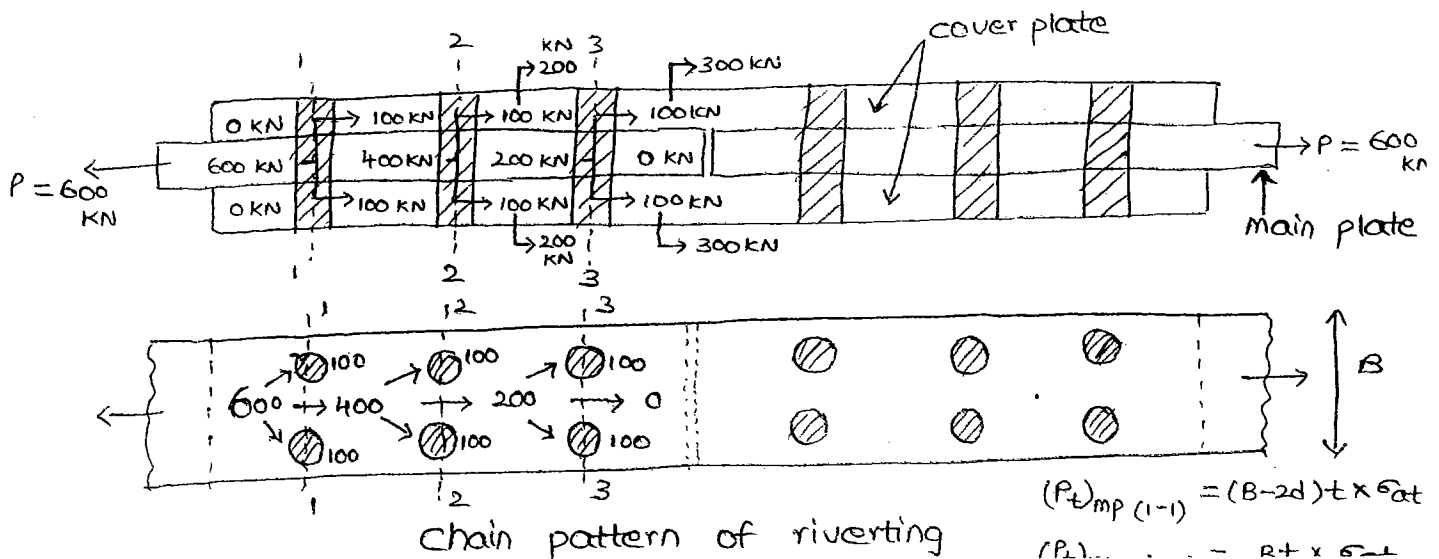
$$= \frac{(B - nd)t \times \sigma_{at}}{Bt} \times 100$$

If $P_t \leq P_{sn}$, $\eta = \left(\frac{B - nd}{B} \right) \times 100$

For diamond riveting, $\eta = \left(\frac{B - d}{B} \right) \times 100$, if $n=1$

EX:- Axial load $P = 600$ KN
 $P_{sn} = 100$ KN

$$\eta = \frac{P}{P_{sn}} = \frac{600}{100} = 6 \text{ no's}$$

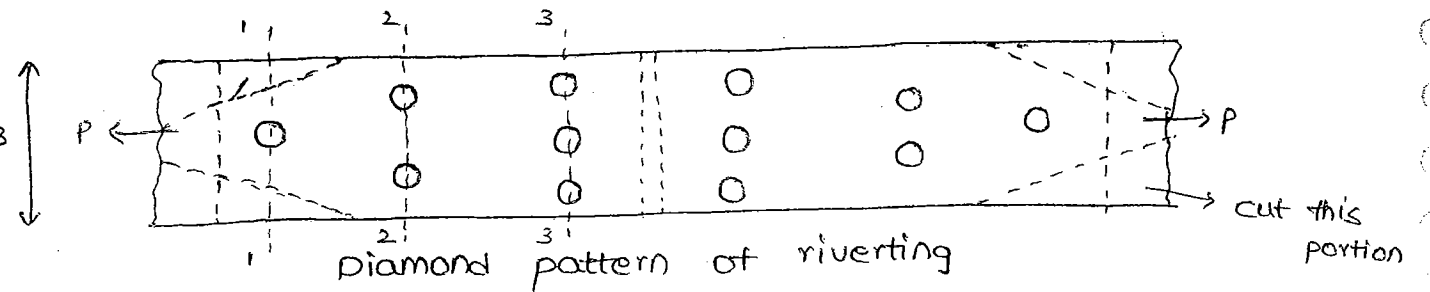
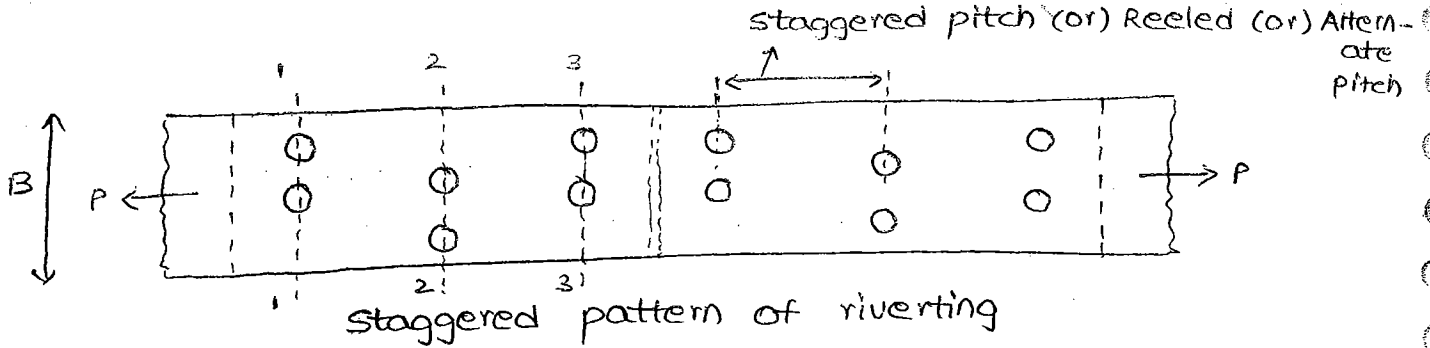


$$(P_t)_{mp (1-1)} = (B - 2d) \times t \times \sigma_{at}$$

$$(P_t)_{\dots} = B \times t \times \sigma_{at}$$

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Notes:-

1. It is desirable to use double cover butt connection with diamond cover riveting or bolting for the following reasons:
 - a. Efficiency of diamond pattern of riveting is more
 - b. Cover plate steel may be saved (weight of steel required for cover plate is less). Cover plate steel may be saved in diamond pattern of riveting.
 - c. Width of main plate required for diamond pattern of riveting is less.

$$B = \frac{P}{t \times \sigma_{at}} + nd$$

$n = 1$, for diamond pattern (At section ①-① only one rivert. so $n = 1$)

Permissible (or) Allowable stress in workshop riverts:-

Type of rivert	Axial tension (σ_{tf}) (Mpa (or) N/mm ²)	Shear (T_{vf}) (Mpa (or) N/mm ²)	Bearing (σ_{Pf}) (Mpa)
Power driven	100	100	300
Hand driven	80	80	250
Field riverts	90	90	290

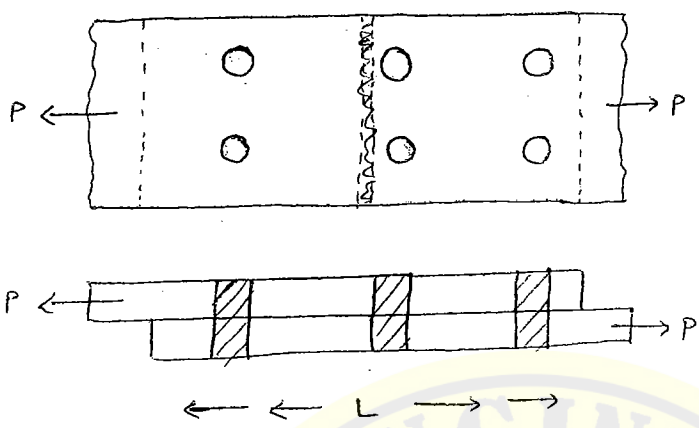
Note:-

For field riverts the above permissible stresses are decreased by 10%.

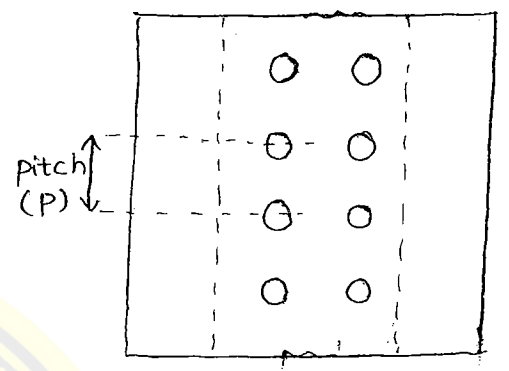
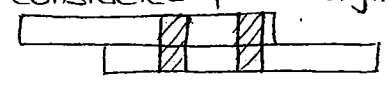
The above permissible stresses in rivets and bolts are to be increased by 25% when wind or earthquake loads are considered.

P.g No:- 11

1.



When width plate is very low then considered pitch length



$$P_s = n \times \frac{\pi}{4} d^2 \times \tau_{vf}$$

$$P_b = d \times t \times \sigma_{pf} \times n$$

$$P_t = (B - nd) t \times \sigma_{at}$$

$$P_t = (B - nd) t \times \sigma_{at}$$

$$= (P - 1 \times d) t \times \sigma_{at}$$

$$= (P - d) t \times \sigma_{at}$$

$\therefore p = \text{pitch}$

$n = \text{no. of rivets according to pitch}$
 $= (\frac{1}{2} \text{ rivert} + \frac{1}{2} \text{ rivert})$
 $n = 1 \text{ rivert}$

$$\eta = \left(\frac{B - nd}{B} \right) \times 100$$

$$= \left(\frac{P - d}{P} \right) \times 100$$

$$= \left(\frac{2.5 \phi - \phi}{2.5 \phi} \right) \times 100$$

$$= 60\%$$

10. Given

$$B = 30 \text{ cm}, \quad t = 10 \text{ mm} = 1 \text{ cm} \quad \phi = 18 \text{ mm}$$

$$d = 18 + 1.5 = 19.5 \text{ mm}$$

$$= 1.95 \text{ cm}$$

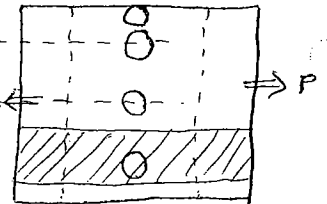
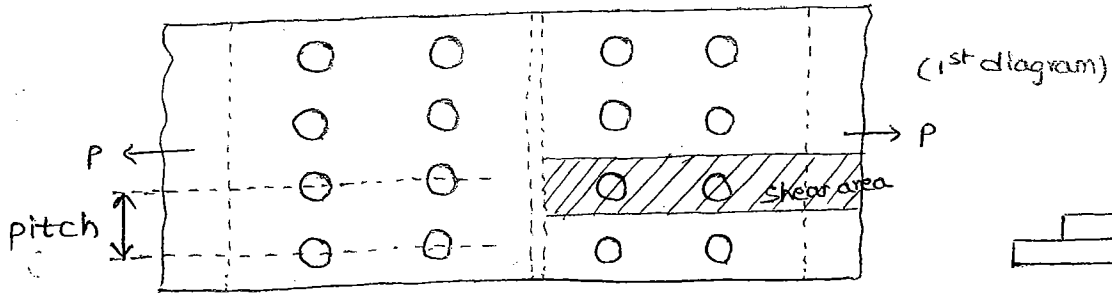
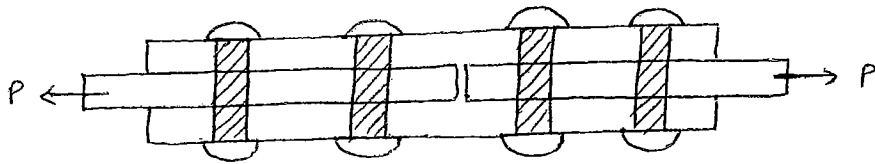
$$A_{net} = (B - d) t$$

$$= (30 - 1.95) 1$$

$$= 28.05 \text{ cm}$$

11. $P_{max} = 32t \text{ (or) } 300 \text{ mm}$
 $= 16t \text{ (or) } 200 \text{ mm}$

14.



$$P_s = 1 \times \frac{\pi}{4} d^2 \times \tau_{vf} \times n \quad [2^{nd} \text{ diagram}]$$

$$\therefore n = 4$$

$$P_s = 2 \times \underbrace{2 \times \frac{\pi}{4} d^2 \times \tau_{vf}}_{\text{shear area (2 rivets)}} \quad [1^{st} \text{ diagram}]$$

\downarrow
 $(4 \times \frac{1}{2} \text{ rivets})$ pitch length

15. Given

$$P_s = 60 \text{ KN}$$

$$P_b = 35 \text{ KN}$$

$$P_t = 70 \text{ KN}$$

$$P = 700 \text{ KN}$$

$$P_{sn} \text{ min of } P_s \text{ (or) } P_b = 35 \text{ KN}$$

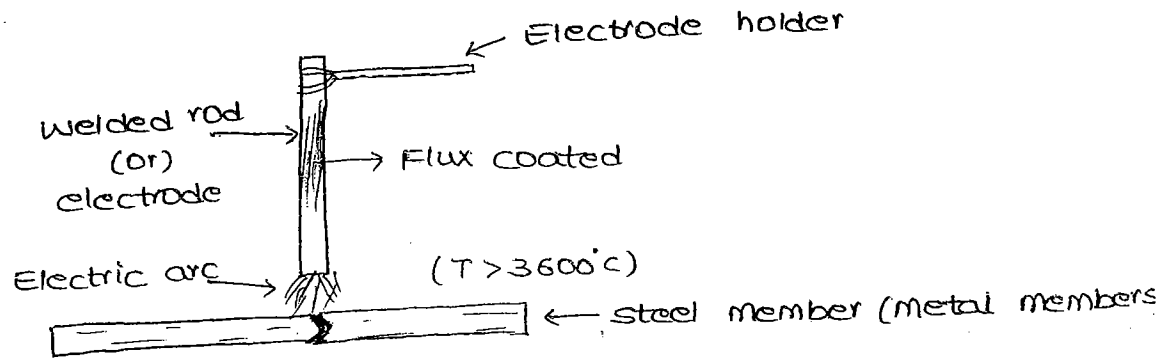
$$n = \frac{P}{P_{sn}}$$

$$= \frac{700}{35}$$

$$n = 20 \text{ nos}$$

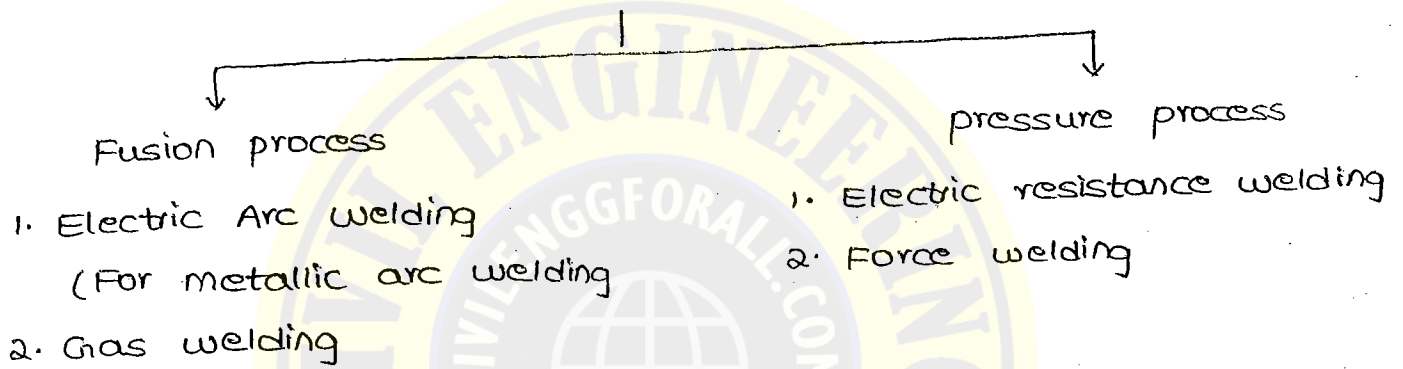
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WELDED CONNECTIONS



Welded classification:-

Welded classification



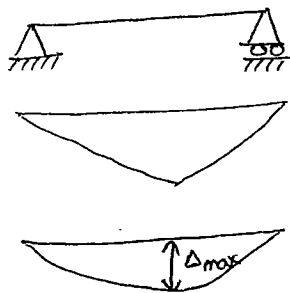
Most commonly used in steel construction is Fusion process.

Advantages:-

S.S. Beams

$$M = \frac{WL}{4}$$

$$\Delta_{max} = \frac{Wu^3}{48EI}$$

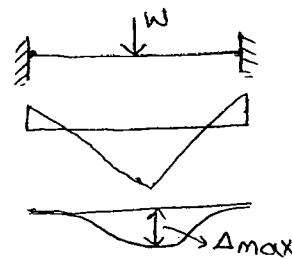


EX:- Bolted or rivert joints

Fixed beams

$$M = \frac{Wu}{8}$$

$$\Delta_{max} = \frac{Wu^3}{192EI}$$



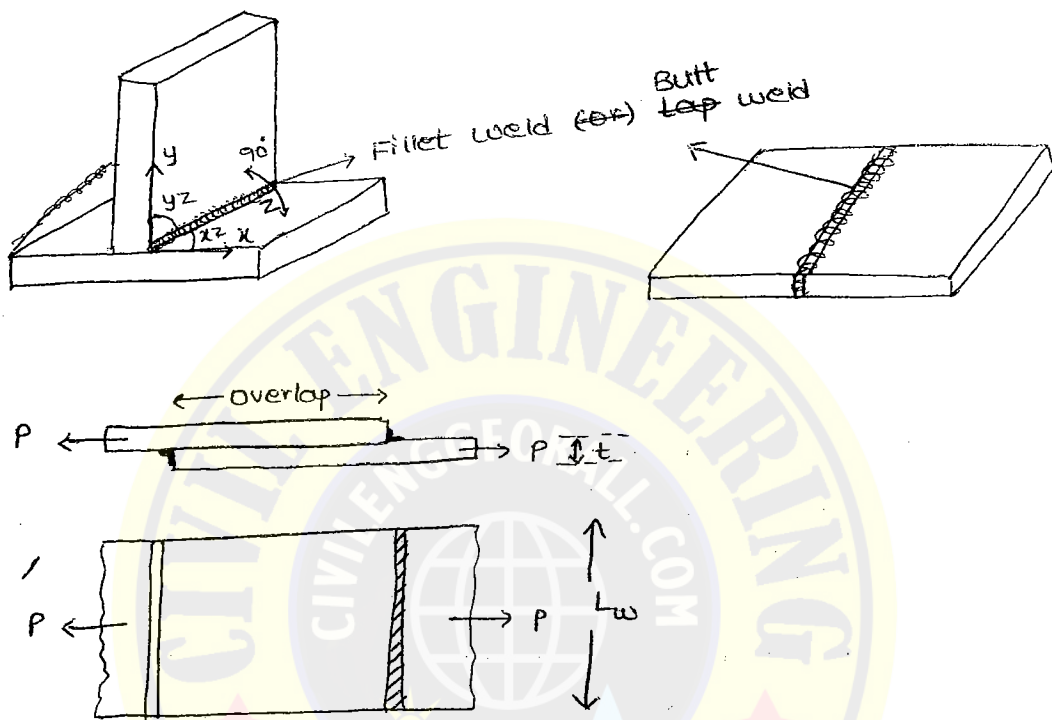
EX:- welded joints

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Type of welds:-

1. Fillet welds (or) Lap welds
2. Butt welds (or) Groove welds
3. Slot welds
4. plug welds

Design of fillet welds (or) Lap welds:-



Minimum overlap:-

1. $\nless 5t$ (IS: 800: 1984)
2. $\nless 4t$ (or) 40 mm whichever is more (IS 800: 2007)

Note:- t = thickness of thinner connected member

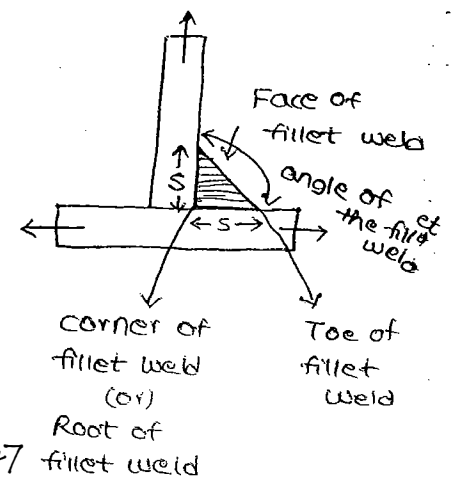
→ \nless standard cross section of fillet weld is Right angle triangle.

→ \nless Fillet weld symbol is

→ \nless standard angle for fillet weld is 45°

→ Fillet welds are to be provided for joining two different members in two different planes (or) when two different members which an angle.

→ Fillet welds are to be designed for shear stress only. Fillet welds may chance to fail in shear.



Size of fillet weld:- (s)

It is the distance between corner of fillet weld to the toe of fillet weld (or) it is minimum leg length of the fillet weld cross section.

$$S_{min} \leq s \leq S_{max}$$

Minimum size of fillet weld (S_{min}):-

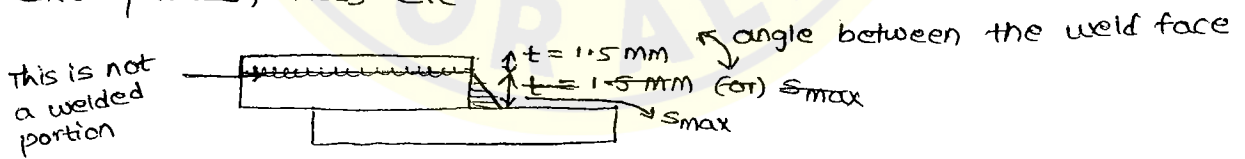
S_{min} depends on thickness of thinner connected member if thickness of thicker connected member.

Over (mm)	Upto & Including (mm)	S_{min}
0	10 mm	3 mm
10 mm	20 mm	5 mm
20 mm	32 mm	6 mm
32 mm	50 mm	8 mm

Maximum size of fillet weld (S_{max}):-

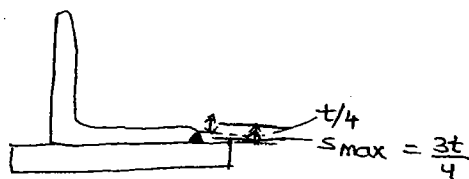
S_{max} depends on

1. Type of edge of a member
Ex:- Round edge (or) square edge
2. Thickness of thinner connected member
3. S_{max} for square edges
Ex:- plates, flats etc



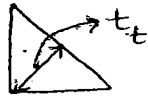
$$S_{max} = t - 1.5 \text{ mm (for square edges)}$$

4. S_{max} for round edges like edge of an angle (or) Flange of an I-section



Effective throat thickness (t_t) :-

It is distance between corner of the fillet weld to the face of the fillet weld. Throat is the weakest plane in the fillet weld cross section.



$$t_t = k \times \text{size of fillet weld}$$

$$t_t = k \cdot s$$

'k' depends on angle between the weld faces (or) fusion faces (α) and size of fillet weld.

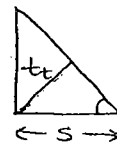
Angle between weld faces (α) value of k.

$60^\circ - 90^\circ$	0.70
$91^\circ - 100^\circ$	0.65
$101^\circ - 106^\circ$	0.60
$107^\circ - 113^\circ$	0.55
$114^\circ - 120^\circ$	0.50

$$\alpha = 90^\circ$$

$$\sin 45^\circ \cdot t_t = \frac{s}{\sqrt{2}} \quad (\text{or}) \quad \sin 45^\circ = \frac{t_t}{s}$$

$$t_t = 0.707s$$



EX:- The effective throat thickness of fillet weld as shown in fig is.

- a) $0.7s$ b) $0.65s$ c) $0.60s$ d) $0.55s$

$$\alpha = 99^\circ$$

$$t_t = \frac{s}{\sqrt{2}}$$

$$t_t = 0.64s$$

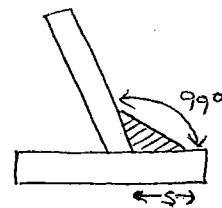
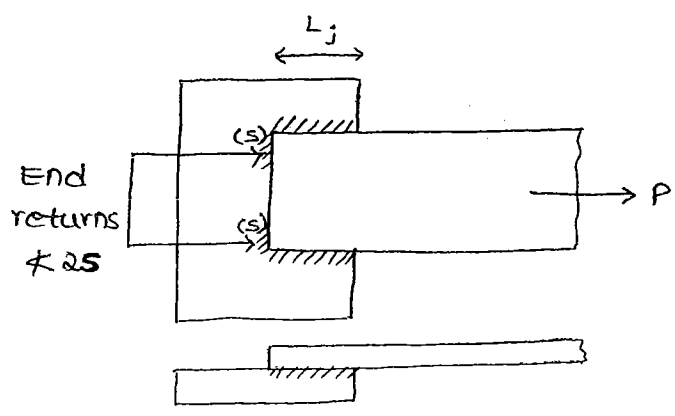


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Effective length of the fillet weld (L_w):-



L_j = Length of the side fillet (length of Lap)

L_w = Effective length of fillet weld

L = Overall length of fillet weld (or) Total length of fillet

s = size of the fillet weld.

Note:-

End returns to be provided at terminating ends not less than 2 times size of weld will minimize stress concentration due to tensile loads or Bending moment.

$$L = L_w + 2s$$

$$L_w = L - 2s$$

$L_w \neq 4s$ (or) 40mm whichever is more

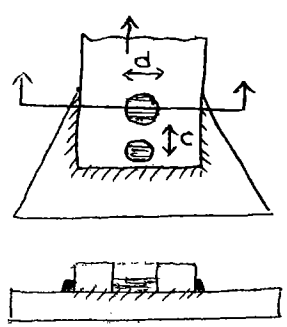
Shear strength of fillet weld (P_s):-

P_s = Effective shear area x permissible shear stress in fillet weld (τ_{vf})

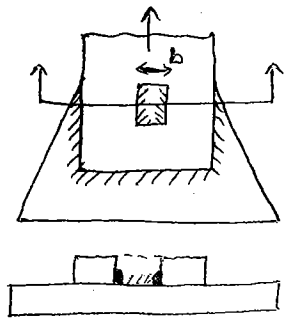
$$P_s = L_w \cdot t_t \times \tau_{vf}$$

$$P_s = L_w (ks) \times \tau_{vf}$$

Design of slot and plug welds:-



plug weld Section A-A



Section A-A slot weld



- d = diameter of slot
- b = width of the slot
- R = corner radius of slotted hole
- c = clear distance between the slotted holes
- t = thickness of the member which is having slot (or) hole

Notes:-

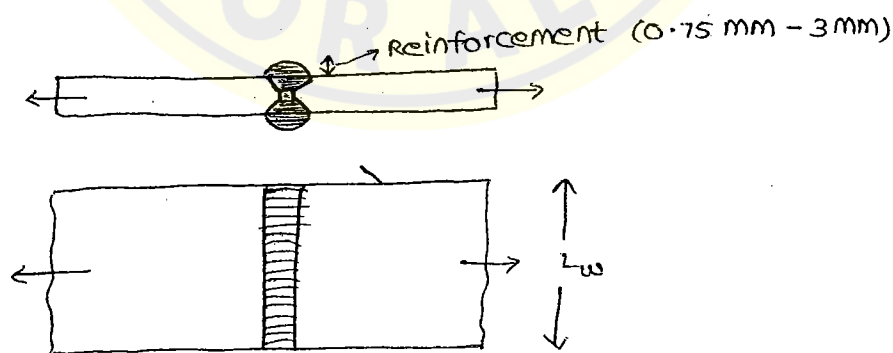
Plug welds and slot welds may be used along with fillet weld when sufficient length of weld not available to have more shear strength for supporting given load.

plug welds and slot welds will reduce unsupported dimension between fillet weld so that local buckling failure may be checked.

Width of the slot or diameter of the slot should be more than are equal to 25 mm and also should be more than are equal to $3t$

1. b (or) $d \geq 25$ and also $\geq 3t$ whichever is more
2. $R \geq 1.5t$ and also ≥ 12 mm whichever is more
3. $c \geq 2t$ and also ≥ 25 mm whichever is more

Design of butt (or) Groove welds :-



Butt (or) Groove weld

Types of butt welds

Partially penetrated butt welds

(or) single butt welds

Ex:- single 'V', single 'U',
Single 'J' welds etc

Fully penetrated butt welds

(or) Double butt welds

Ex:- Double 'U', Double 'V',
Double 'J', butt welds etc

Type of Butt weld

Diagrammatic Representation

Symbol

Throat (t_e) thickness

Single 'v'



∇

$\frac{5}{8}t$

Single 'u'



\cup

$\frac{5}{8}t$

Double 'v'



\times

t

Double 'u'



\times

t

Axial strength of butt weld:- (T_s):-

$T_s = \text{Effective sectional area} \times \text{permissible axial stress } (\sigma_{tf})$

$$T_s = L_w \cdot t_e \times \sigma_{tf}$$

L_w = effective length of the butt weld

t_e = effective throat thickness

= $\frac{5t}{8}$ for single butt welds like single v, single U.

$t_e = t$, for double butt welds like double v, double U

t = thickness of the thinner connected member

Permissible stress in weld for mild steel members:-

1. permissible shear stress in weld $\tau_{wf} = 108 \text{ Mpa}$
2. permissible axial stress in weld, $\sigma_{tf} = 150 \text{ Mpa}$
3. permissible bending compressive (or) bending tensile stress in welds $\sigma_{bc} = \sigma_{bt} = 165 \text{ Mpa}$.

Note:-

The above permissible stresses are increased by 25% if wind (or) earth quake loads are to be used in design calculations.

The above permissible stresses are decreased by 25% for site welding (or) field welding

P.9 No:-22

1. Effective sectional (shear) area = $L_w \times t_t$
 $= L_w \times ks$ $\alpha = 75^\circ$
 $= L_w (0.75) \leftarrow \text{standard value}$
 $= 0.75 \times L_w$

4. $L_w = 200 \text{ mm}$ $\sigma_{tf} = 150 \text{ Mpa}$ $t = 12 \text{ mm}$

Throat thickness, $t_e = \frac{5}{8}t$

$= \frac{5}{8} \times 12 = 7.5$

Axial strength of Butt weld, $P_t = L_w \times t_e \times \sigma_{tf}$

$= 200 \times (7.5) \times 150$

$= 225 \text{ KN}$

5. Given $s = 3 \text{ mm}$ $\tau_{vf} = 110 \text{ mpa}$

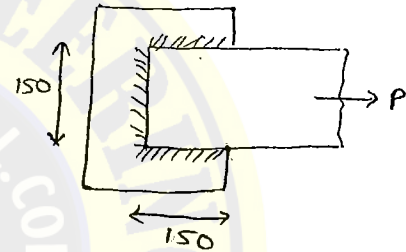
Service load allowed,

$P = P_s = L_w \times t_t \times \tau_{vf}$

$= (3 \times 150) \times (0.7 \times 3) \times 110$

$= 103.95 \times 10^3 \text{ N}$

$= 103.95 \text{ KN}$



$P_s = \text{shear area along failure plane}$

6. $t_1 = 10 \text{ mm}$ $t_2 = 20 \text{ mm}$ $t_e = t = 10 \text{ mm}$

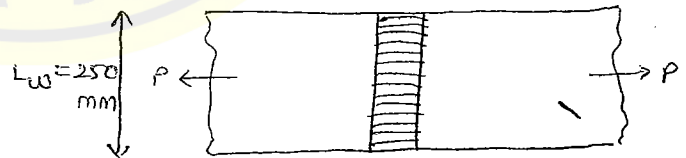
$\sigma_{tf} = 150 \text{ mpa}$

Load = $P = T_s = L_w \times t_e \times \sigma_{tf}$

$= 250 \times 10 \times 150$

$= 375 \times 10^3 \text{ N}$

$= 375 \text{ KN}$



7. $\tau_{vf} = 108 \text{ Mpa}$ $t = 16 \text{ mm}$

Minimum size of fillet weld

$s_{min} = s = 5 \text{ mm}$

$t_t = ks$

$= 0.707 \times 5$

$= 3.5 \text{ mm}$

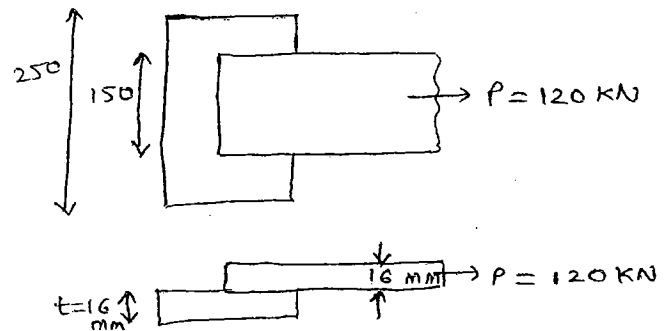


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$$\text{If } P \leq P_s$$

$$P = P_s$$

$$= L_w \cdot t_e \cdot \tau_{vf}$$

$$= L_w \cdot k_s \cdot \tau_{vf}$$

By equating $P = P_s$

$$120 \times 10^3 = L_w \times (0.707 \times 5) \times 108$$

$$L_w = 314.42 \text{ mm}$$

8.

$$L_w = L_j + L_j + 100 + 100$$

$$= 2L_j + 200 \text{ mm}$$

Equating load $P \leq P_s$

$$P = P_s$$

$$P = L_w t_e \tau_{vf}$$

$$\text{Overlap } L_j = \frac{L_w - 200}{2}$$

Factored load = $\gamma_L \times$ service load (or) working load

$$= \gamma_L \times \frac{400}{\gamma_L} \Rightarrow \gamma_L \times \frac{400}{1.5} \Rightarrow 266.67 \text{ kN}$$

Maximum size of weld which is depending on thinner connected member

$$S_{max} = t - 1.5$$

$$= 10 - 1.5$$

$$= 8.5 \text{ mm}$$

By equating $P = P_s$

$$266 \times 10^3 = L_w t_e \tau_{vf}$$

$$= L_w k_s \tau_{vf}$$

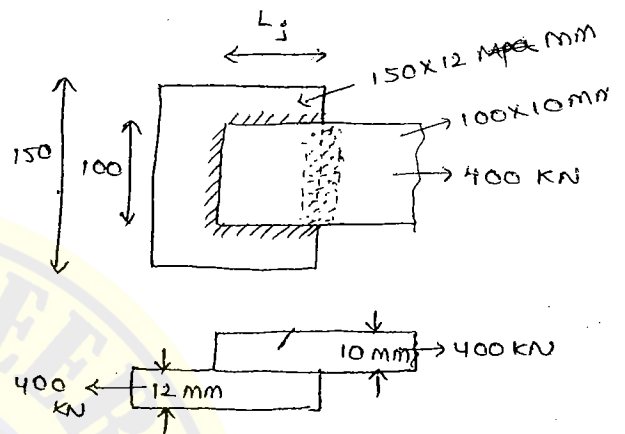
$$= L_w \times (0.707 \times 8.5) \times (108)$$

$$L_w = 414.9 \text{ mm}$$

$$\text{Overlap, } L_j = \frac{L_w - 200}{2}$$

$$= \frac{414.9 - 200}{2}$$

$$= 110 \text{ mm}$$



10. Allowable stress in plate,

$$\sigma_{st} = 150 \text{ Mpa}$$

$$t = 20 \text{ mm}$$

$$B = 150 \text{ mm}$$

Size of weld, $s = 12 \text{ mm}$

Throat thickness, $t_t = k_s$

$$= 0.7 \times 12$$

$$= 8.4 \text{ mm}$$

Average shear stress in weld = $\frac{P}{\text{Shear area}}$

$$= \frac{P}{L_w \cdot t_t}$$

Full strength of plate, $P_t = P = A_{\text{gross}} \times \sigma_{at}$

$$= (150 \times 20) \times 150$$

$$= 450 \times 10^3 \text{ N}$$

Shear area of fillet weld

$$A_e = L_w \times t_t$$

$$= \left[\left[2(150 + 70.72) + 50 \right] \times \pi(60) \right] \times (8.4)$$

$$A_e = 5711 \text{ mm}^2$$

Avg. shear stress in fillet weld $\tau = \frac{P}{A_e}$

$$= \frac{450 \times 10^3}{5711}$$

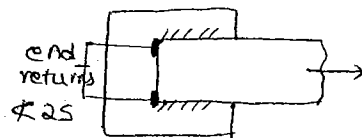
$$= 78.79 \text{ N/mm}^2$$

$$\tau = 78.79 \text{ N/mm}^2$$

P.9 NO:-23

$$1. L = L_w + 2s$$

$$L_w = L - 2s$$



$$3. \sin 45 = \frac{t_t}{s}$$

$$\Rightarrow t_t = s \cdot \sin 45$$

$$= \frac{s}{\sqrt{2}}$$

$$= 0.707s$$

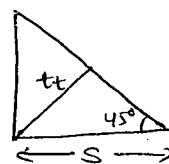


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4. Allowable stress in plate

$$\sigma_{at} = 200 \text{ Mpa}$$

Allowable stress in weld $\tau_{vf} = 100 \text{ Mpa}$

Size of weld, $s = 6 \text{ mm}$

Throat thickness, $t_t = k s$

$$= 0.7 \times 6$$

$$= 4.2 \text{ mm}$$

$$L_w = L_j + L_j + 100$$

$$L_w = 2L_j + 100$$

$$L_j = \frac{L_w - 100}{2}$$

Strength of 150 mm plate = $A_g \times \sigma_{at}$ (Bt)

$$= 150 \times 10 \times 200$$

$$= 300 \times 10^3 \text{ N}$$

Strength of 100 mm plate = $100 \times 100 \times 200$

$$= 200 \text{ kN}$$

\therefore safe load allowed is $200 \times 10^3 \text{ N}$

By equating $p = P_s$

$$= L_w \cdot t_t \cdot \tau_{vf}$$

$$200 \times 10^3 = (2L_j + 100) \times 4.2 \times (100)$$

$$L_w = 476.19 \text{ mm}$$

$$L_j = \frac{L_w - 100}{2}$$

$$= 188 \text{ mm}$$

11. Given $s = 6 \text{ mm}$

$$t_t = 0.707 s$$

$$= 0.707 \times 6$$

$$= 4.24 \text{ mm}$$

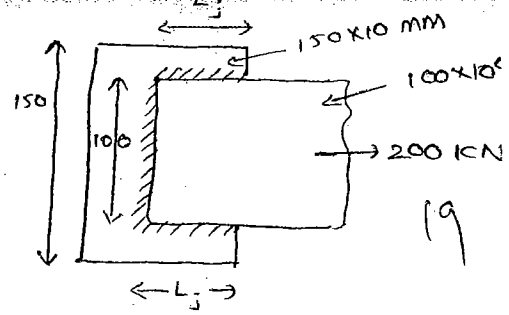
12. $L_w = 100 \text{ mm}$, $t_e = 8 \text{ mm}$, $\sigma_{tt} = 150 \text{ N/mm}^2$

Strength of butt weld, $T_s = L_w t_e \sigma$

$$T_s = 100 \times 8 \times 150$$

$$= 120 \times 10^3 \text{ N}$$

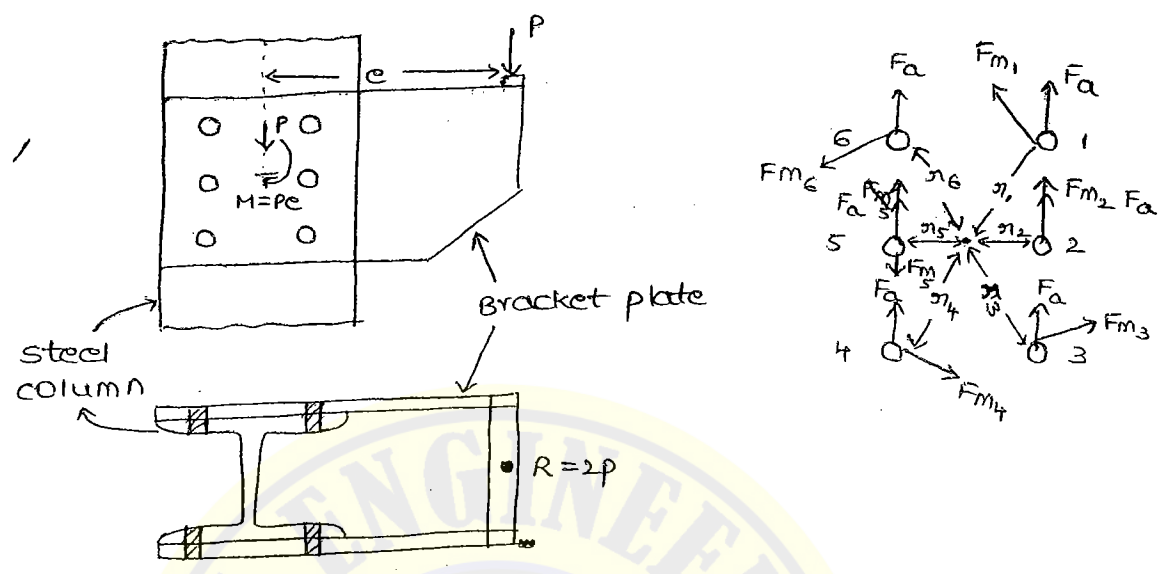
$$= 120 \text{ kN}$$



UNIT - 4

ECCENTRIC CONNECTIONS

1. Bracket type Riveted connection - I (Elastic Analysis) :-



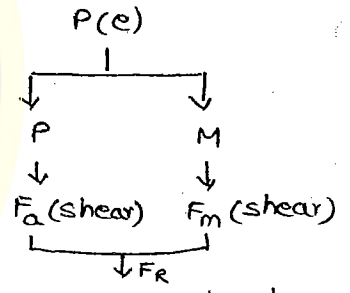
When load (or) moment is lying in the plane of Rivet (or) Bolt group.

Rivet group is subjected to

1. Direct Eccentric load (P)
2. Twisting moment (M = Pe)

p = Eccentric load

e = distance between the C.G. of Bolt (or) Rivet group to the applied load line.



→ Vertical shear force in any rivet (or) Bolt due to 'P' is F_a

$$F_a = \frac{P}{n}$$

n = no. of rivets or bolts present in the connection.

→ Shear force in each rivet (or) bolt due to twisting moment (M) is F_m .

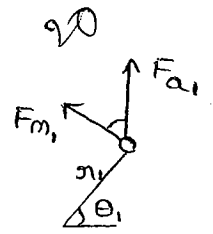
$$F_m = \frac{M \cdot r}{\sum r^2} = \frac{P \cdot e \cdot r}{\sum r^2}$$

$$\sum r^2 = r_1^2 + r_2^2 + r_3^2 + \dots + r_n^2$$

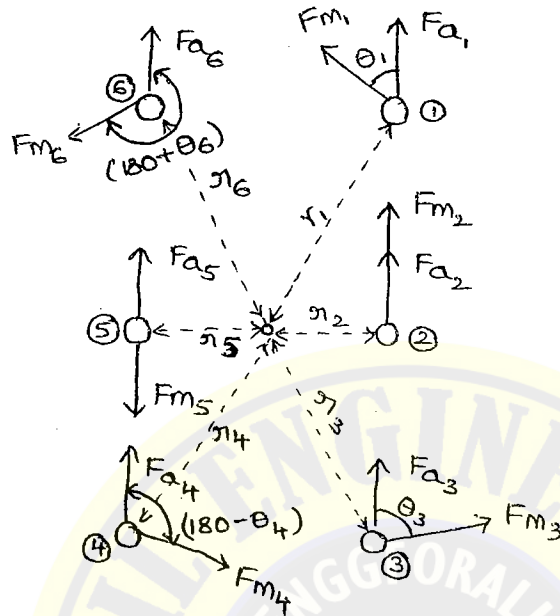
r = radial distance from C.G. rivet group to the rivet.

→ Resultant force between F_a and F_m is F_R

$$F_R = \sqrt{F_a^2 + F_m^2 + 2F_a F_m \cos \theta}$$



Condition for maximum resultant shear force (F_{Rmax}):-



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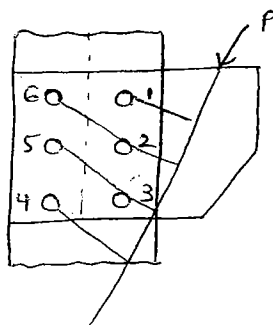
Critical bolt (or) critical rivet is one which is subjected to maximum resultant shear force is farthest from c.g. of rivet group, (r is maximum) and which may be close to the applied load line.

$r \rightarrow$ maximum (1, 3, 4, 6)

$\theta \rightarrow$ minimum (1, 3)

Hence critical rivets (or) Bolts are Rivet/bolt no. 1 & 3

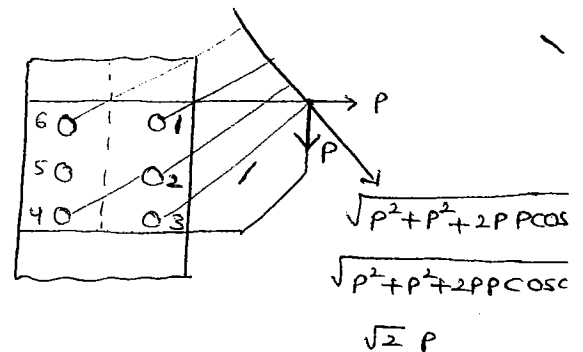
EX:-



$r \rightarrow$ maximum (1, 3, 4, 6)

$\theta \rightarrow$ minimum (3)

critical rivet: rivet '3'.



$r \rightarrow$ maximum (1, 3, 4, 6)

$\theta \rightarrow$ minimum (1)

critical rivet: Rivet no. 1

For safety of Rivert group:-

Design requirement:-

$$(FR)_{\max} \leq P_n = \text{Rivert value (or) strength of one rivert}$$

$$P_n = \text{minimum of } P_s \text{ or } P_b$$

$$P_s = \text{shear strength of one rivert}$$

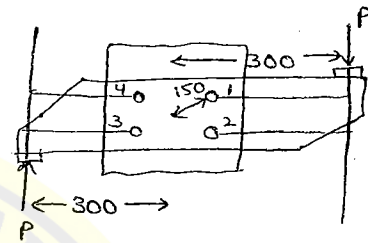
$$P_b = \text{bearing strength of one rivert}$$

P.g NO:- 29

1. $n \rightarrow$ maximum (1, 2, 3, 4)

$\theta \rightarrow$ minimum (1, 2, 3, 4)

critical riverts are Rivert NO. 1, 2, 3, 4

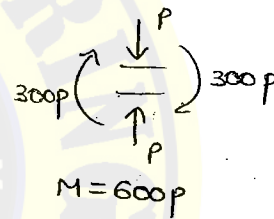


$$F_a = \frac{P}{n} \quad P=0 \quad F_a=0$$

$$F_m = \frac{Mn}{\sum n^2}$$

$$= \frac{600P(150)}{4 \times 150^2}$$

$$= P$$



$$FR_{\max} = FR_1 = FR_2 = FR_3 = FR_4$$

$$= \sqrt{F_a^2 + F_m^2 + 2F_a F_m \cos \theta}$$

$$= \sqrt{F_m^2}$$

$$= F_m$$

$$FR_{\max} = F_m = P \quad (\text{since } F_a=0)$$

5. Given Moment, $M = 80 \times 10^3 \text{ KN-mm}$

$$r_A = \sqrt{\left(\frac{120}{2}\right)^2 + \left(\frac{160}{2}\right)^2}$$

$$= 100 \text{ mm}$$

$$P=0, \quad F_a=0$$

$$F_{m_a} = \frac{M r_A}{\sum r^2}$$

$$= \frac{80 \times 10^3 \times 100}{4 \times 100^2}$$

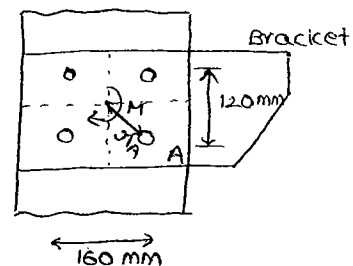


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$$\therefore (FR)_A = F_{m_a} = 200 \text{ KN} \quad (\text{since } F_a=0)$$

6. Critical bolts

$n \rightarrow$ maximum (1, 2, 3, 4)

$\theta \rightarrow$ minimum (1, 2)

Hence critical bolts are bolt No. 1, 2

$$F_{Rmax} = F_{R1} = F_{R2}$$

1. $P \rightarrow F_a$

2. $M = Pe$

$$= 10(100)$$

$$= 1000 \text{ KN-mm} \rightarrow F_m$$

$$F_a = \frac{P}{n} = \frac{10}{4} = 2.5 \text{ KN}$$

$$F_m = \frac{M \cdot r_1}{\sum r_i^2}$$

$$= \frac{1000(50)}{4 \times (50)^2}$$

$$= 5 \text{ KN}$$

$$r_1 = r_2 = r_3 = r_4 = \sqrt{(30)^2 + (40)^2}$$

$$= 50 \text{ mm}$$

Maximum resultant shear force $(F_R)_{max} = F_{R1} = F_{R2}$

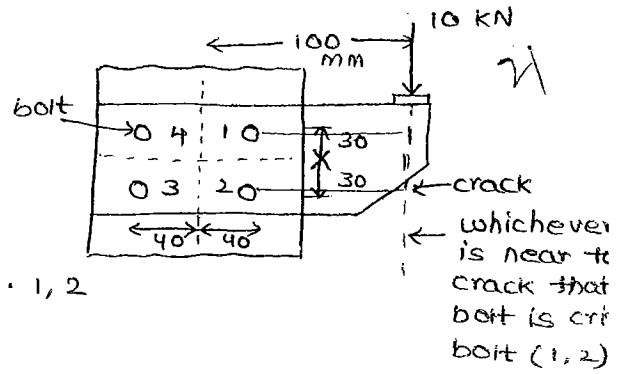
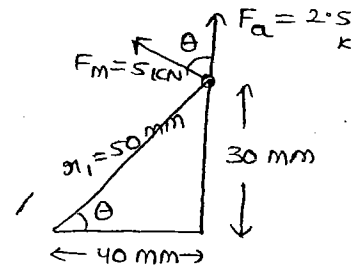
$$= \sqrt{F_a^2 + F_m^2 + 2F_a F_m \cos \theta}$$

$$\cos \theta = \frac{40}{50}$$

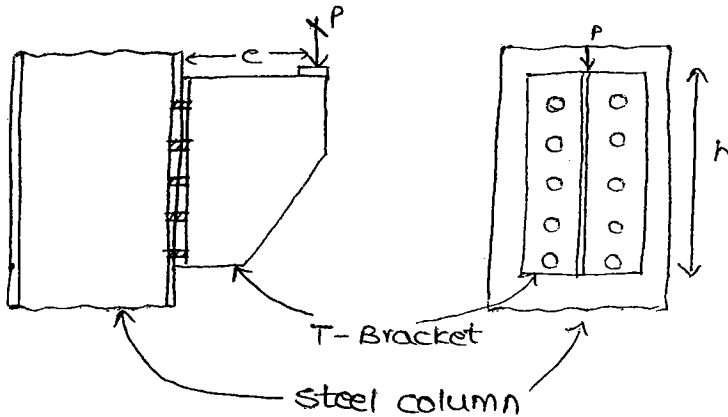
$$\cos \theta = 0.8$$

$$= \sqrt{(2.5)^2 + 5^2 + 2(2.5)(5) \cos(0.8)}$$

$$= 7.18 \text{ KN}$$



2. Bracket type Riveted connection - II (Elastic Analysis):-



$P =$ eccentric load

$e =$ eccentricity of the load

$h =$ depth of T-bracket

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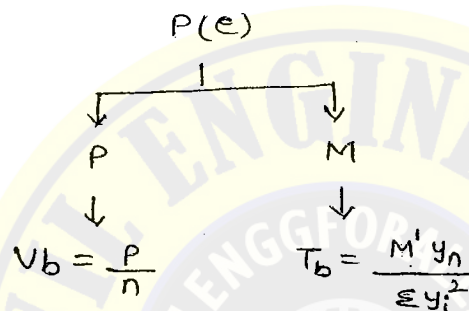
When load (or) moment is not lying in the plane of rivet (cold driven rivet group - No initial tension) group (or) Bolt group. Rivet or Bolt group is subjected to in

1. Direct concentric load (P)
2. Bending Moment ($M = Pe$).

→ Vertical shear force in any rivet (or) Bolt due to P is

$$V_b = \frac{P}{n} \quad (\text{shear})$$

n = no. of rivet (or) Bolt present in the connection.



→ Tensile force in i th bolt due to M' is the T_{bi}

$$T_{bi} = \frac{M' y_n}{\sum y_i^2}$$

Tensile force in extreme bolt (or) rivet due to M' is T_b

$$T_b = \frac{M' y_n}{\sum y_i^2} \quad (\text{Tension})$$

y_n = distance from C.G. of bolt (or) rivet group to extreme bolt (or) rivet.

(IES) → If axis of bending is $\frac{h}{2}$ from bottom

$$M' = \frac{M}{2} = \frac{Pe}{2}$$

If axis of bending is $\frac{h}{7}$ from bottom edge.

$$M' = \left[\frac{M}{1 + \frac{2h}{21} \frac{\sum y_i^2}{\sum y_i^2}} \right]$$

For safety (or) rivet group connection is checked when rivet is subjected to shear and Tension as per IS 800: 1984

$$\frac{\tau_{va, cal}}{\tau_{va}} + \frac{\sigma_{tf, cal}}{\sigma_{tf}} \leq 1.4$$

$\tau_{va, cal}$ = calculated shear stress in rivet

$$= \frac{V_b}{\frac{\pi(d)^2}{4}}, \quad V_b = \frac{P}{n}$$

$\sigma_{tf, cal}$ = calculated tensile stress in rivet

$$= \frac{T_b}{\frac{\pi(d)^2}{4}}, \quad T_b = \frac{M' y_n}{\sum y_i^2}$$

d = Gross (or) effective diameter of rivet

τ_{vf} = permissible shear stress in rivet

σ_{tf} = permissible tensile stress in rivet.

P.9 NO:- 29

3. Shear capacity, $V_{bc} = 20 \text{ KN}$

Tension capacity, $T_{bc} = 15 \text{ KN}$

1. $P \rightarrow V_b$

2. $M = Pe \rightarrow T_b$

$$= P(150)$$

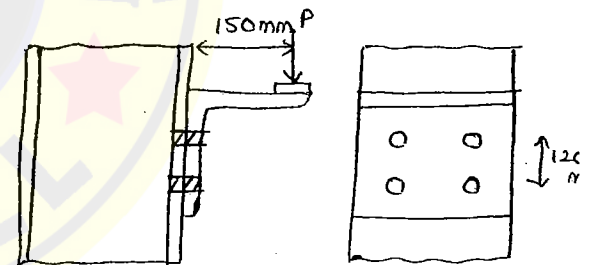
$$= 150P \text{ KN-mm}$$

$$V_b = \frac{P}{n} = \frac{P}{4} \text{ (shear)}$$

$$T_b = \frac{M' y_n}{\sum y_i^2}$$

$$= \frac{75P(60)}{2 \times 60^2}$$

$$= \frac{5P}{8} \text{ (Tension)}$$



$$\begin{aligned} M^I = M^{II} &= \frac{M}{2} \\ &= \frac{150P}{2} \\ &= 75P \end{aligned}$$

$$y_n = \frac{120}{\sqrt{2}} = 60 \text{ mm}$$

$$\frac{V_b}{\sigma_{bc}} + \frac{T_b}{T_{bc}} \leq 1.4$$

$$\left(\frac{P}{4} \cdot \frac{1}{20} \right) + \left(\frac{5P}{8} \cdot \frac{1}{15} \right) \leq 1.4$$

$$P_{max} = 25.85 \text{ kN}$$

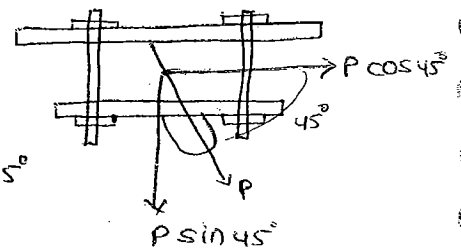
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4. Shear capacity, $V_{bc} = 30 \text{ kN}$

Tension capacity, $T_{bc} = 40 \text{ kN}$

Shear force in rivert due to $P \cos 45^\circ$

$$V_b = \frac{P \cos 45^\circ}{4}$$
$$= \frac{P}{4\sqrt{2}}$$



Tension due to force in rivert due to $P \sin 45^\circ$

$$T_b = \frac{P \sin 45^\circ}{4}$$
$$= \frac{P}{4\sqrt{2}}$$

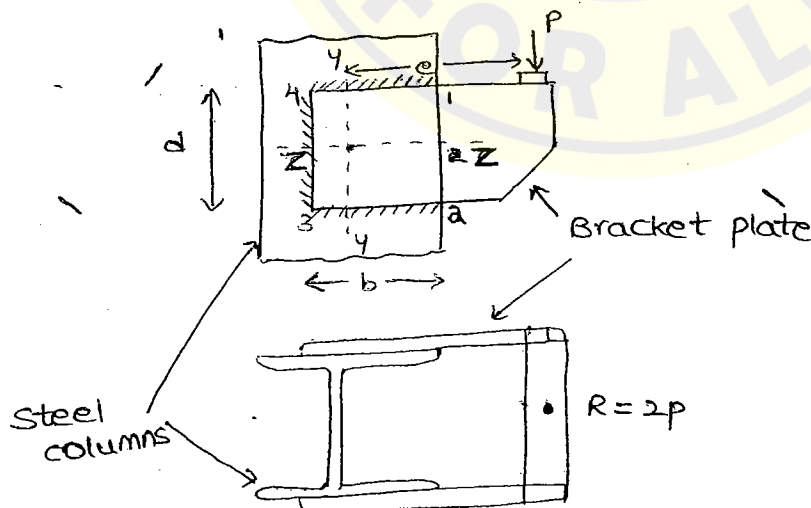
Interaction equation as per IS 800:1984

$$\frac{V_b}{V_{bc}} + \frac{T_b}{T_{bc}} \leq 1.4$$
$$\frac{P}{4\sqrt{2}} \cdot \frac{1}{30} + \frac{P}{4\sqrt{2}} \cdot \frac{1}{40} = 1.4$$

$$P_{\max} = 135.76 \text{ kN}$$

Eccentric Welded connections:-

1. Fillet weld Bracket Type connection - I (Elastic Analysis):



P = eccentric load
 e = eccentricity of the load (distance from C.G. of weld group to the applied load line)
 d = depth of bracket plate

When load or moment is lying in the plane of fillet weld group. Fillet weld group is subjected to

1. Direct concentric load (P)
2. Twisting moment ($M = Pe$)

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→ Vertical shear stress in weld due to P is q_1 , 23

$$q_1 = \frac{P}{\text{effective sectional area}}$$

$$= \frac{P}{L_w \cdot t_t}$$

$$= \frac{P}{(d+2b) \cdot t_t}$$

L_w = effective length of fillet weld = $d+2b$

t_t = effective throat thickness = ks

s = size of fillet weld

→ Shear stress in weld due to M is q_2

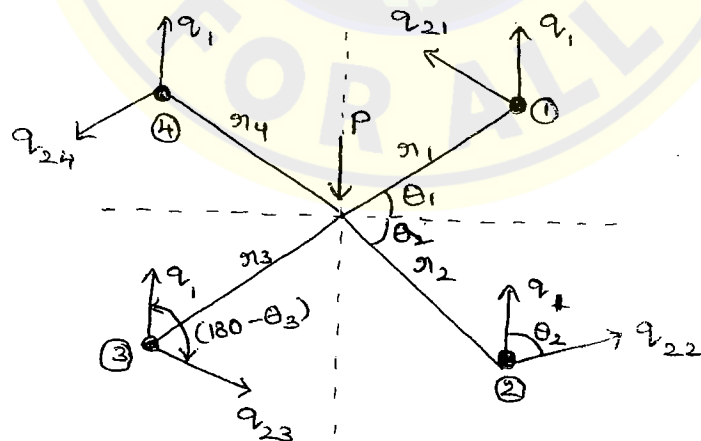
$$q_2 = \frac{M}{I_p} \times r$$

$$M = Pe$$

I_p = polar moment of inertia of fillet weld

$$= I_{zz} + I_{yy} \text{ of fillet weld}$$

r = radial distance from C.G. of weld to point on weld length.



Resultant shear stress between q_1 and q_2 is q_R

$$q_R = \sqrt{q_1^2 + q_2^2 + 2q_1q_2 \cos \theta}$$

Condition for maximum Q_R ($Q_{R \max}$):-

1. $r \rightarrow$ maximum
2. $\theta \rightarrow$ minimum

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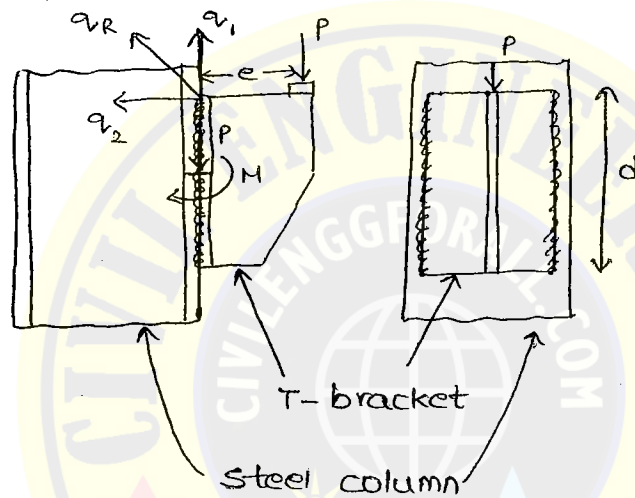
For safety of fillet weld group:-

$$(\tau_R)_{\max} \leq \tau_{vf} = \text{permissible shear stress in fillet weld}$$

$$\tau_{vf} = 108 \text{ Mpa}$$

Note:-

1. Heavily stressed point is ① at which the resultant shear stress is maximum which is farthest from C.G. of weld group (R is max.) and which may be closed to the applied load line.
2. Fillet weld Bracket type connection - II (Elastic Analysis):-



P = eccentric load
 e = eccentricity of the load
 d = depth of T-bracket

When load (or) moment is not lying in the plane of fillet weld group. Fillet weld group is subjected to

1. Direct concentric load (P)
2. Bending Moment ($M = Pe$)

→ vertical shear stress in weld due to P is τ_1

$$\tau_1 = \frac{P}{\text{effective sectional area}}$$

$$\begin{aligned} \tau_1 &= \frac{P}{L_w \cdot t_t} \\ &= \frac{P}{2d \cdot t_t} \end{aligned}$$

L_w = effective length of fillet weld = $2d$

t_t = Effective throat thickness = ks

s = size of fillet weld.

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→ stress is weld due to M is q_2

24

$$q_2 = \frac{M}{I} \cdot y$$

$$M = Pe$$

I = Moment of inertia of fillet weld about bending axis.

$$y = \frac{d}{2}$$

combined stress between q_1 and q_2 is q_R

$$q_R = \sqrt{q_1^2 + q_2^2 + 2q_1q_2 \cos \theta}$$

$$= \sqrt{q_1^2 + q_2^2 + 2q_1q_2 \cos 90^\circ}$$

$$q_R = \sqrt{q_1^2 + q_2^2}$$

Condition for max. q_R :-

$y \rightarrow$ maximum.

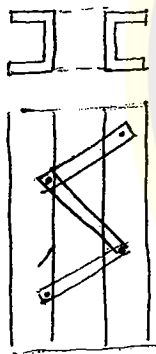
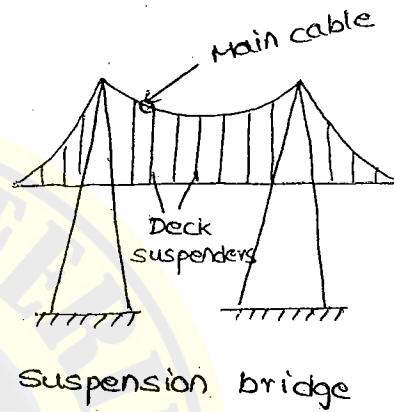
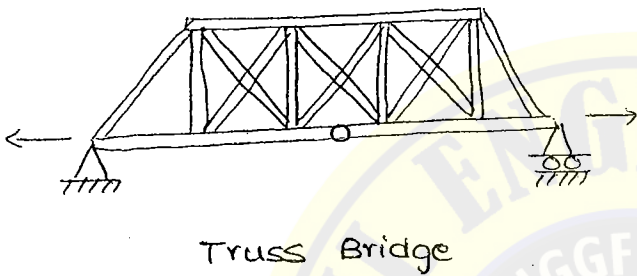
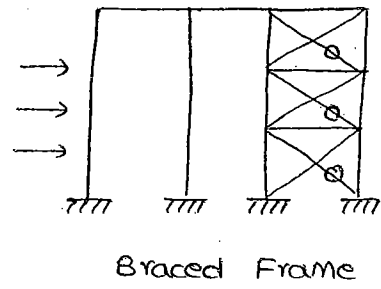
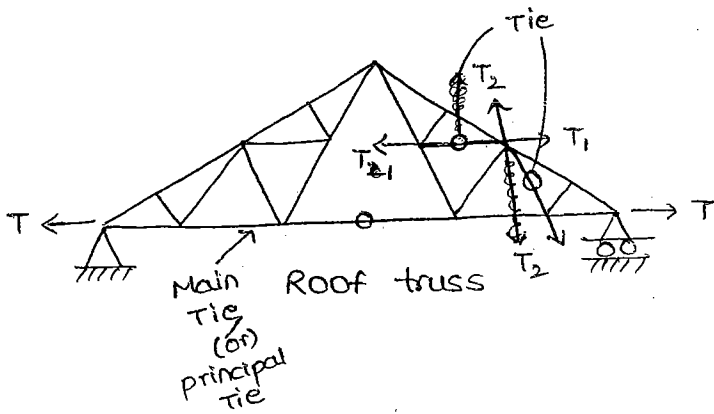
For safety of fillet weld :-

$$(q_R)_{\max} \leq \tau_{wf} = 108 \text{ Mpa}$$

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UNIT - 5

TENSION MEMBERS



Builtup column with lacing

Axial tensile strength of a member (or) a section (P_t):-

$$P_t = A_{net} \times \sigma_{at}$$

A_{net} = Net effective sectional area of a member

σ_{at} = permissible (or) Allowable axial tensile stress

$$= 0.6 f_y \text{ (IS 800 : 1984)}$$

f_y = yield stress for steel

$$\text{working stress} = \frac{P}{A_{net}} \leq \sigma_{at} = 0.6 f_y$$

$$P \leq A_{net} \times \sigma_{at} = P_t$$

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Net sectional area of a member (A_{net}):-

1. plate with chain riveting (or) bolting:

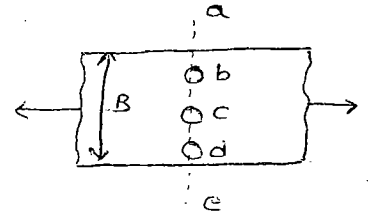
$$A_{net} = A_g - \text{Area of rivet or bolt holes}$$

A_{net} along section a-b-c-d-e

$$A_{net} = B \cdot t - n d t$$

$$= (B - n d) t$$

n = no. of rivet holes along section
 d = Gross dia of rivet



2. Plate with staggered (or) zig zag riveting (or) Bolting:-

A_{net} along section a-b-c-d-e

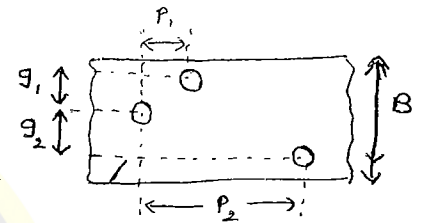
$$A_{net} = (B - n d) t + \frac{P_1^2 t}{4 g_1} + \frac{P_2^2 t}{4 g_2}$$

d = Gross (or) effective dia of rivet

B = width of the plate

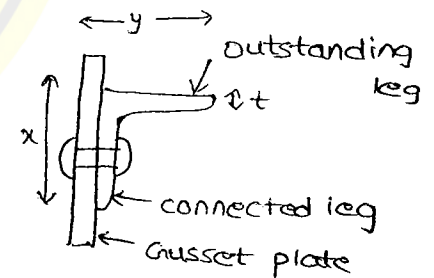
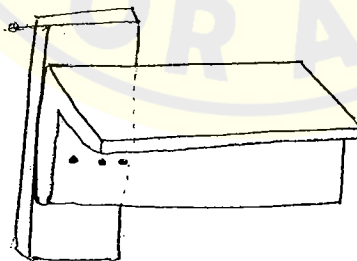
P_1 & P_2 = staggered pitches
 (Measured parallel to direction of a load)

g_1 & g_2 = gauge distances.



3. Single angle connected by one leg only:-

i. Riveted Angle:-



$$A_{net} = A_1 + A_2 k_1$$

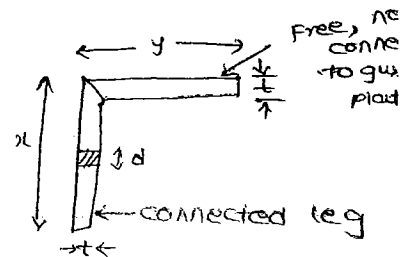
k_1 = Reduction factor

$$k_1 = \frac{3 A_1}{3 A_1 + A_2} = \frac{1}{1 + 0.33 A_2 / A_1}$$

A_1 = Net sectional area of connected leg

$$= x t - d t - \frac{1}{2} t \cdot t$$

$$A_1 = \left(x - d - \frac{t}{2} \right) t$$



$A_2 =$ Gross sectional area of outstanding leg

$$= yt - \frac{1}{2}tt$$

$$A_2 = (y - \frac{t}{2})t$$

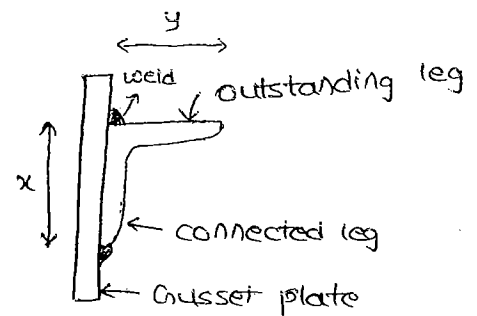
(ii) Welded Angle:-

$$A_1 = (x - \frac{t}{2})t$$

$$A_2 = (y - \frac{t}{2})t$$

$$A_{net} = A_1 + A_2 K_1$$

$$K_1 = \frac{3A_1}{3A_1 + A_2}$$



Note:-

The net area of welded angle tie member is higher than net sectional area of riveted angle or bolted angle. Hence, welded angle tie has more tensile strength than riveted (or) bolted angle tie.

Unequal angle tie its longer leg is connected to gusset plate provides more net area than equal angle section with same sectional area. Hence unequal angle section has more tensile strength than equal angle section.

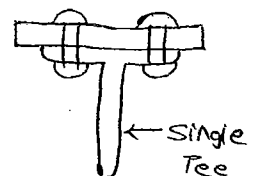
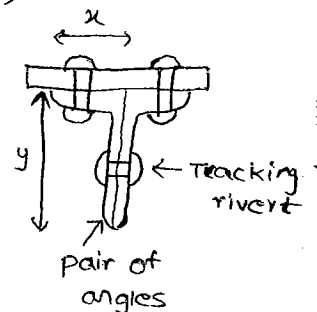
4. Pair of an angles [(or) single Tee's] placed back to back connected by one leg of each angle [(or) flange of Tee] to the same side of gusset plate (Tracking rivets).

$$A_{net} = A_1 + A_2 K_2$$

$$K_2 = \frac{5A_1}{5A_1 + A_2} = \frac{1}{1 + 0.2 \frac{A_2}{A_1}}$$

$$A_1 = \text{Net sectional area of connected leg} \\ = 2[x - d - \frac{t}{2}]t$$

$$A_2 = \text{Gross sectional area of outstanding leg} \\ = 2[y - \frac{t}{2}]t$$



If not tacking riveted

$$A_{net} = A_1 + A_2 k_1$$

$$k_1 = \frac{3A_1}{3A_1 + A_2} = \frac{1}{1 + 0.33 \frac{A_2}{A_1}}$$

$$A_1 = 2(x - d - \frac{t}{2})t$$

$$A_2 = 2(y - \frac{t}{2})t$$

5. Pair of angles (or) Double Tee's] placed back to back, connected one leg of each angle (or) Flange of Tee] to the either side of Gusset plate (Tacking Riveted).



$$A_{net} = A_g - \text{Area of rivet holes}$$

If not tacking riveted :-

$$A_{net} = A_1 + A_2 k_1$$

$$k_1 = \frac{3A_1}{3A_1 + A_2} = \frac{1}{1 + 0.33 \frac{A_2}{A_1}}$$

Design of axially loaded tension members :-

Slenderness ratio of tension member (λ) :-

$$\lambda = \frac{\text{unsupported length}}{\text{Min. radius of gyration}}$$

$$\lambda = \frac{L}{r_{min}}$$

** Limiting (or) Maximum slenderness ratio (λ_{limit}) (IS 800 stiffness crite

1. A tension member is subjected to load reversal (or) stress reversal, due to loads other than wind (or) earth forces

S.R due to loads other than W.L (or) E.L ≤ 180

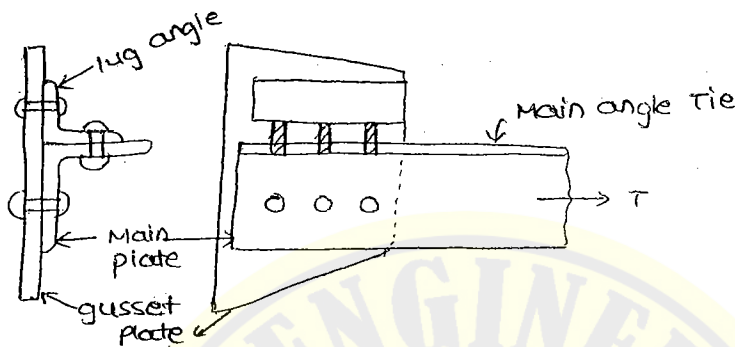
$$\frac{L}{r_{min}} \leq 180$$

2. A member is used as a tie member in roof truss or a bracing system subjected to load or stress reversal due to wind loads (or) earthquake loads.

$$S.R \text{ due to } W.L \text{ (or) } E.L \leq \underline{350}$$

3. For any other tension members (other than pretensioned member) $\leq \underline{400}$

Lug angle:-



Lug angle is a short length of an angle used at a joint location to join outstanding leg of angle type to the gusset plate (Outstanding flange of a channel type to the gusset plate) so that length of connection is reduced. By using Lug angle strength and efficiency of outstanding leg of an angle type and outstanding flange of a channel type may be improved.

$$A_{net} = A_g - \text{Area of rivet holes}$$

P.g No:- 37

$$1. A_{net} = A_1 + A_2 \cdot K_1$$

$$= a + bk$$

$$K_1 = \frac{3A_1}{3A_1 + A_2}$$

$$= \frac{3a}{3a + b}$$

$$= \frac{1}{1 + \frac{b}{3a}}$$

$$= \frac{1}{1 + 0.33 \frac{b}{a}}$$

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3. Given $A_1 = 50 \text{ mm}^2$ $A_2 = 100 \text{ mm}^2$

$$A_{net} = A_1 + A_2 k_1$$

$$k_1 = \frac{3(50)}{3(50) + 100}$$

$$= \frac{150}{250} = \frac{3}{5}$$

$$A_{net} = 50 + 100 \left(\frac{3}{5}\right)$$

$$= 50 + 60$$

$$= 110 \text{ mm}^2$$

4. Diameter of rivert = 20 mm, $d = 21.5 \text{ mm}$

Net effective area $A_{net} = ?$

$A_1 = (100 \times 6)$ - Net area of connected leg

$$A_1 = (x - d - \frac{t}{2})t$$

$$= (100 - 21.5 - \frac{6}{2})6$$

$$= 453 \text{ mm}^2$$

$A_2 =$ Gross area of Outstanding leg

$$= (y - \frac{t}{2})t$$

$$= (100 - \frac{6}{2})6$$

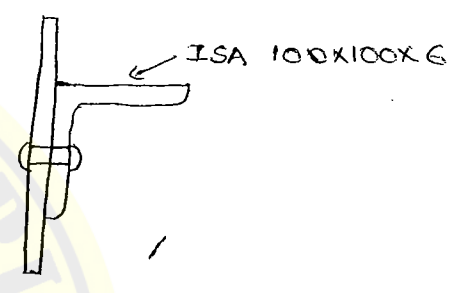
$$= 582 \text{ mm}^2$$

$$k_1 = \frac{3A_1}{3A_1 + A_2} = \frac{3 \times 453}{3 \times 453 + 582} = 0.70$$

$$A_{net} = A_1 + A_2 k_1$$

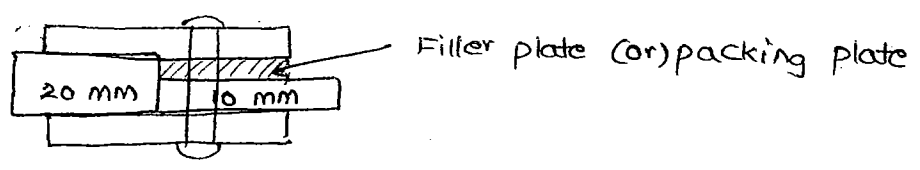
$$= 453 + 582(0.7)$$

$$= 860.4 \text{ mm}^2$$



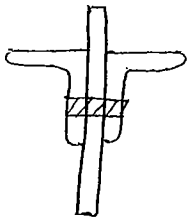
∴ If nominal dia is use then ans is wrong.

5.



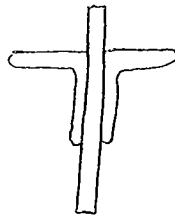
P.9 NO: 37

8. (a)



$$A_{net} = A_g - \text{area of rivet hole}$$

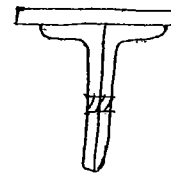
(b)



$$A_{net} = A_1 + A_2 k_1$$

$$k_1 = \frac{3A_1}{3A_1 + A_2}$$

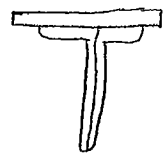
(c)



$$A_{net} = A_1 + A_2 k_2$$

$$k_2 = \frac{SA_1}{SA_1 + A_2}$$

(d)



$$A_{net} = A_1 + A_2 k_1$$

$$k_1 = \frac{3A_1}{3A_1 + A_2}$$

$$P_t = A_{net} \times \sigma_{at}$$

$$\therefore P_t(a) > P_t(c) > P_t(b) > P_t(d)$$

9. $f_y = 400 \text{ Mpa}$

$$\sigma_{at} = 0.6 f_y = 0.6 \times 400$$

$$= 240 \text{ Mpa}$$

10. Given $B = 300 \text{ mm}$ $t = 10 \text{ mm}$ $\phi = 16 \text{ mm}$ $d = 17.5 \text{ mm}$ (Gross dia)

$$A_{net} = (B - d)t$$

$$= (300 - 17.5) 10$$

$$= 2825 \text{ mm}^2$$

11. Given $B = 300 \text{ mm}$, $t = 10 \text{ mm}$, $\phi = 18 \text{ mm}$, $d = 19.5 \text{ mm}$

$$A_{net} = (B - d)t$$

$$= (300 - 19.5) \times 10$$

$$= 2805 \text{ mm}^2$$

12. $\sigma_{at} = 150 \text{ N/mm}^2$

$$P_t = A_{net} \times \sigma_{at}$$

$$A_{net} = A_1 + A_2 k_1$$

$$A_1 = (x - d - \frac{t}{2})t$$

$$= (100 - 21.5 - \frac{10}{2})10$$

$$= 735 \text{ mm}^2$$

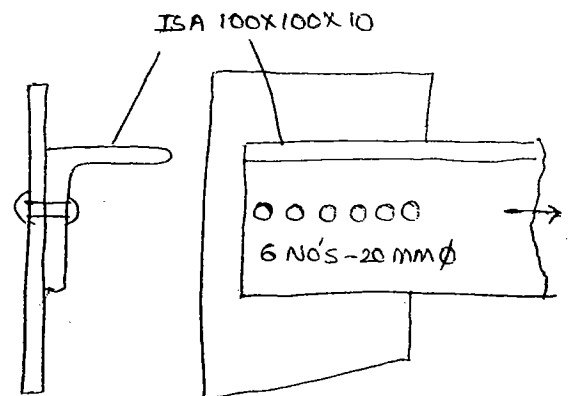


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$$A_2 = (y - \frac{t}{2})t$$

$$= (100 - \frac{10}{2})10$$

$$= 550 \text{ mm}^2$$

$$k_1 = \frac{3A_1}{3A_1 + A_2}$$

$$= \frac{3 \times 735}{3 \times 735 + 550}$$

$$= 0.698$$

$$A_{net} = 735 + 550 (0.698)$$

$$= 1399 \text{ mm}^2$$

$$P_t = A_{net} \times \sigma_{at}$$

$$= 1399 \times 150$$

$$= 209.85 \times 10^3 \text{ N}$$

$$= 209.85 \text{ kN}$$

13. $A_1 = 775 \text{ mm}^2$
 $A_2 = 950 \text{ mm}^2$

If not tack riveted

$$A_{net} = A_1 + A_2 k_1$$

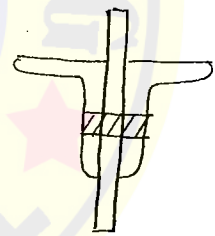
$$k_1 = \frac{3A_1}{3A_1 + A_2}$$

$$= \frac{3(775)}{3(775) + 950} = \frac{3(2 \times 775)}{3(2 \times 775) + (2 \times 950)}$$

$$= 0.71$$

$$A_{net} = (2 \times 775) + (2 \times 950)(0.71)$$

$$= 2899 \text{ mm}^2$$



Effective length of a compressive member (L_{effec}):-

$L_{effective}$ depends on

1. Type of end condition (such as fixed, pinned, roller, free etc)
2. NO. of rivets (or) bolts (or) length of weld used at a joint
3. No. of members are meeting at a joint.

L = unsupported length of column.

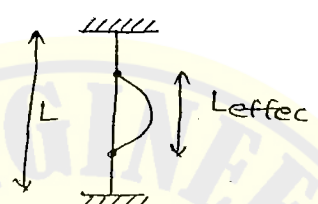
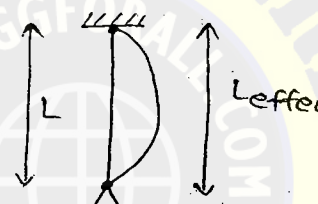
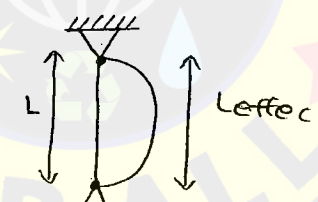

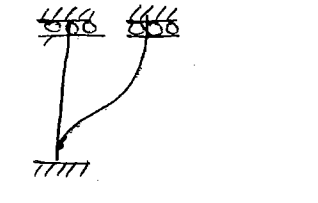
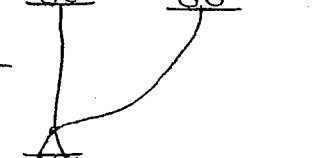
End condition	Diagrammatic Representation	Effective length
1. Fixed - Fixed		Actual Length $(0.5L)$ $0.5L$
2. Fixed - Hinged		$(0.707L)$ $0.80L$
3. pinned - pinned		$1.0L$
4. Fixed - Free		$2.0L$
5. Fixed - moment roller		$1.2L$
6. pinned - Moment roller		$2.0L$

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Design of axially loaded compression member:-

29

1. Allowable axial compressive stress in compression member is assumed.

For angle strut $\sigma_{ac} = 65 \text{ Mpa}$

For beam section columns, $\sigma_{ac} = 80 \text{ Mpa}$

For Heavy axial columns load, $\sigma_{ac} = 110 \text{ Mpa}$

2. Calculate approximate sectional area required to support axial compressive load, P

$$A_e \text{ required} = \frac{P}{\sigma_{ac} \text{ assumed}}$$

3. Select a trial section from steel table to match with sectional area required and calculate minimum radius of gyration of total section.
4. Effective length of trial section is to be calculated based on end conditions and slenderness ratio of trial section to be calculated which should be less than maximum slenderness ratio as per IS 800.

****** Maximum slenderness ratio (stiffness Requirement):-

1. A member is subjected to compressive loads due to loads dead and imposed loads

$$\lambda_{max} \leq 180$$

2. A member is subjected to compressive load due to wind (or) Earthquake loads

$$\lambda_{max} \leq 250$$

3. For compression flange of beam, $\lambda_{max} \leq 300$

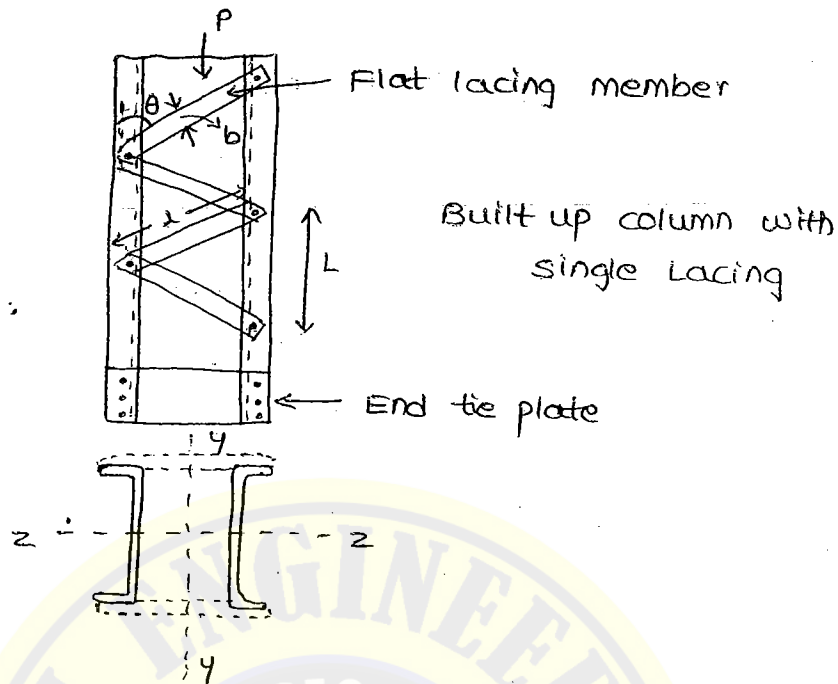
Built up sections (or) columns:-

Connecting systems used for built up columns are:

1. Lacing system (Generally preferred for eccentrically loaded columns)

2. Battens (Battens are normally used for axially loaded columns)

Lacing System:-



Flats, angles, channels and tube sections are used for section of lacing members.

θ = angle of inclination with longitudinal axis

L = spacing of lacing

l = length of lacing member

b = width of flat lacing member

t = thickness of flat lacing

P = Axial column load.

P.9 No. - 46

1. Slenderness ratio, $\lambda = \frac{\text{effective length}}{\text{min. radius of gyration}}$

$$L_{\text{eff}} = L$$

$$\begin{aligned} r_{\min} &= \sqrt{\frac{I_{\min}}{A}} = \sqrt{\frac{I}{A}} \\ &= \sqrt{\frac{\frac{\pi d^4}{64}}{\frac{\pi d^2}{4}}} \\ &= \frac{d}{4} \end{aligned}$$

$$\lambda = \frac{L_{\text{eff}}}{r_{\min}} = \frac{L}{\frac{d}{4}}$$

$$200 = \frac{4L}{d} \Rightarrow \frac{L}{d} = 50$$

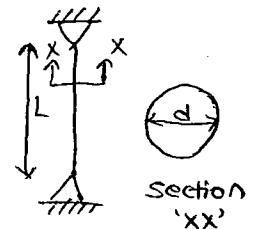


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6. Given ISA 100x100x10

$$A = 1903 \text{ mm}^2$$

$$I_{zz} = I_{yy} = 177 \times 10^4$$

$$\sigma_{zz} = \sqrt{\frac{I_{zz}}{A_e}} \quad \sigma_{yy} = \sqrt{\frac{I_{yy}}{A_e}}$$

$$I_{zz} = 2 [I_{C.G} + Ay^2]$$

$$= 2 [I_{zz} + Ay^2]$$

$$= 2 [I_{zz} + A(0)^2]$$

$$= 2 \cdot I_{zz}$$

$$\therefore I_{min} = I_{zz} = 2 \cdot I_{zz}$$

$$I_{yy} = 2 [I_{yy} + Az^2]$$

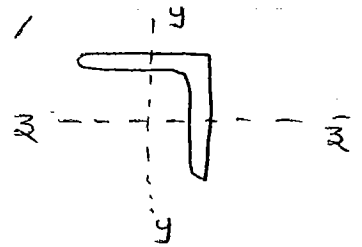
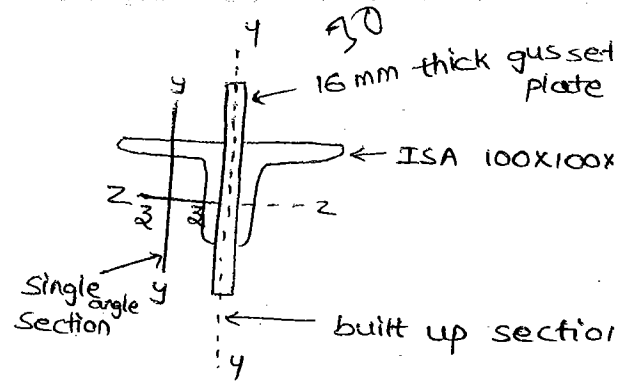
$$= 2 I_{yy} + 2 \cdot Az^2$$

$$\sigma_{min} = \sqrt{\frac{I_{min}}{A_e}}$$

$$= \sqrt{\frac{2 I_{zz}}{2A}}$$

$$= \sqrt{\frac{177 \times 10^4}{1903}}$$

$$= 30.6 \text{ mm}$$



$$A_e = \text{combined angle area}$$

$$= A + A$$

$$= 2A$$

8. Given ISA 150x150x10 mm A = 2921 mm²

$$I_{yy} = I_{zz} = 6335000 \text{ mm}^4$$

I_{min} = minimum of I_{zz} (or) I_{yy}

$$I_{zz} = 2 [I_{zz} + Ay^2]$$

$$= 2 I_{zz} + 2A(0)^2$$

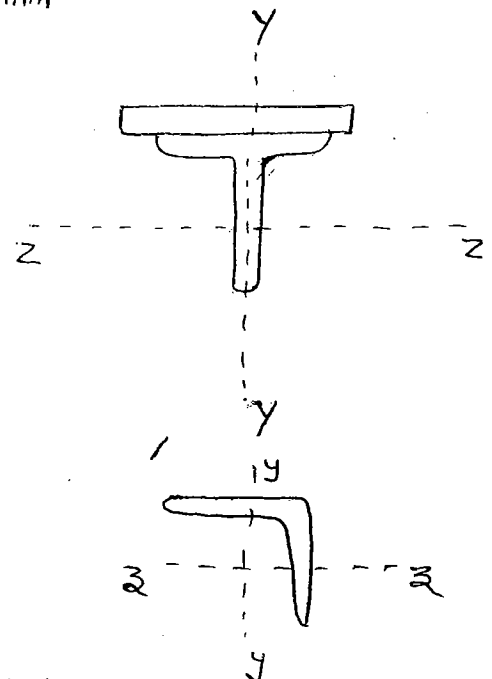
$$I_{zz} = 2 I_{zz}$$

$$I_{yy} = 2 [I_{yy} + Az^2]$$

$$= 2 I_{yy} + 2A(40.8)^2$$

$$\therefore I_{min} = 2 I_{zz}$$

$$\sigma_{min} = \sqrt{\frac{I_{min}}{A_e}} = \sqrt{\frac{2 I_{zz}}{2A}} = \sqrt{\frac{6335000}{2921}} = 40.6 \text{ mm}$$



1. M Lacing system:-

General specification:-

1. The radius of gyration normal to the plane of lacing not less than parallel to the plane of lacing.

$$r_{yy} \neq r_{zz}$$

To have a optimum condition (or) economical design

$$r_{yy} = r_{zz}$$

$$I_{yy} = I_{zz}$$

$$2 [I_{yy} + A z^2] = 2 [I_{zz} + A y^2]$$

$$2 I_{yy} + 2 A z^2 = 2 I_{zz} \quad \therefore y = 0$$

Note:-

For single lacing system the lacing on one plane should reflect mirror image to the lacing on other plane.

2. There should not be any variation in the lacing system (The spacing of lacing, angle of inclination of lacing should be maintained throughout the height of the column).

Design specification:-

1. $40^\circ \leq \theta \leq 70^\circ$

Optimum angle used in practise

$$\theta = 45^\circ \text{ to } 50^\circ$$

2. slenderness ratio of lacing member $(\lambda)_{\text{lacing}} \leq 145^\circ$

$$(\lambda)_{\text{lacing}} = \left(\frac{L_{\text{eff}}}{r_{\text{min}}} \right)_{\text{of lacing}} \leq 145^\circ$$

The above condition is required to eliminate local buckling failure of lacing member.

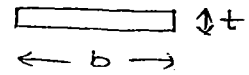
* Effective length of lacing member (l_{eff}):-

l_{eff}

- 1. For single lacing with one rivet (or) one bolt l
- 2. For Double lacing $0.7l$
- 3. For welded lacing $0.7l$

For Flat lacing ($b \times t$)

Min. radius of gyration



$$r_{min} = \sqrt{\frac{I_{min}}{A}}$$

$$= \sqrt{\frac{bt^3/12}{bt}}$$

$$= \frac{t}{\sqrt{12}}$$

Minimum width of flat lacing

$$(b)_{min} = 3 \times \text{Nominal diameter of rivet}$$

(or)

$$\text{shank diameter of rivet/bolt}$$

Nominal dia of rivet (mm)	16	18	20	22
Min. width of flat lacing b_{min} (mm)	50	55	60	65

Minimum thickness of flat lacing

$$t_{min} \neq \frac{l}{40} \text{ for single lacing}$$

$$\neq \frac{l}{60} \text{ for Double lacing}$$

$$\lambda_{lacing} = \frac{l_{effec}}{r_{min}}$$

$$= \frac{l_{effec}}{t/\sqrt{12}} \leq 145$$

$$= \frac{\sqrt{12} \cdot l_{effec}}{t} \leq 145 \text{ for flat lacing}$$

4. $\frac{L}{r_{\phi}^{\min}} \leq 50$
 $\leq 0.7 \times \text{slenderness ratio of whole builtup column}$
 whichever is less.

5. r_c^{\min} = minimum radius of gyration of individual column component.

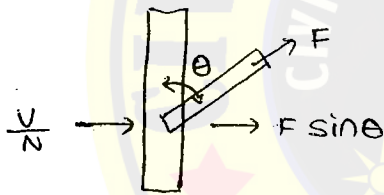
The above condition is required to eliminate local buckling failure of individual column component between lacings.

The lacing member should be design for transfer shear of 2.5% axial column load.

5. Transfer shear force (V) = 2.5% Axial column load (P)

$$V = \frac{2.5P}{100}$$

6. Axial force in lacing member is $F = \frac{V}{N \sin \theta}$



$$\sum F_x = 0$$

$$\frac{V}{N} + F \sin \theta = 0$$

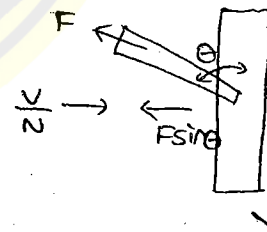
$$F = \frac{-V}{N \sin \theta}$$



$$\sum F_x = 0$$

$$\frac{V}{N} - F \sin \theta = 0$$

$$F = \frac{V}{N \sin \theta}$$



7. Axial force in lacing member (F)

$$F = \frac{V}{2 \sin \theta} \quad (N=2, \text{ for single lacing})$$

$$F = \frac{V}{4 \sin \theta} \quad (N=4, \text{ for double lacing})$$

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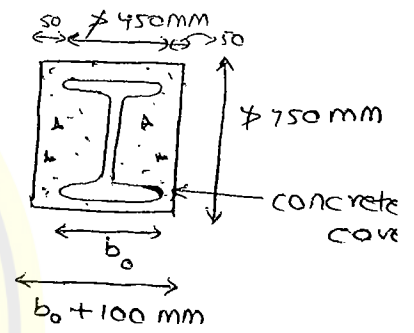
Batten system:-

General specification:-

1. The effective length of batten & column should be increased by 10% as per IS specification.
2. The batten should divide the entire column into minimum of three components. (A min. four no. of battens may required for batten column out of this 4 battens two are intermediate battens and other two are end battens).

Encased column (or) cased column:-

1. Minimum width of encased column
 $= b_0 + 100 \text{ mm}$
 $b_0 = \text{width of steel column}$



2. Maximum dimensions of encased column $\nless 450 \text{ mm} \times 750 \text{ mm}$
3. Minimum concrete cover = 50 mm
4. Minimum radius of gyration of encased column

$$r_{\min} = 0.2(b_0 + 100)$$

P.9 NO:- 46

4. Given $N=2$, $\theta = 45^\circ$ $P = 1000 \text{ KN}$

Transverse shear, $V = \frac{2.5P}{100}$

$$= \frac{2.5(1000)}{100}$$

$$= 25 \text{ KN}$$

5. Axial force in leading lacing

$$F = \frac{V}{N \sin \theta}$$

$$= \frac{25}{2 \sin 45^\circ}$$

$$= \frac{25}{2 \sin 45^\circ}$$

$$= 17.68 \text{ KN}$$

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P.9 NO:- 47

1. Given $\sigma_{min} = 25 \text{ mm}$

Maximum length of compression member L_{max}

$$\frac{L_{effec}}{\sigma_{min}} \leq \lambda_{limit}$$

$$\frac{L_{effec}}{25} \leq 180$$

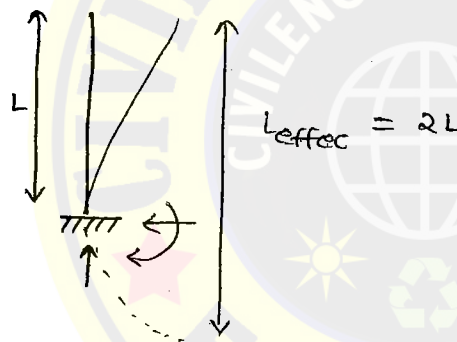
$$\begin{aligned} L_{effec} &\leq 180 \times 25 \\ &\leq 4500 \text{ mm} \\ &\leq 4.5 \text{ m} \end{aligned}$$

$$\begin{aligned} 2. \quad t_{min} &\geq \frac{L}{40} \\ &\geq \frac{L}{60} = \frac{500}{60} = 8.33 \text{ mm} \\ &= 10 \text{ mm} \end{aligned}$$

$$\begin{aligned} 13. \quad P &= 2.5 \times 10^6 \text{ N} \\ &= 2.5 \times 10^3 \text{ kN} \end{aligned}$$

$$\begin{aligned} V &= \frac{2.5P}{100} \\ &= \frac{2.5(2.5 \times 10^3)}{100} \\ V &= 62.5 \text{ kN} \end{aligned}$$

6.



8.

$$\Delta_{max} \leq 2 \times \frac{h}{325}$$

$$\frac{\Delta_{max}}{h} \leq \frac{2}{325}$$

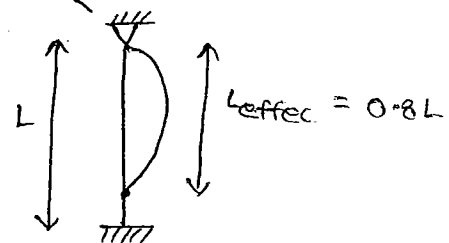


9.

$$\begin{aligned} L_{effec} &= 0.8L \\ &= 0.8 \times 5 \\ &= 4 \text{ m} \end{aligned}$$

$$\sigma_{min} = 100$$

$$\begin{aligned} \lambda &= \frac{L_{eff}}{\sigma_{min}} \\ &= \frac{4000}{100} \\ &= 40 \end{aligned}$$



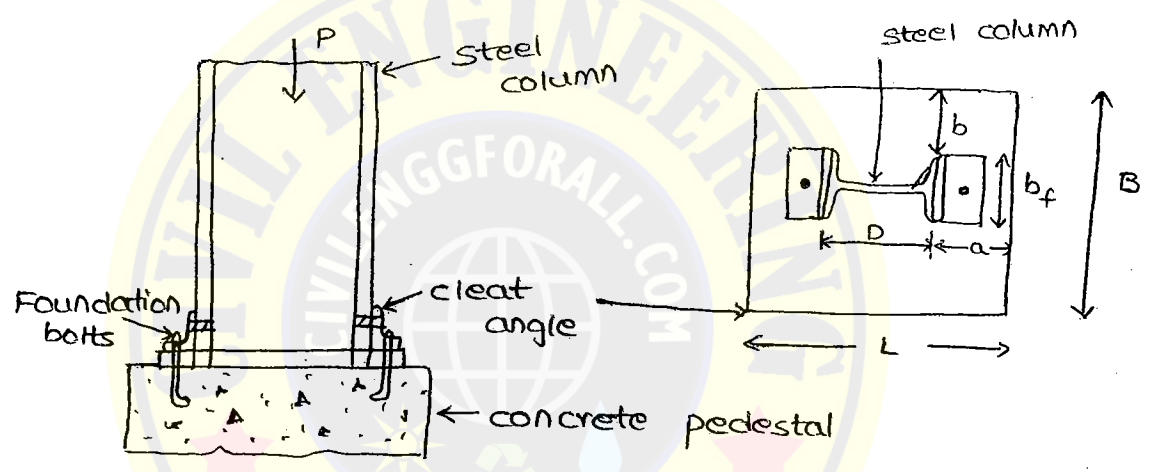
UNIT - 7

DESIGN OF COLUMN BASES & COLUMN SPLICES

Types of column base:-

- 1. Slab base (Generally provided under axially loaded columns)
- 2. Gussseted base (Normally preferred under heavy axially loaded columns and column is subjected to axial load with moment).
- 3. Grillage foundation (When heavy loaded columns is support by black cotton soils).

Design of slab base:-



- P = axial load in column
- L = Length of slab base
- B = width of slab base
- t_s = thickness of slab base
- a = Bigger projection of slab base beyond the steel column
- b = lesser projection of slab base beyond the steel column

→ Area of slab base, $A = \frac{\text{Axial of column load}}{\text{permissible bearing stress in concrete}}$

$$A = \frac{P}{\sigma_c}$$

→ For square slab base, $L = B = \sqrt{A}$
 side of slab base, $L = B = \sqrt{A}$

→ For rectangular slab base:-

$$A = L \times B$$

$$\therefore a = b$$

$$A = (D + 2a)(b_f + 2b)$$

→ To have a optimum thickness of slab base, the condition must be $a = b$

$$A = (D + 2a)(b_f + 2a)$$

→ Upward pressure from concrete pedestal (w)

$$w = \frac{\text{Axial column load}}{\text{provided area of slab base}}$$

$$w = \frac{P}{A}$$

→ Thickness of slab base (t_s)

$$t_s = \sqrt{\frac{3w}{\sigma_{bs}} \left(a^2 - \frac{b^2}{4} \right)}$$

σ_{bs} = permissible bending stress in slab base

$$\sigma_{bs} = 185 \text{ Mpa (for all steels)}$$

P.g No:- 52

1. Axial column load, $P = 2 \text{ MN}$
 $= 2 \times 10^6 \text{ N}$

$$\sigma_{bs} = 200 \text{ Mpa}$$

$$t_s = \sqrt{\frac{3w}{\sigma_{bs}} \left(a^2 - \frac{b^2}{4} \right)}$$

$$a = b = \frac{500 - 300}{2} = 100$$

$$w = \frac{P}{A} = \frac{2 \times 10^6}{500 \times 500} = 8 \text{ N/mm}^2$$

$$t_s = \sqrt{\frac{3 \times 8}{200} \left(100^2 - \frac{100^2}{4} \right)}$$

$$= 30 \text{ mm}$$

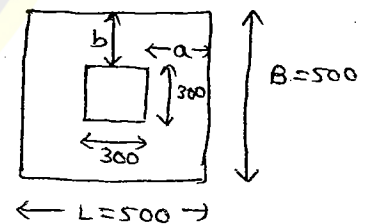


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P.9 NO:- 51

3. Given $P = 2000 \text{ kN}$

$$\sigma_{bc} = 185 \text{ Mpa}$$

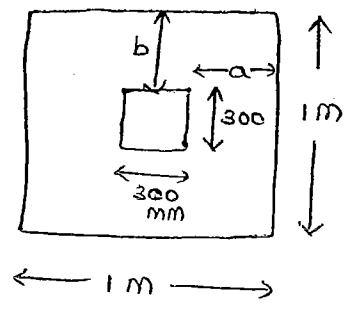
$$w = \frac{P}{A} = \frac{2000 \times 10^3}{1000 \times 1000} = 2 \text{ N/mm}^2$$

$$a = b = \frac{1000 - 300}{2} = 350 \text{ mm}$$

$$t_s = \sqrt{\frac{3w}{\sigma_{bs}}(a^2 - \frac{b^2}{4})}$$

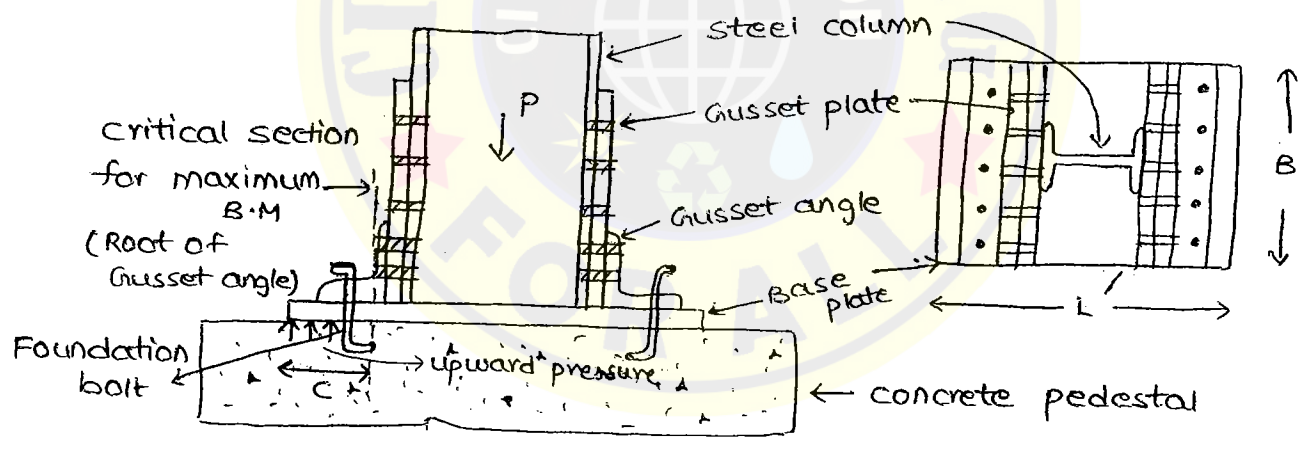
$$= \sqrt{\frac{3 \times 2}{185} (350^2 - \frac{350^2}{4})}$$

$$= 55 \text{ mm}$$



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Design of Gusseted Base:-



L = Length of Base plate

B = width of Base plate

t = thickness of base plate

c = cantilever projection beyond the root of Gusset angle

Area of Base plate, $A = \frac{\text{Axial column load}}{\text{permissible bearing stress in concrete}}$

$$A = \frac{P}{\sigma_c}$$

→ Length of Base plate, $L = \text{Depth of steel column} + 2 \times \text{thickness of gusset plate} + 2 \times \text{leg width of Gusset angle} + 2 \times \text{min. overhand (for riveted/bolted connection)}$

$L = \text{Depth of steel column} + 2 \times \text{thickness of Gusset plate} + 2 \times \text{Min. overhand (for welded connection)}$

→ Width of the base plate, $B = \frac{\text{Area of Base plate}}{\text{Length of Base plate}}$

→ Upward pressure from concrete pedestal, $w = \frac{P}{A_1}$

$$M = w \cdot c \cdot \frac{c}{2}$$

$$M \leq M_d$$

$$\leq \sigma_{bs} \cdot z$$

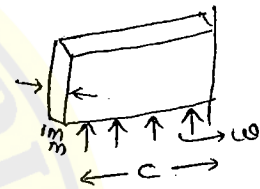
$$\frac{w c^2}{2} = \sigma_{bs} \cdot \frac{t^2}{26}$$

$$t = c \sqrt{\frac{3w}{\sigma_{bs}}}$$

$$z = \frac{I}{y}$$

$$= \frac{1 \cdot t^3}{12 \cdot \frac{t}{2}}$$

$$z = \frac{t^2}{6}$$



→ ∴ Thickness required,

$$t = c \sqrt{\frac{3w}{\sigma_{bs}}}$$

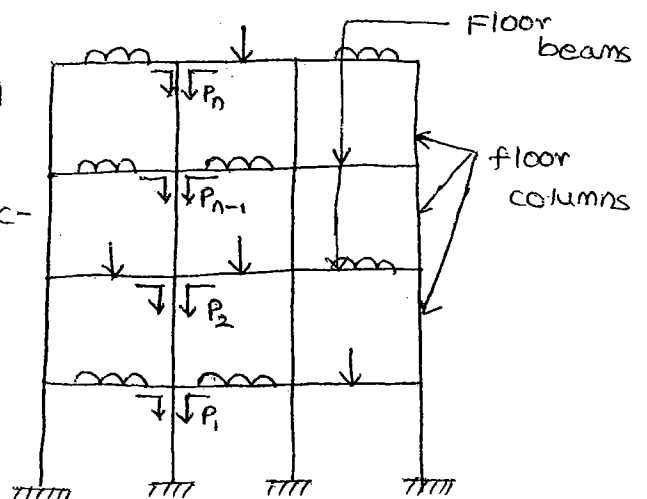
→ Thickness of base plate, $t_b = t - \text{thickness of Gusset angle}$
(for riveted/bolted connection)

$t_b = t$ (for welded connection)

Column splice :-

Let $P_1, P_2, \dots, P_{n-1}, P_n$ axial column loads in various floor columns from bottom floor respectively.

$$P_1 > P_2 > \dots > P_{n-1} > P_n$$



Steel frame

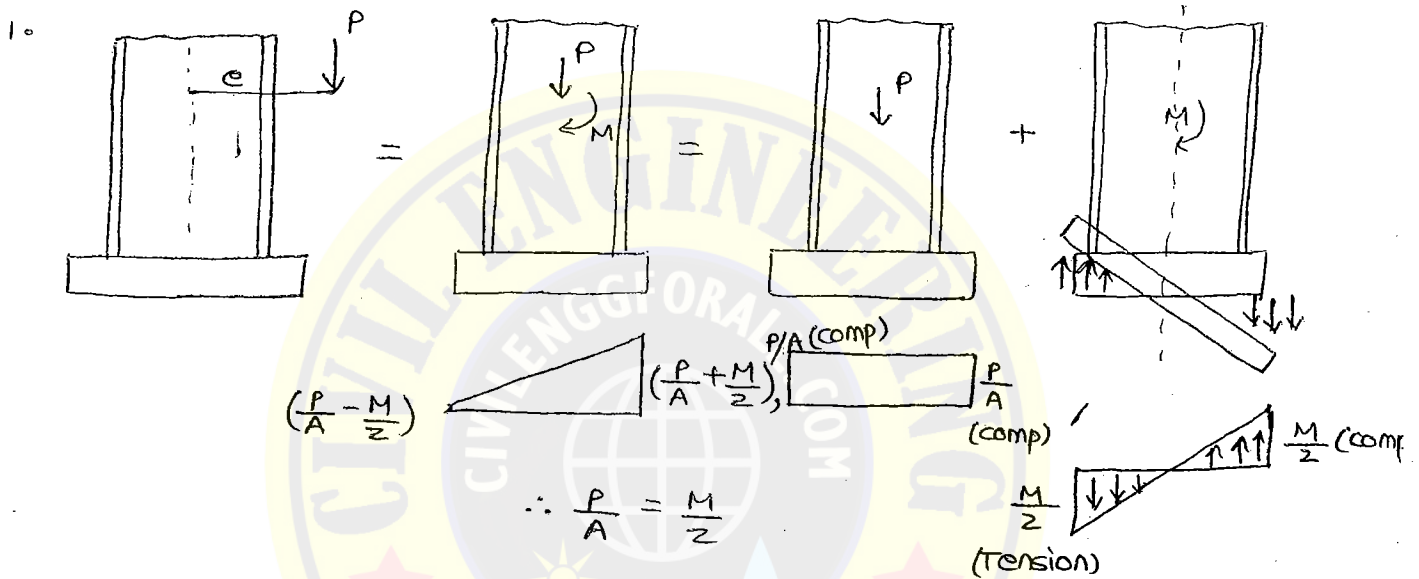
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Note:-

If ends of columns are machined or milled (or) smooth end theoretically no connecting system is required. But in practice connection to be designed for 50% column load and full moment if any.

If ends of columns are not machined complete bearing connecting system to be designed for full column load and full moment if any.

P.g NO:-51



P.g NO:-52

4. Given $\sigma_{bs} = 185 \text{ Mpa}$

$$w = \frac{P}{A}$$

In all four diagrams Area is same then w is const

$$t = c \sqrt{\frac{3w}{\sigma_{bs}}}$$

$$t \propto c$$

If 'c' is less then base plate thickness is minimum

(a) $\rightarrow c = \frac{600 - 140}{2} = 230$

(b) $\rightarrow c = \frac{600 - 400}{2} = 100$

(c) $\rightarrow c = \frac{500 - 140}{2} = 80$

(d) $\rightarrow c = \frac{720 - 400}{2} = 160$

$$\therefore \downarrow t \propto \downarrow c$$

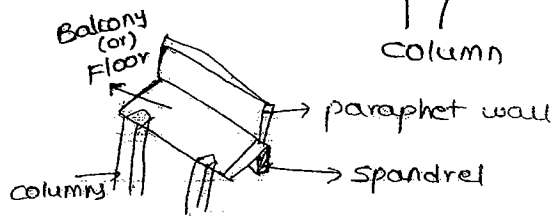
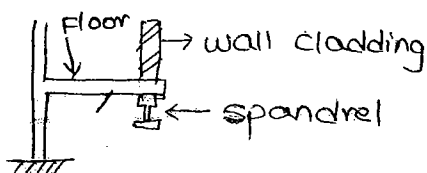
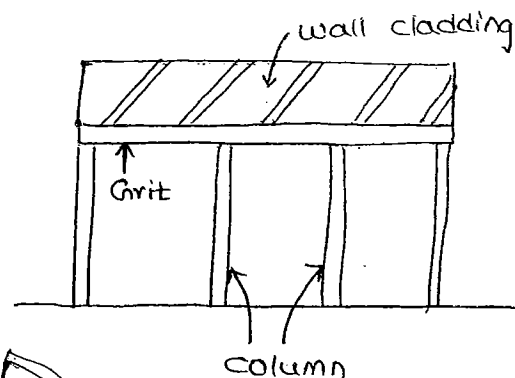
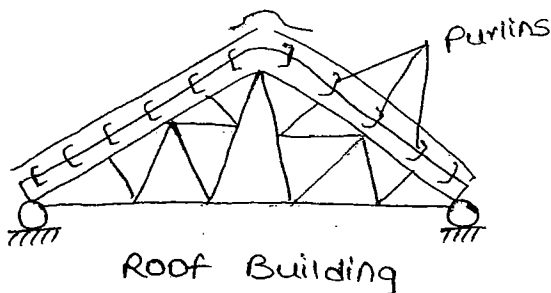
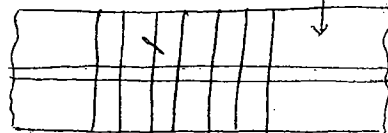
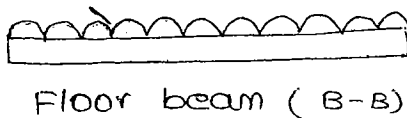
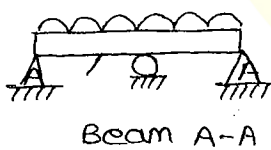
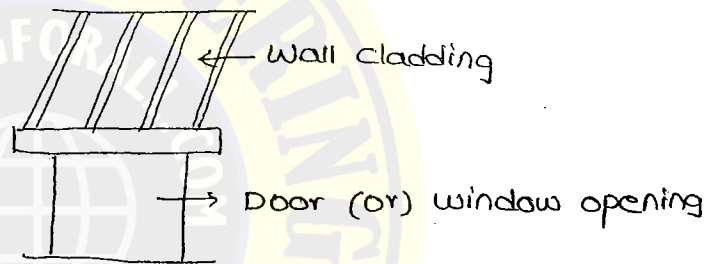
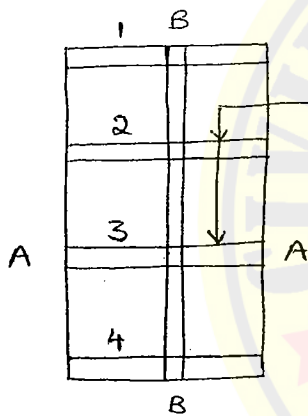
UNIT - 8

DESIGN OF BEAMS

Different terms used to designate a beam

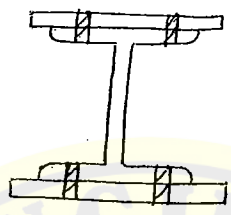
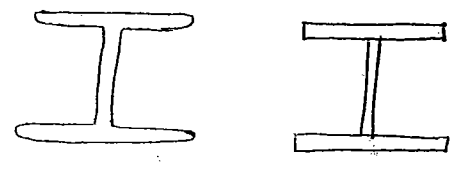
1. Joist
2. Floor Beam
3. Girder (or) Floor beam
4. Lintel
5. purlin
6. Spandrel
7. Stringer
8. Grit
9. Header

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Types of section :-

1. Rolled I-section (Or) I-beam (Or) I-section is built with plates.
2. Rolled channel section (for light transverse loads)
EX:- purlin, lintel, Girt etc.
3. Built up beams (Or) plated beams (Or) compounded beams



Design criteria of Beam:-

1. Design for Bending moment (M)
2. Design for shear force (V)
3. Check for deflections ($\Delta_{cal} \leq \Delta_{max}$)
4. Check for secondary affects like under concentrated loads and Reactions.
 - i. Web buckling
 - ii. Web crippling

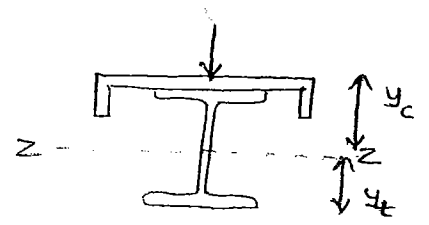
Lateral stability of beams :-

Based on lateral stability beams are two types.

1. Supported Laterally supported beams (Or) Laterally Restrained beams (where compression flange of a beam is not affected by lateral buckling (Or) lateral torsional buckling)
2. Laterally unrestrained beams (Compression flange of a beam is affected to either torsional (Or) lateral (Or) lateral torsional buckling)

Design of Bending Moment (M) :-

The actual (Or) calculated Bending compressive stress ($\sigma_{bc\ cal}$) (Or) calculated Bending tensile stress ($\sigma_{bt\ cal}$)



$$\sigma_{bc \text{ cal}} = \frac{M}{I} \cdot y_c$$

$$\sigma_{bt \text{ cal}} = \frac{M}{I} \cdot y_t$$

M = Maximum B.M

I = Moment of Inertia about Bending axis

y_c = Distance from axis of bending extreme compressive fibre

y_t = Distance from axis of bending extreme tension fibre.

Permissible bending stresses:-

1. For laterally restrained beams

a. permissible bending tensile stress

$$\sigma_{bt} = 0.66 f_y$$

b. permissible bending compressive stress

$$\sigma_{bc} = 0.66 f_y$$

2. For laterally unrestrained beams

$$\sigma_{bt} = 0.66 f_y$$

σ_{bc} may be calculated by using Rankine merchant formula based on (L/r_{yy})

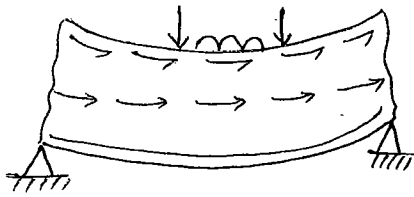
$$\sigma_{bc} = \frac{0.66 f_y f_{cb}}{[(f_y)^n + (f_{cb})^n]^{1/n}} \leq 0.66 f_y$$

$$f_{cb} = \text{Elastic critical stress in bending} = \frac{\pi^2 E}{(L/r_{yy})^2}$$

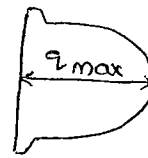
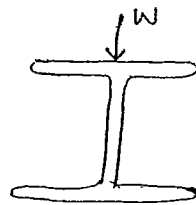
Note:-

The above permissible bending stresses are increased by $33 \frac{1}{3} \%$ when wind loads are used for design calculations.

Design for shear force:-



Longitudinal (or) Horizontal shear distribution.



Actual shear stress distribution



Avg. shear stress (assumed)

Longitudinal shear stress at any level,

$$q_v = \frac{VA\bar{y}}{Ib}$$

V = maximum SF

$A\bar{y}$ = First moment of area

b = width of section, where intensity of shear stress is calculated.

I = moment of inertia of section about bending axis

Calculated avg. shear stress $\tau_{v, cal} = \frac{V}{Dt_w}$

D = Overall depth of beam

t_w = thickness of a web.

Permissible shear stress:-

Maximum permissible shear stress, $\tau_{v, max} = 0.45 f_y$

permissible avg. shear stress, $= 0.4 f_y$

Note:-

The above permissible stresses are increased by $33\frac{1}{3}\%$ if wind loads are considered in design calculation.

Check for deflections:-

$$\Delta_{cal} \leq \Delta_{max} \text{ (or) } \Delta_{limit}$$

Δ_{cal} = calculated deflection due to external transverse loads.

Δ_{max} (or) Δ_{limit} = Maximum (or) limiting deflections as per

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Maximum deflection (Δ_{max}) (IS 800 : 1984) :-

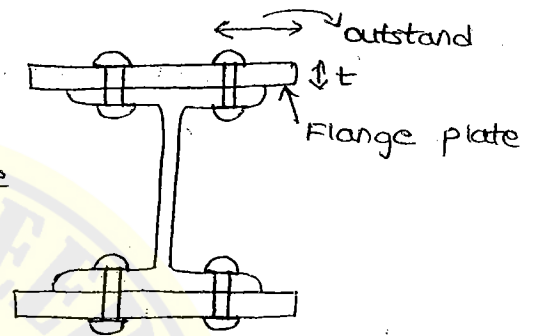
1. For simple span = $\frac{\text{Span}}{325}$ (or) $\frac{d}{325}$

2. For cantilever beam = $\frac{2 \times \text{Span}}{325}$ (or) $\frac{2d}{325}$ (or) $2 \times (A_{cal})_{S.S. \text{ Beam}}$

3. For purlin = $\frac{\text{Span}}{200}$ (or) $\frac{d}{200}$

Builtup beams (or) plated (or) compounded Beams :-

Builtup beams are used when beam member is subjected to heavy transverse load with large spans and also used where depth of beam section is restricted to have more clear height.



Builtup beam

M = Maximum Bending moment

Z_{gr} = required section modulus

$$Z_{gr} = \frac{M}{\sigma_{bt}}$$

σ_{bt} = permissible bending stress

Z_a = available section modulus of rolled I-section.

If $Z_{gr} > Z_a$ (plated or builtup beam is to be provided).

Area of each flange plate, $A_p = \frac{Z_{gr} - Z_a}{d}$

d = Overall depth of a beam.

$$I_{B.S} = I_{R.B} + I_{P.L}$$

$$\frac{I_R}{d/2} = \frac{I_a}{d/2} + \frac{A_p \cdot (\frac{d}{2}) \times 2}{\frac{d}{2}}$$

$$Z_{gr} A_p = Z_a + 2 A_p \cdot d/2$$

$$A_p = \frac{Z_{gr} - Z_a}{d}$$

$$t = \frac{A_p}{200}$$

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Outstand :-

1. Outstand $\nless 16T$ (or) $\frac{256T}{\sqrt{f_y}}$ for compression flange
(whichever is less)
2. Outstand $\nless 20T$ for tension flange.

Note :-

The distance from centre of rivet hole (or) bolt hole to the extreme edge of flange plate (outstand) to be limited to avoid local buckling failure of flange plate

T = thickness of outstand.

P.9 NO:- 58

1. Given $f_y = 300$ Mpa

$$\begin{aligned}\tau_{va \text{ max}} &= 0.45 f_y \\ &= 0.45 (300) \\ &= 135 \text{ Mpa}\end{aligned}$$

2. permissible bearing stress = $0.75 f_y$
 $= 0.75 \times 250$
 $= 187.5 \text{ Mpa}$

6. Given $f_y = 250$ Mpa

$$\begin{aligned}\tau_{va \text{ max}} &= 0.45 f_y \\ &= 0.45 (250) \\ &= 112.5 \text{ Mpa}\end{aligned}$$

7. For mild steel, $f_y = 250$ Mpa

$$\begin{aligned}\sigma_{bt} &= 0.66 f_y \\ &= 0.66 (250) \\ &= 165 \text{ Mpa}\end{aligned}$$

8. Mild steel, $f_y = 250$ Mpa

$$\begin{aligned}\tau_{va} &= 0.4 f_y \\ &= 0.4 (250) \\ &= 100 \text{ Mpa}\end{aligned}$$

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P.9 NO:- 57

1. For ISLB 300

$$d = 300 \text{ mm} \quad t_w = 5.4 \text{ mm}$$

$$I_{xx} = 1696.6 \text{ cm}^4 \\ = 1696.6 \times 10^4 \text{ mm}^4$$

Flange 100 mm x 7.3 mm

$$\text{Max. BM} = \frac{wL^2}{2} = \frac{20(3)^2}{2} \\ = 90 \text{ kN-m}$$

$$= 90 \times 10^6 \text{ N-mm}$$

$$\sigma_{bc \text{ cal}} = \sigma_{bt \text{ cal}} = \frac{M \cdot y}{I_{xx}} \\ = \frac{90 \times 10^6}{1696.6 \times 10^4} \times \left(\frac{300}{2}\right) \\ = 530.45 \text{ N/mm}^2$$

$$\text{Max. SF} = V = wL \\ = 20(3) \\ = 60 \times 10^3 \text{ N}$$

$$\tau_{va \text{ cal}} = \frac{V}{d t_w} \\ = \frac{60 \times 10^3}{300 \times 5.4} \\ = 55.55 \text{ N/mm}^2$$

2. Given $D = 56 \times 10^3 \text{ N}$

ISWB 350, $d = 350 \text{ mm}$

$$t_w = 8 \text{ mm}$$

$$\tau_{va \text{ cal}} = \frac{V}{d t_w} \\ = \frac{56 \times 10^3}{350 \times 8} \\ = 20 \text{ N/mm}^2$$

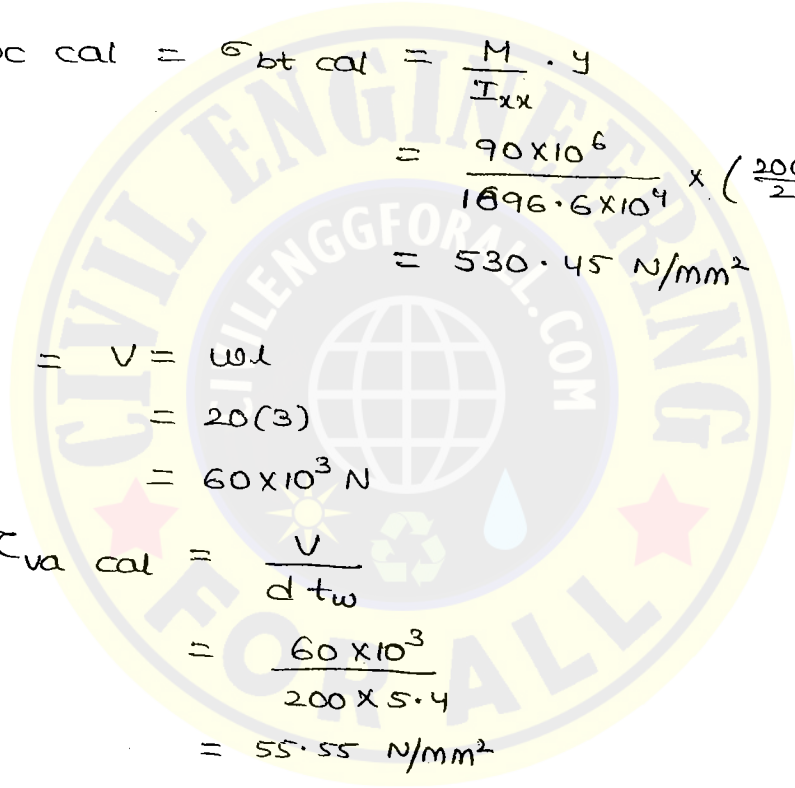
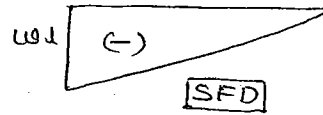
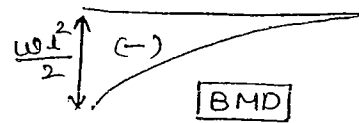
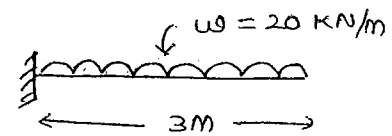


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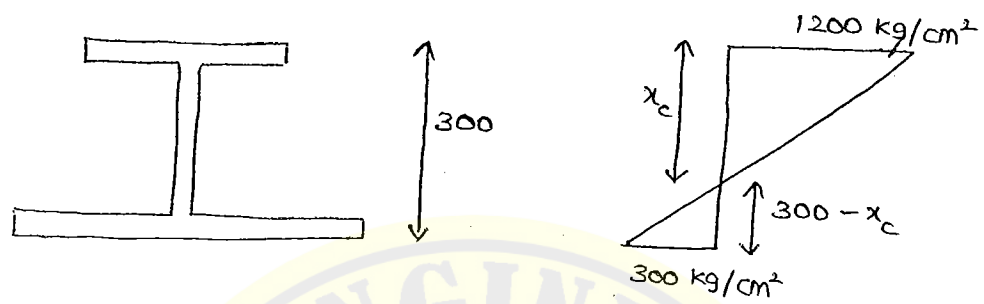
3. 1. $M \leq M_d = \sigma_{bt} \cdot z$

2. $\frac{\text{span}}{\text{depth}}$ ratio S.S. beam - 20, continuous beam - 26
cantilever beam - 7

$$\Delta_{cal} < \Delta_{max} = \frac{\text{span}}{325}$$

$$= 2 \times \frac{\text{span}}{325}$$

7.



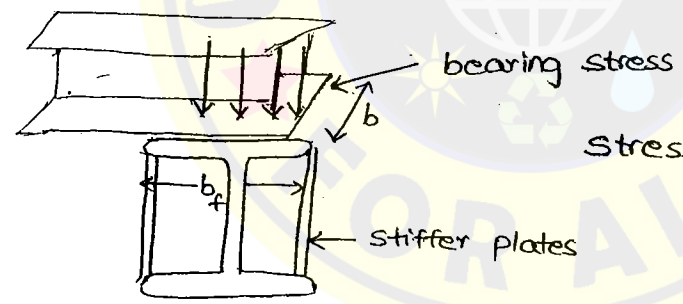
$$\frac{x_c}{1200} = \frac{300 - x_c}{300}$$

$$300 x_c = 1200 \times 300 - 1200 x_c$$

$$1500 x_c = 1200 \times 300$$

$$x_c = 240 \text{ mm}$$

8.



$$\text{Stress} = \frac{W}{b_f \times b} \leq \text{permissible stress}$$

$$\leq 0.75 f_y$$

9.

$$\tau_{\text{cal}} = \frac{V}{d \times t_w}$$

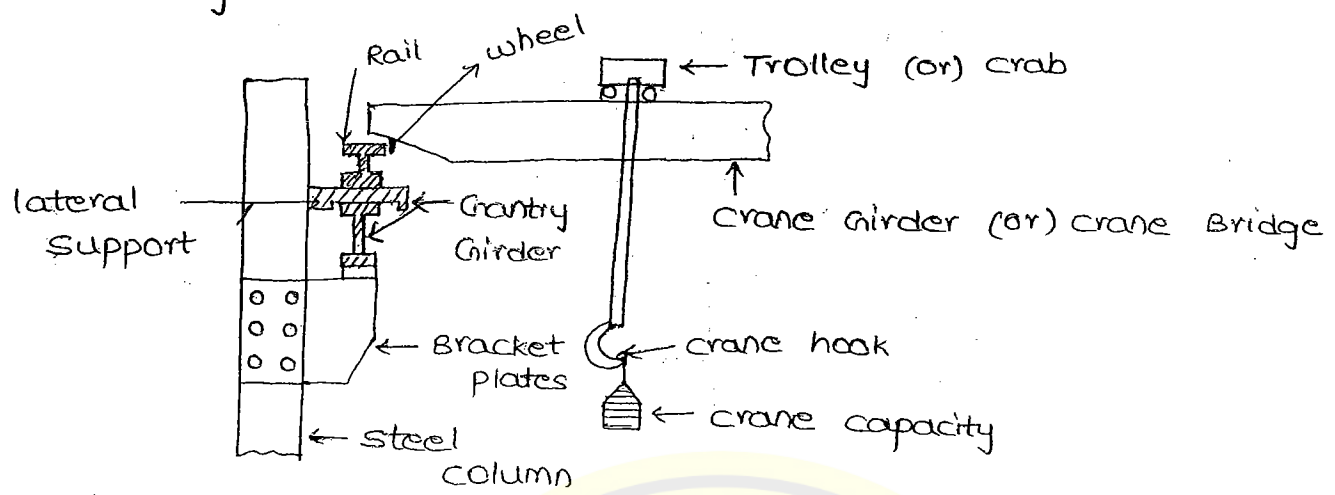
If increase thickness (t_w), τ_{cal} will decrease \rightarrow 1st priority

If increase height (d), τ_{cal} will decrease \rightarrow 2nd priority

1-2-20

UNIT - 10 GANTRY GIRDERS

Gantry Girder :-



EOT cranes :-

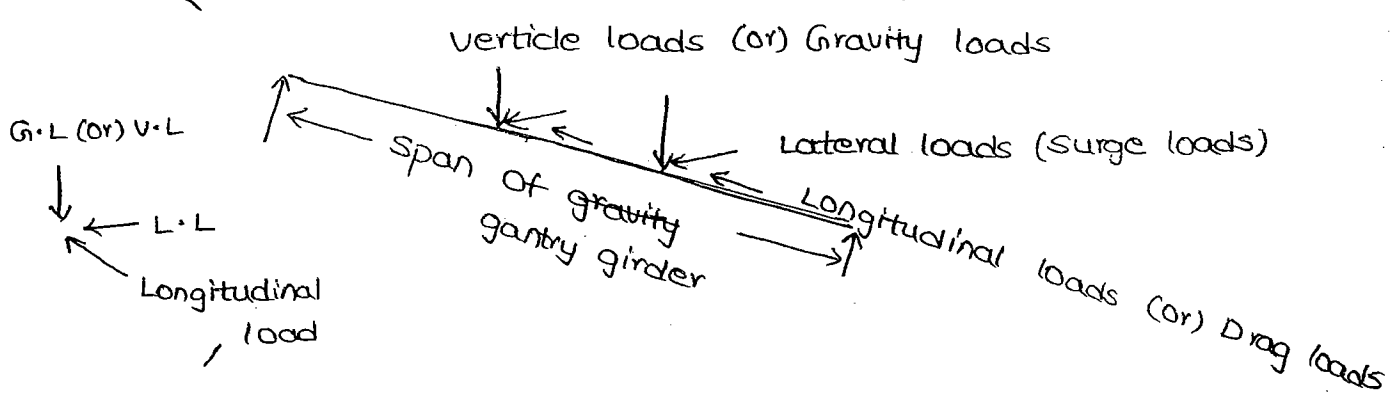
Electrically Operated Overhead travelling cranes

MOT cranes (or) HOT cranes :-

Manually (or) Hand Operated Overhead travelling cranes

Design loads on Gantry Girders :-

1. Gravity loads (or) verticle loads
2. Horizontal loads (or) Normal loads rails (or) lateral loads (\leftrightarrow) (or) Surge loads (Due to crab movement)
3. Longitudinal loads (or) Drag loads (Due to crane movement)
4. Impact loads.



→ Lateral loads are weight of crab + wt. lifted by crane

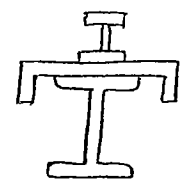
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Types of sections:-

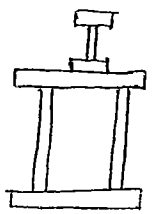
1. ISWB section



2. A channel section is reinforced to compression flange of ISWB section.



3. Box girder



Design criteria of gantry girders:-

- 1. Design for Bending Moment ($M \leq M_d$)
- 2. Design for shear force ($V \leq V_d$)
- 3. Design for deflections ($\Delta_{cal} \leq \Delta_{limit}$ (or) Δ_{max})

** Limiting deflections (or) Maximum deflections (Δ_{max}) (IS 800):-

- 1. For HOT (or) MOT crane $\frac{\Delta_{max}}{L/500}$
- 2. For EOT crane (or) crane capacity upto 50 tons (or) 500 KN $L/750$
- 3. For EOT cranes with crane capacity more than 50 tons (or) 500 KN $L/1000$
- 4. For other moving equipments $L/600$

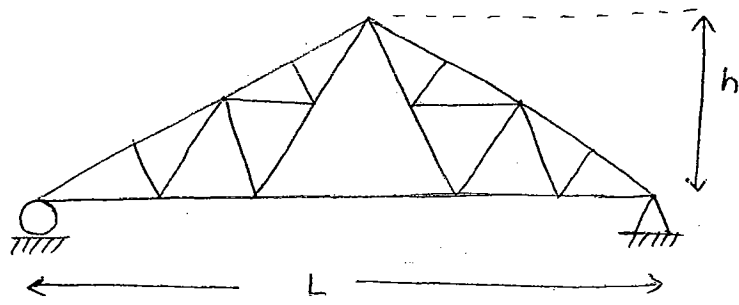
$\therefore L = \text{span of Gantry Girder.}$

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UNIT - II ROOF TRUSSES



Compound Fink Truss

L = span of the truss

h = Rise of the truss

Selection criteria for roof truss:-

- 1. span of the truss (L)
- 2. pitch of the truss.

pitch of the truss depends on

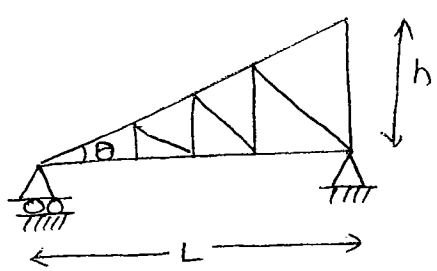
- 1. Type of roof covering material to be used (like AC sheets, GI sheets, plastic sheets etc).
- 2. Light of ventilation requirement.

3. $\text{pitch} = \frac{\text{Rise}}{\text{span}} = \frac{h}{L}$

4. $\text{slope} = \tan \theta = \frac{\text{Rise}}{\text{half span}} = \frac{h}{L/2} = \frac{2h}{L}$

slope = 2 x pitch for symmetric truss

North light truss:-



$\text{slope} = \tan \theta = \frac{h}{L}$

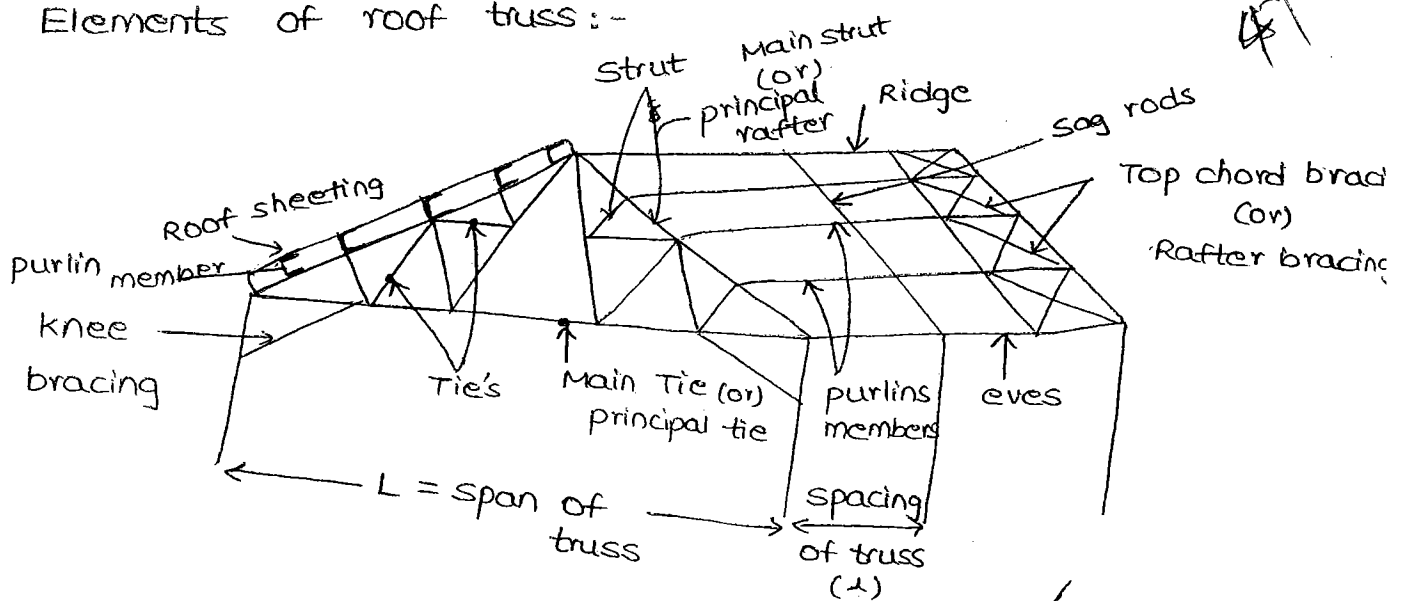
$\text{pitch} = \frac{h}{L}$

$\text{slope} = \text{pitch} = \frac{h}{L}$ for north light roof truss

Day light is main criteria for selection of North light roof truss.

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Elements of roof truss :-



purlin member always design as continuous beam.

Economical spacing of truss (λ) :-

Spacing at which the total cost of the roof building is minimum.

$$\text{Economical spacing of truss } (\lambda) = \frac{L}{3} \text{ to } \frac{L}{5}$$

L = span of the truss

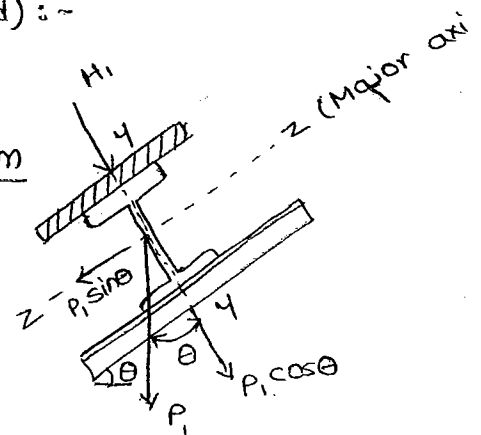
$$\text{Cost of the truss per unit area (t)} = 2 \times \text{cost of purlin per unit area} + \text{cost of roof sheeting per unit area (s)}$$

$$t = 2P + s$$

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Design of purlin (working stress method) :-

Is 800 is recommended to design as purlin as a continuous beam and subjected to axial bending moment



P_1 = Gravity (or) vertical load due to self weight of purlin, self weight of sheeting, live load.

H_1 = Load due to wind pressure

P = Load along minor axis of purlin = $(H_1 + P_1 \cos \theta)$

H = Load along major axis of purlin = $P_1 \sin \theta$

θ = slope of roof.

λ = span of purlin (i.e., spacing between truss)

Design criteria:-

$$\text{Actual bending stress} = \left[\frac{M_{zz} \cdot y}{I_{zz}} + \frac{M_{yy} \cdot z}{I_{yy}} \right]$$

$$\sigma_{bc \text{ cal}} \text{ (or) } \sigma_{bt \text{ cal}} = \left[\frac{M_{zz}}{Z_{zz}} + \frac{M_{yy}}{Z_{yy}} \right] \leq \text{permissible bending stresses}$$

$$\Delta_{\text{cal}} \leq \Delta_{\text{max}} = \frac{\text{Span}}{200} \text{ (maximum or limiting deflection)}$$

Design of angle purlin:-

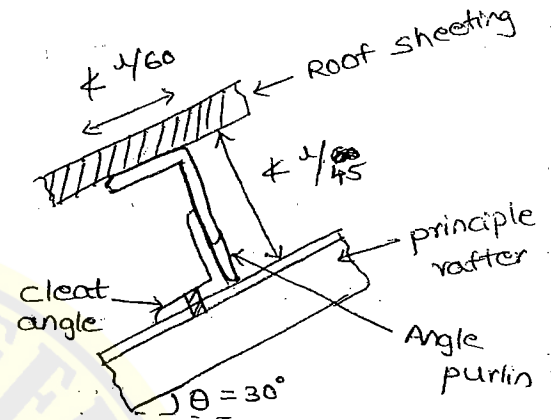
IS 800:1984 recommends

to use an angle member as purlin

1. When slope of roof $\theta \leq 30^\circ$
 $\neq 30^\circ$
2. Minimum live load to be used $\neq 750 \text{ N/m}^2$
3. Bending moment about minor axis is neglected.
4. Maximum bending moment, $M = \frac{w l^2}{10}$

l = span of purlin

w = uniformly distributed load inclusive of wind load.



P.9 No.1-74

2.

$$M_{xx} = P \cos 30^\circ$$

$$= \frac{\sqrt{3}}{2} P$$

$$= \frac{\sqrt{3}}{2} M$$

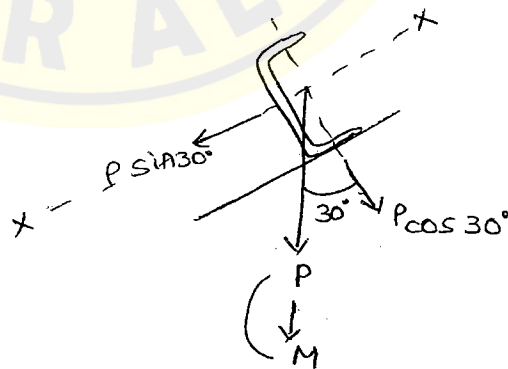


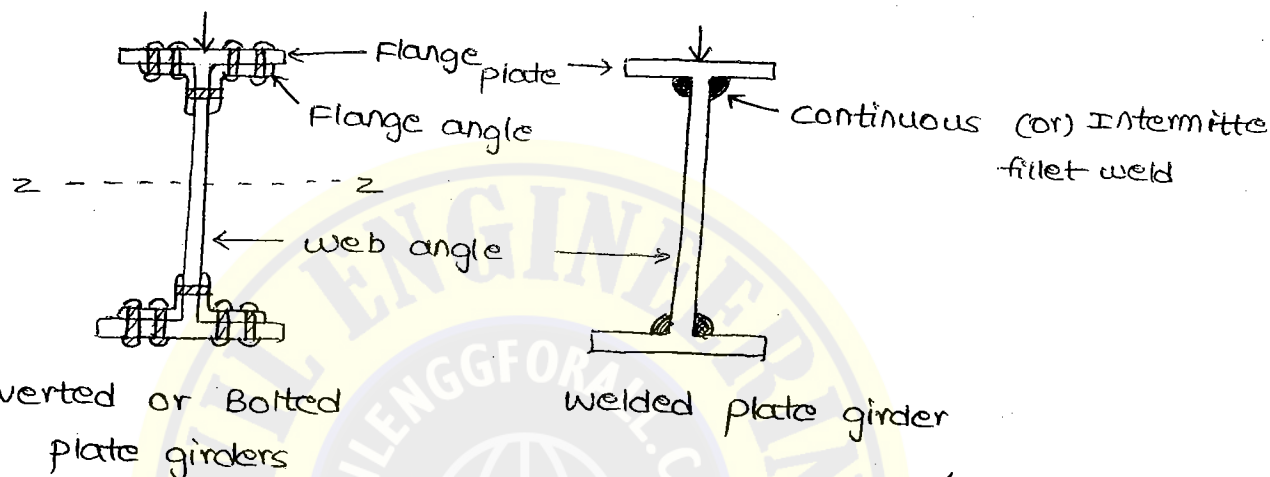
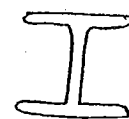
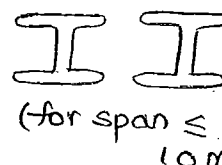
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UNIT - 9

PLATE GIRDERS

Alternatives for large span beam with heavy transverse loads.

1. Two rolled I-section (placed side by side)
2. Plate girders (for span 20m - 100m)
3. Truss girders (for span > 100m)



Riveted or Bolted plate girders

welded plate girder

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Note:-

In terms of simplicity and efficiency welded plate girders are far superior than bolted or riveted plate girder.

To support the same transverse load welded plate are demanding 5-15% of lesser area of steel than bolted or riveted plate girder.

Self weight of Riveted (or) Bolted plate girder (w_r)

$$w_r = \frac{W}{300} \text{ KN/m}$$

self weight of welded plate girder

$$w_w = \frac{W}{400} \text{ KN/m}$$

W = Total superimposed load in KN

→ The riverts or bolts used between flange angle and flange plate to be designed for horizontal shear due to transverse load. Here riverts or bolts subjected to single shear only.

→ The riveted connection between flange angle and web plate to be designed for horizontal shear due to transverse load and vertical live load. Here rivets are subjected to double shear.

Web equivalent:-

It is a portion of web between flange angles is behaves as part of flange area is called web equivalent.

$$\text{Area of web plate, } A_w = d_w \cdot t_w$$

d_w = Depth of web plate

t_w = Thickness of web plate.

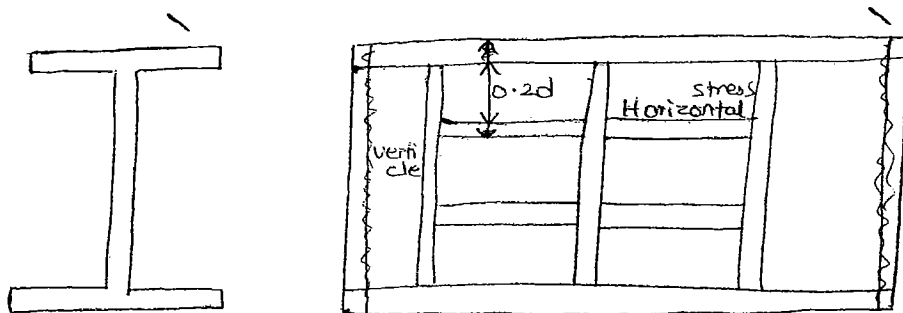
Area of web equivalent (A_{we})

$$\text{Gross web equivalent, } A_{we} = \frac{A_w}{6} \quad (\text{used in compressive side})$$

$$\text{Net web equivalent, } A_{we} = \frac{A_w}{8}$$

Local buckling failures of web plate in a plate girder:-

1. Shear buckling failure (or) Diagonal compression ~~for~~ buckling failure of web.
2. Horizontal (or) Longitudinal (or) bending buckling failure of web
3. Vertical (or) Bearing buckling failure of web.



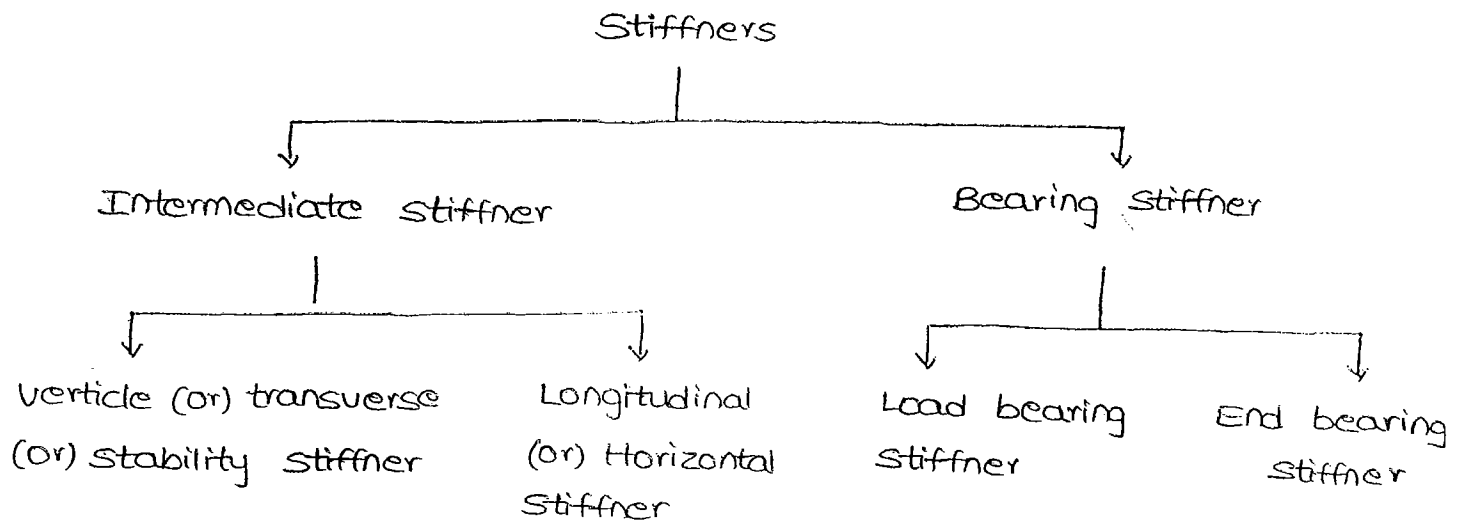
Elements of plate girder:-

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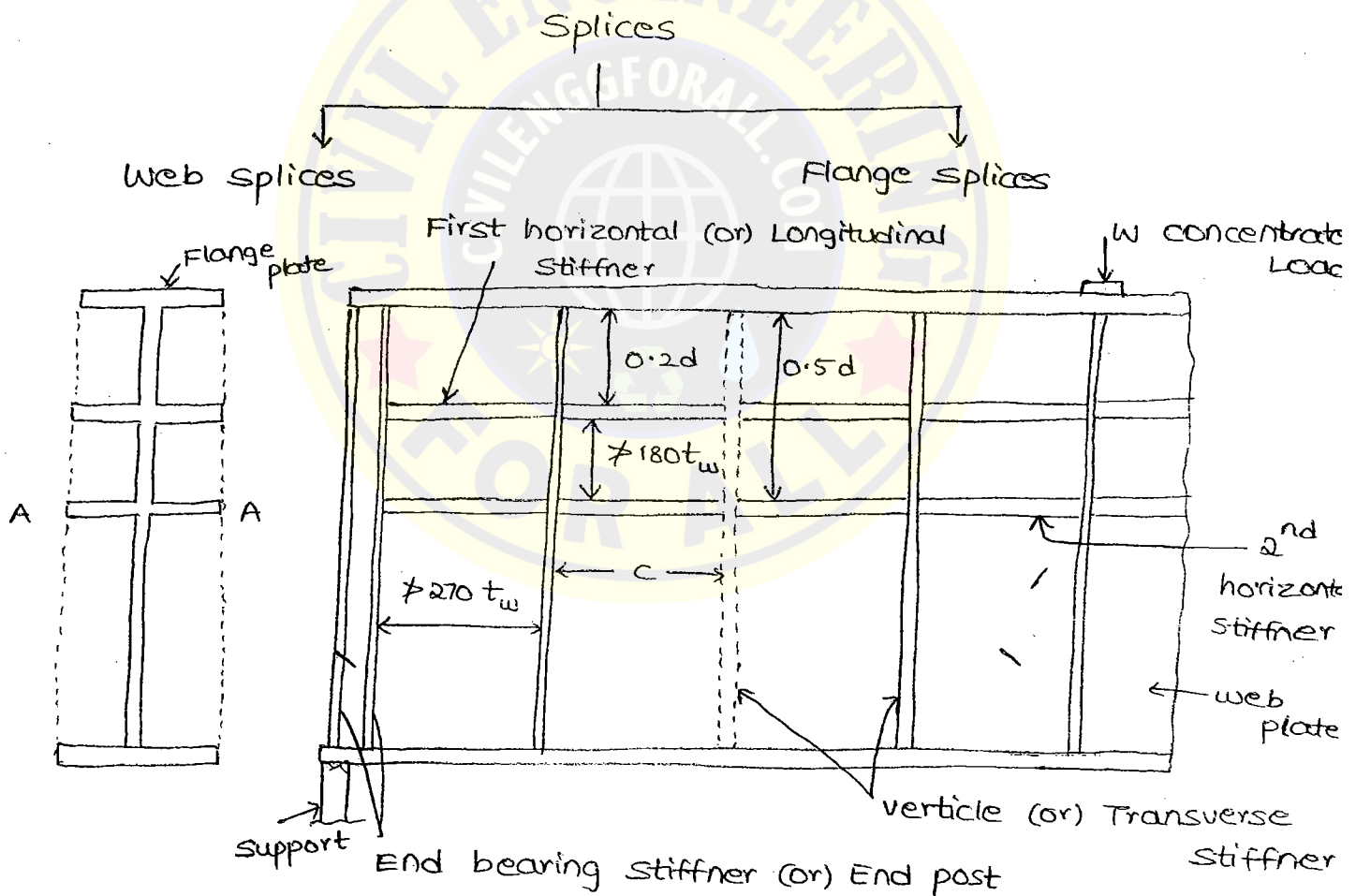
1. Web plate
2. Flange plates only for welded plate girder. Flange angles and flange plates for Bolted (or) Riveted plate girder.

Stiffeners :-

43



Splices :-



d_w = Depth of web plate

t_w = thickness of web plate

c = spacing between verticle (or) Transverse stiffener

d = depth of plate girder

d_2 = 2 x distance from compression flange plate (or) Flange angle to neutral axis.

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Web plate:-

Minimum thickness of web plate (t_w):-

1. $t_w \geq 6$ mm (when web plate is exposed to weather but accessible for painting)
2. $t_w = 8$ mm (when web plate exposed to weather but inaccessible for painting)

Economical depth of plate girder (D):-

It is concept based on minimum area of steel

(or) minimum weight of steel required for plate girder.

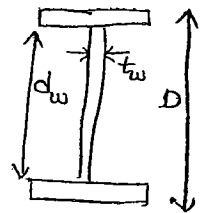
For riveted plate girder, $D = 1.1 \sqrt{\frac{M}{\sigma_{bt} \cdot t_w}}$

For welded plate girder, $D = 5 \sqrt{\frac{M}{\sigma_{bc}}}$

/ $M =$ Maximum bending moment

$\sigma_{bt} =$ permissible bending stresses

$d_w \leq$



Web stiffeners (IS 800: 1984):-

1. When $\frac{d_1}{t_w} \leq$ lesser of $\frac{816}{\sqrt{\tau_{va, cal}}}$ and 85 - No stiffener is required
2. When $85 < \frac{d_2}{t_w} \leq 200$ - verticle, transverse stiffner are to be provided.
3. when $200 < \frac{d_2}{t_w} \leq 250$ - Transverse stiffner + 1st horizontal stiffner at 0.2d from compression flange.
4. When $250 < \frac{d_2}{t_w} \leq 400$ - Transverse stiffner + 1st Horizontal stiffner + 2nd horizontal stiffner at Neutral Axis.

Verticle transverse stiffner:- Photo Copy By Jain's 09700291147

Verticle stiffners (or) transverse stiffners are to be provided to eliminate shear buckling failure of web plate.

Flat stiffener to be selected for welded plate girder or an angle member to be used as a stiffener as riveted (or) bolted connection.

Minimum moment of Inertia for transverse stiffener

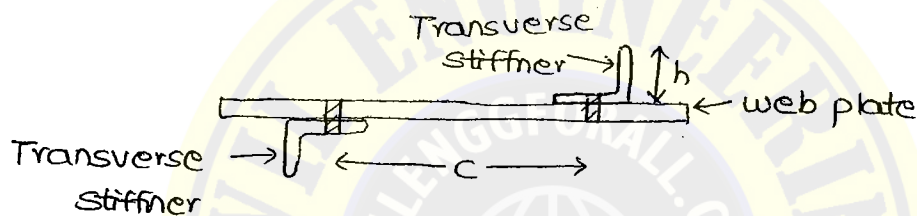
$$I_s \geq \frac{1.5 d_1^3 t_w^3}{c^2}$$

t_w = minimum required thickness of web

c = maximum permitted distance between verticle stiffener. stress

Maximum spacing between transverse stiffener = $1.5 d_1$

Minimum spacing between transverse stiffener = $0.33 d_1$



h = Outstand of stiffener.

Horizontal stiffener (or) Longitudinal stiffener:-

1. First horizontal stiffener at $0.2d$ from compression flange

Minimum M.I $I_s \geq 4c t_w^3$

2. Second horizontal stiffener at Neutral axis

Minimum M.I $I_s \geq d_2^3 t_w^3$

d_2 = 2x distance from compression flange plate (or) flange angle to N.A.

Note:-

Horizontal (or) Longitudinal stiffeners are to be provide to eliminate longitudinal (or) bending (or) Horizontal buckling failure of web plate.

The connection between verticle stiffener to the web plate (or) Horizontal stiffener to the web plate to be design for a S.F not less than $\frac{125 t_w^2}{h}$ (KN/m)

$\therefore h$ = Outstand of stiffener Photo Copy By Jain's 09700291147

P.g NO1-65

2. Given $d_1 = 1800 \text{ mm}$ $\tau_{va, cal} = 81 \text{ Mpa}$

$$\frac{d_1}{t_w} \leq \frac{816}{\sqrt{\tau_{va, cal}}}$$

$$\frac{1800}{t_w} \leq \frac{816}{\sqrt{81}}$$

$$t_w \geq \frac{1800 \times 9}{816}$$

$$t_w \geq 20 \text{ mm}$$

3. Given $d = 1000 \text{ mm}$ $t_w = 10 \text{ mm}$

$$\frac{d_2}{t_w} = \frac{1000}{10} = 100$$

$$\frac{d_1}{t_w} \leq 85 \text{ — No stiffener required}$$

$$85 < \frac{d_2}{t_w} \leq 200 \text{ — Intermediate vertical stiffener required}$$

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